## Detailed Analysis of GATE 2016 Paper

GATE ME Solved 2016 Paper (Set I) Detailed Analysis

| Subject | Topic | 1 Mark Questions | 2 Marks Questions | Total Questions | Total Marks |
| :---: | :---: | :---: | :---: | :---: | :---: |
| General Aptitude | Numerical Ability | 2 | 4 | 6 | 10 |
|  | Verbal Ability | 3 | 2 |  | 5 |
| Total marks |  |  |  |  | 15 |
| Engineering Maths | Linear Algebra | 1 |  | 1 | 1 |
|  | Differential Equations | 1 | 1 | 2 | 3 |
|  | Complex Variables | 1 | 1 | 2 | 3 |
|  | Probability | 1 |  | 1 | 1 |
|  | Numerical Methods | 1 | 1 | 2 | 3 |
|  | Calculus |  | 1 | 1 | 2 |
| Total marks |  |  |  |  | 13 |
| Strength of Materials | Torsion of Shafts | 1 | 0 | 1 | 1 |
|  | Deflection of Beam | 1 | 0 | 1 | 1 |
|  | Properties of Metals | 1 | 1 | 2 | 3 |
|  | Stress and Strain | 0 | 1 | 1 | 2 |
|  | Moment of Inertia | 0 | 1 | 1 | 2 |
|  | Shear Force and Bending Moment | 0 | 1 | 1 | 2 |
|  | Prinicipal Stresses and Strains | 0 | 1 | 1 | 2 |
| Total Marks |  |  |  |  | 13 |
| Theory of Machines | Gyroscope | 1 | 0 | 1 | 1 |
|  | Vibration | 1 | 1 | 2 | 3 |
|  | Dynamic Analysis of Linkages | 0 | 1 | 1 | 2 |
|  | Gear Trains | 0 | 1 | 1 | 2 |
| Total Marks |  |  |  |  | 8 |
| Machine Design | Springs | 1 | 0 | 1 | 1 |
| Total Marks |  |  |  |  | 1 |
| Thermodynamics | First Law of Thermodynamics | 1 | 1 | 2 | 3 |
|  | I.C. Engines | 1 | 0 | 1 | 1 |
|  | Vapour Power Cycle | 1 | 1 | 2 | 3 |
|  | Thermodynamic Relations | 0 | 1 | 1 | 2 |
|  | Properties of Pure Substances | 0 | 1 | 1 | 2 |
| Total Marks |  |  |  |  | 11 |
| Fluid Mechanics \& Machinary | Fluid Kinematics | 1 | 0 | 1 | 1 |
|  | Buoyancy | 1 | 0 | 1 | 1 |
|  | Hydaulic Turbine | 1 | 0 | 1 | 1 |
|  | Manometry | 0 | 1 | 1 | 2 |
|  | Dimensional Analysis | 0 | 1 | 1 | 2 |
|  | Boundary Layer Theory | 0 | 1 | 1 | 2 |
| Total Marks |  |  |  |  | 9 |

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| Manufacturing Technology | Welding | 1 | 0 | 1 | 1 |
| :---: | :---: | :---: | :---: | :---: | :---: |
|  | Metal Casting | 1 | 1 | 2 | 3 |
|  | Non-traditional Machining | 1 | 0 | 1 | 1 |
|  | Metal Cutting | 1 | 1 | 2 | 3 |
|  | Metrology | 1 | 0 | 1 | 1 |
|  | Metal Forming | 0 | 1 | 1 | 2 |
|  | CIM | 0 | 1 | 1 | 2 |
| Total Marks |  |  |  |  | 13 |
| Industrial Engineering | Inventory Control |  | 1 | 1 | 2 |
|  | Linear Programming |  | 1 | 1 | 2 |
| Total Marks |  |  |  |  | 4 |
| Heat and Mass Transfer | Conduction | 1 | 0 | 1 | 1 |
|  | Transient Conduction | 0 | 1 | 1 | 2 |
|  | Radiation | 0 | 1 | 1 | 2 |
|  | Convection | 0 | 1 | 1 | 2 |
| Total Marks |  |  |  |  | 7 |
| Engineering Mechanics | FBD and Equilibrium | 1 | 0 | 1 | 1 |
|  | Kinematics of Bodies | 1 | 0 | 1 | 1 |
|  | Friction | 0 | 1 | 1 | 2 |
|  | Trusses | 0 | 1 | 1 | 2 |
| Total Marks |  |  |  |  | 6 |

GATE ME Solved 2016 Paper (Set 2) Detailed Analysis

| Subject | Topic | 1 Mark Questions | 2 Marks Questions | Total Questions | Total Marks |
| :---: | :---: | :---: | :---: | :---: | :---: |
| General Aptitude | Numerical Ability | 2 | 4 | 6 | 10 |
|  | Verbal Ability | 3 | 2 | 5 | 5 |
| Total marks |  |  |  |  | 15 |
| Engineering Maths | Linear Algebra | 1 |  | 1 | 1 |
|  | Calculus | 1 | 1 | 2 | 3 |
|  | Differential Equations | 1 |  | 1 | 1 |
|  | Complex Variables | 1 | 1 | 2 | 3 |
|  | Numerical Methods | 1 | 1 | 2 | 3 |
|  | Probability |  | 1 | 1 | 2 |
| Total marks |  |  |  |  | 13 |
| Strength of materials | Prinicipal stresses and strains | 2 | 0 | 2 | 2 |
|  | Thin cylinders | 1 | 0 | 1 | 1 |
|  | Stress and strain | 1 | 2 | 3 | 5 |
|  | Deflection of beam | 0 | 2 | 2 | 4 |
| Total Marks |  |  |  |  | 12 |
| Theory of Machines | Vibration | 1 | 1 | 2 | 3 |
|  | Velocity Analysis | 0 | 1 | 1 | 2 |
| Total Marks |  |  |  |  | 5 |


| Machine Design | Brakes | 2 | 0 | 2 | 2 |
| :---: | :---: | :---: | :---: | :---: | :---: |
|  | Design Against Static Load | 0 | 1 | 1 | 2 |
| Total Marks |  |  |  |  | 4 |
| Thermodynamics | First Law of Thermodynamics | 1 | 1 | 2 | 3 |
|  | Gas Turbine | 1 | 0 | 1 | 1 |
|  | Second Law of Thermodynamics | 1 | 1 | 2 | 3 |
|  | RAC | 0 | 2 | 2 | 4 |
| Total Marks |  |  |  |  | 11 |
| Fluid mechanics \& machinary | Fluid Kinematics | 1 | 0 | 1 | 1 |
|  | Manometry | 1 | 0 | 1 | 1 |
|  | Buoyancy | 0 | 2 | 2 | 4 |
|  | Viscous Flow | 0 | 1 | 1 | 2 |
| Total Marks |  |  |  |  | 8 |
| Manufacturing Technology | Engg. Material | 1 | 1 | 2 | 3 |
|  | Welding | 1 | 1 | 2 | 3 |
|  | Metal Cutting | 1 | 1 | 2 | 3 |
|  | Non-traditional Machining | 1 | 1 | 2 | 3 |
|  | Metal Casting | 0 | 1 | 1 | 2 |
|  | Metrology | 0 | 1 | 1 | 2 |
| Total Marks |  |  |  |  | 16 |
| Industrial Engineering | Inventory Control | 0 | 1 | 1 | 2 |
|  | PERT/CPM | 0 | 1 | 1 | 2 |
|  | Queuing Theory | 1 | 0 | 1 | 1 |
| Total Marks |  |  |  |  | 5 |
| Heat and Mass Transfer | Conduction | 1 | 0 | 1 | 1 |
|  | Radiation | 1 | 0 | 1 | 1 |
|  | Heat Exchangers | 0 | 1 | 1 | 2 |
|  | Transient Conduction | 0 | 1 | 1 | 2 |
| Total Marks |  |  |  |  | 6 |
| Engineering Mechanics | Momentum | 1 | 1 | 2 | 3 |
|  | FBD and Equilibrium |  | 1 | 1 | 2 |
| Total Marks |  |  |  |  | 5 |

GATE ME Solved 2016 Paper (Set 3) Detailed Analysis

| Subject | Topic | 1 Mark Questions | 2 Marks Questions | Total Questions | Total Marks |
| :---: | :---: | :---: | :---: | :---: | :---: |
| General Aptitude | Numerical Ability | 2 | 4 | 6 | 10 |
|  | Verbal Ability | 3 | 2 |  | 5 |
| Total marks |  |  |  |  | 15 |
| Engineering Maths | Linear Algebra | 1 | 1 | 2 | 3 |
|  | Calculus | 1 | 2 | 3 | 5 |
|  | Differential Equations | 1 |  | 1 | 1 |
|  | Probability | 1 | 1 | 2 | 3 |
|  | Numerical Methods | 1 |  | 1 | 1 |
| Total marks |  |  |  |  | 13 |

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| Strength of Materials | Prinicipal Stresses and Strains | 1 | 0 | 1 | 1 |
| :---: | :---: | :---: | :---: | :---: | :---: |
|  | Stress and Strain | 1 | 1 | 2 | 3 |
|  | Torsion of Shafts | 0 | 1 | 1 | 2 |
|  | Bending Stress | 1 | 0 | 1 | 1 |
|  | Deflection of Beam | 0 | 1 | 1 | 2 |
| Total Marks |  |  |  |  | 9 |
| Theory of Machines | Planar Mechanism | 1 | 0 | 1 | 1 |
|  | Vibration | 1 | 1 | 2 | 3 |
|  | Balancing | 0 | 1 | 1 | 2 |
| Total Marks |  |  |  |  | 6 |
| Machine Design | Bolted Joints | 0 | 1 | 1 | 2 |
|  | Bearings | 1 | 0 | 1 | 1 |
| Total Marks |  |  |  |  | 3 |
| Thermodynamics | Properties of Pure Substances | 1 | 0 | 1 | 1 |
|  | SFEE | 0 | 1 | 1 | 2 |
|  | RAC | 0 | 2 | 2 | 4 |
|  | Compressors | 0 | 1 | 1 | 2 |
| Total Marks |  |  |  |  | 9 |
| Fluid Mechanics \& Machinary | Flow Through Pipe | 1 | 0 | 1 | 1 |
|  | Fluid Kinematics | 1 | 1 | 2 | 3 |
|  | Hydaulic Turbine | 1 | 0 | 1 | 1 |
|  | Viscous Flow | 0 | 1 | 1 | 2 |
|  | Impact of Jets | 0 | 1 | 1 | 2 |
| Total Marks |  |  |  |  | 9 |
| Manufacturing Technology | Non-traditional Machining | 1 | 0 | 1 | 1 |
|  | Metal Cutting | 1 | 3 | 4 | 7 |
|  | CIM | 1 | 1 | 2 | 3 |
|  | Welding | 0 | 1 | 1 | 2 |
|  | Metrology | 0 | 1 | 1 | 2 |
| Total Marks |  |  |  |  | 15 |
| Industrial Engineering | Forecasting | 0 | 1 | 1 | 2 |
|  | Linear Programming | 0 | 1 | 1 | 2 |
|  | PERT/CPM | 1 | 0 | 1 | 1 |
| Total Marks |  |  |  |  | 5 |
| Heat and Mass Transfer | Conduction | 2 | 0 | 2 | 2 |
|  | Convection | 1 | 0 | 1 | 1 |
|  | Heat Exchangers | 1 | 0 | 1 | 1 |
|  | Radiation | 0 | 1 | 1 | 2 |
|  | Transient Conduction | 0 | 1 | 1 | 2 |
| Total Marks |  |  |  |  | 8 |
| Engineering Mechanics | FBD and Equilibrium | 1 | 1 | 2 | 3 |
|  | Relative Velocity | 1 | 0 | 1 | 1 |
|  | Kinematics of Bodies | 0 | 2 | 2 | 4 |
| Total Marks |  |  |  |  | 8 |

# GATE 2016 Solved Paper ME: Mechanical Engineering Set - I 

## Number of Questions: 65

Total Marks:100.0

Wrong answer for MCQ will result in negative marks, ( $-1 / 3$ ) for 1 Mark Questions and ( $-2 / 3$ ) for 2 Marks Questions.

## General Aptitude

## Number of Questions: 10

Section Marks: 15.0

## Q. 1 to Q. 5 carry 2 mark each and Q. 6 to Q. 10 carry 1 marks each.

## Question Number: 1 <br> Question Type: MCQ

A person moving through a tuberculosis-prone zone has a $50 \%$ probability of becoming infected. However, only $30 \%$ of infected people develop the disease. What percentage of people moving through a tuberculosis-prone zone remains infected but does not show symptoms of the disease?
(A) 15
(B) 33
(C) 35
(D) 37

Solution: We can assume that $50 \%$ of the people moving through the zone get infected for a large population. As $30 \%$ of these actually develop the disease, the other $70 \%$ (i.e., $70 \%$ of $50 \%$ which is $50 \%$ of $70 \%$ or $35 \%$ ) do not develop the disease or do not show symptoms of the disease.
Hence, the correct option is (C).

## Question Number: 2 <br> Question Type: MCQ

In a world filled with uncertainty, he was glad to have many good friends. He had always assisted them in times of need and was confident that they would reciprocate. However, the events of the last week proved him wrong.
Which of the following inference(s) is/are logically valid and can be inferred from the above passage?
(i) His friends were always asking him to help them.
(ii) He felt that when in need of help, his friends would let him down.
(iii) He was sure that his friends would help him when in need.
(iv) His friends did not help him last week.
(A) (i) and (ii)
(B) (iii) and (iv)
(C) (iii) only
(D) (iv) only

Solution: Options (iii) and (iv) can be logically inferred from the given text.
Hence, the correct option is (B).

Question Number: 3
Question Type: MCQ
Leela is older than her cousin Pavithra. Pavithra's brother Shiva is older than Leela. When Pavithra and Shiva are visiting Leela, all three like to play chess. Pavithra wins more often than Leela does.
Which one of the following statements must be TRUE based on the above?
(A) When Shiva plays chess with Leela and Pavithra, he often loses.
(B) Leela is the oldest of the three.
(C) Shiva is a better chess player than Pavithra.
(D) Pavithra is the youngest of the three.

Solution: Shiva $>$ Leela $>$ Pavithra
Statement (B) can be immediately removed as it is wrong.
Statement (D) is TRUE.
Statements (A) and (C), we cannot be certain
Hence, the correct option is (D).
Question Number: 4
Question Type: MCQ
If $q^{-a}=\frac{1}{r}$ and $r^{-b}=\frac{1}{s}$ and $s^{-c}=\frac{1}{q}$, the value of $a b c$ is
(A) $(r q s)^{-1}$
(B) 0
(C) 1
(D) $r+q+s$

Solution:

$$
\begin{align*}
& q^{-a}=r^{-1} \Rightarrow q^{a}=r  \tag{1}\\
& r^{-b}=s^{-1} \Rightarrow r^{b}=s  \tag{2}\\
& s^{-c}=q^{-1} \Rightarrow s^{c}=q \tag{3}
\end{align*}
$$

Substituting the value of $q$ from Eq. (3) in Eq. (1), we get $s^{c a}=r$. Substituting the value of $s$ from Eq. (2) in this, we get $r^{a b c}=r$. Therefore,

$$
r=-1,0,1 \text { or } a b c=1
$$

None of $p, q, r, s$ is 0 . But they could all be $-1(a, b, c$ would have to be odd numbers, say for example $(-1,-3,-5))$. We see that their values satisfy the equations.

$$
\begin{aligned}
& q^{-a}=(-1)^{1}=-1=\frac{1}{r} ; \\
& r^{-b}=(-1)^{3}=-1=\frac{1}{s} \\
& s^{-c}=(-1)^{5}=-1=\frac{1}{q}
\end{aligned}
$$

In this case, $a b c$ is -15 . Similarly, it can have infinitely many values. If the additional data (that none of $q, r, s$ is -1 or 1 ) is given, we would be able to conclude that $a b c=1$.
In the exam, in the absence of the correct option, we can guess that the expected answer is choice C .
Hence, the correct option is (C).

## Question Number: 5

Question Type: MCQ
$P, Q, R$ and $S$ are working on a project. $Q$ can finish the task in 25 days, working alone for 12 hours a day. $R$ can finish the task in 50 days, working alone for 12 hours per day. $Q$ worked 12 hours a day but took sick leave in the beginning for two days. $R$ worked 18 hours a day on all days. What is the ratio of work done by $Q$ and $R$ after 7 days from the start of the project?
(A) $10: 11$
(B) $11: 10$
(C) $20: 21$
(D) $21: 20$

Consider the table given below:

|  | $\boldsymbol{Q}$ | $\boldsymbol{R}$ |
| :--- | :--- | :--- |
| Rate | 2 | 1 |
| No of days | 5 | 7 |
| No of hours/day | 12 | 18 |

The ratio of the work done by $Q$ and $R$ is

$$
W_{Q} / W_{R}=\frac{2(5)(12)}{1(7)(18)}=\frac{20}{21} .
$$

Hence, the correct option is (C).
Question Number: $6 \quad$ Question Type: MCQ
Which of the following is CORRECT with respect to grammar and usage?
Mount Everest is $\qquad$
(A) the highest peak in the world
(B) the highest peak in the world
(C) one of highest peak in the world
(D) one of the highest peak in the world

Solution: The superlative adjective 'highest' should always be preceded by the definite article 'the'.
Hence, the correct option is (A).
Question Number: 7
Question Type: MCQ
The policeman asked the victim of a theft, 'What did you
$\qquad$ ?'
(A) loose
(B) lose
(C) loss
(D) louse

Solution: Loss, which is a noun, does not make sense here. Lose means to have something taken away, which is appropriate in the blank. Loose means not firmly fixed. Louse refers to a small insect that lives in the bodies of human and animals.
Hence, the correct option is (B).
Question Number: 8
Question Type: MCQ
Despite the new medicine's $\qquad$ in treating diabetes, it is not $\qquad$ widely.
(A) effectiveness --- prescribed
(B) availability --- used
(C) prescription --- available
(D) acceptance --- proscribed

Solution: Prescribe means to be told by a doctor to take a particular medicine or have a particular treatment. Proscribe is to ban. The words given in option (A) are precise in the given blanks.

Hence, the correct option is (A).
Question Number: 9
Question Type: MCQ
In a huge pile of apples and oranges, both ripe and unripe mixed together, $15 \%$ are unripe fruits. Of the unripe fruits, $45 \%$ are apples. Of the ripe ones, $66 \%$ are oranges. If the pile contains a total of $56,92,000$ fruits, how many of them are apples?
(A) $20,29,198$
(B) $24,67,482$
(C) $27,89,080$
(D) $35,77,422$

Solution: Consider the table given below:

|  | Apples | Oranges | Total <br> (\%) | Total |
| :--- | :--- | :--- | ---: | :--- |
| Unripe | $(0.45) 15 \%$ <br> $=6.75 \%$ |  | 15 |  |
| Ripe | $(0.34) 85 \%$ <br> $=28.9 \%$ | $(0.66)$ | 85 |  |
|  |  | 100 | $56,92,000$ |  |

Among the ripe fruits, $66 \%$ are oranges. $\therefore 34 \%$ are apples. The percentage of apples in the total number of apples and oranges is
$=(0.45)(15)+(0.34)(85)=6.75+28.90=35.65$.
$\therefore$ The number of apples $=\frac{35.65}{100}(56,92,000)=20,29,198$
Hence, the correct option is (A).
Question Number: 10
Question Type: MCQ
Michael lives 10 km away from where I live. Ahmed lives 5 km away and Susan lives 7 km away from where I live.

Arun is farther away than Ahmed but closer than Susan from where I live. From the information provided here, what is one possible distance (in km ) at which I live from Arun's place?
(A) 3.00
(B) 4.99
(C) 6.02
(D) 7.01

## Solution:

Ahmed - 5

Arun -
Susan-7
Michael-10
The distance at which Arun lives could be $x$ where $5<x$ $<7$.

Among the options, it can only be C.
Hence, the correct option is (C).

## Mechanical Engineering

## Number of Questions: 55

Q. 11 to Q. 25 carry 1 mark each and Q. 36 to Q. 65 carry 2 marks each.

## Question Number: 11

Question Type: MCQ
Thesolutiontothesystemofequations $\left[\begin{array}{cc}2 & 5 \\ -4 & 3\end{array}\right]\left[\begin{array}{l}x \\ y\end{array}\right\}=\left\{\begin{array}{c}2 \\ -30\end{array}\right\}$
(A) 6,2
(B) $-6,2$
(C) $-6,-2$
(D) $6,-2$

Solution: The system of equations given is

$$
\begin{array}{rlrl} 
& & {\left[\begin{array}{cc}
2 & 5 \\
-4 & 3
\end{array}\right]\left[\begin{array}{l}
x \\
y
\end{array}\right]=\left[\begin{array}{c}
2 \\
-30
\end{array}\right]} \\
\Rightarrow & 2 x+5 y & =2 \\
-4 x+3 y & =-30  \tag{2}\\
2(1)+(2) \Rightarrow \\
4 x+10 y & =4 \\
& -4 x+3 y & =-30 \\
& & ---------------13 y & =-26 \\
\Rightarrow & y & =-26 / 13
\end{array}
$$

Substituting value of $y$ in Eq. (1), we get

$$
\begin{aligned}
& 2 x+5(-2) & =2 \\
\Rightarrow & x & =6
\end{aligned}
$$

The solution to the given system of equations is $x=6$ and $y=-2$
Hence, the correct option is (D).
Question Number: 12
Question Type: MCQ
If $f(t)$ is a function defined for all $t \geq 0$, its Laplace transform $F(s)$ is defined as
(A) $\int_{0}^{\infty} e^{s t} f(t) d t$
(B) $\int_{0}^{\infty} e^{-s t} f(t) d t$
(C) $\int_{0}^{\infty} e^{i s t} f(t) d t$
(D) $\int_{0}^{\infty} e^{-i s t} f(t) d t$

Section marks: 85.0
Solution: If $f(t)$ is a function defined for all $t \geq 0$, its Laplace transform $F(s)$ is

$$
\int_{0}^{\infty} e^{-s t} f(t) d t
$$

Hence, the correct option is (B).
Question Number: 13
Question Type: MCQ
$f(z)=u(x, y)+i v(x, y)$ is an analytic function of complex variable $z=x+i y$
where $i=\sqrt{-1}$. If $u(x, y)=2 x y$, then $v(x, y)$ may be expressed as
(A) $-x^{2}+y^{2}+$ constant
(B) $x^{2}-y^{2}+$ constant
(C) $x^{2}+y^{2}+$ constant
(D) $-\left(x^{2}+y^{2}\right)+$ constant

Solution: $f(z)=u(x, y)+i v(x, y)$ is given to be analytic.
Also

$$
\begin{gather*}
u(x, y)=2 x y \\
\Rightarrow \frac{\partial u}{\partial x}=2 y \text { and } \frac{\partial u}{\partial y}=2 x \tag{1}
\end{gather*}
$$

As $f(z)$ is analytic, $u(x, y)$ and $v(x, y)$ satisfy CauchyRiemann equations.

$$
\begin{equation*}
\therefore \frac{\partial u}{\partial x}=\frac{\partial v}{\partial y} \text { and } \frac{\partial u}{\partial y}=-\frac{\partial v}{\partial x} \tag{2}
\end{equation*}
$$

We know that the exact differential of $v(x, y)$ is

$$
\begin{array}{rlr}
d v & =\frac{\partial v}{\partial x} d x+\frac{\partial v}{\partial y} d y \\
& =-\frac{\partial u}{\partial y} d x+\frac{\partial u}{\partial x} d y & \text { (from Eq. (2)) } \\
& \delta v=-2 x d x+2 y d y & \text { (from Eq. (1)) }
\end{array}
$$

Integrating on both sides the above equation, we get

$$
\int d v=-\int 2 x d x+\int 2 y d y
$$

$\Rightarrow v=-x^{2}+y^{2}+$ constant
$\therefore v(x, y)=-x^{2}+y^{2}+$ constant
Hence, the correct option is (A).

## Question Number: 14 <br> Question Type: MCQ

Consider a Poisson distribution for the tossing of a biassed coin. The mean for this distribution is $\mu$. The standard deviation of this distribution is given by
(A) $\sqrt{\mu}$
(B) $\mu^{2}$
(C) $\mu$
(D) $1 / \mu$

Solution: We know that the tossing of a biassed coin follows Poisson distribution.

$$
\text { mean }=\lambda=\mu
$$

Therefore, the variance $=\lambda=\mu$
Standard deviation $=\sqrt{\lambda}=\sqrt{\mu}$
Hence, the correct option is (A).
Question Number: 15
Question Type: NAT
Solve the equation $x=10 \cos (x)$ using the Newton-Raphson method. The initial guess is $x=\pi / 4$. The value of the predicted root after the first iteration, up to second decimal, is

Solution: The equation is

$$
x=10 \cos x
$$

Let

$$
f(x)=x-10 \cos x
$$

The derivative of $f(x)$ will be

$$
\Rightarrow \quad f^{\prime}(x)=1+10 \sin x
$$

Given that the initial guess value of $x$ is $x_{0}=\frac{\pi}{4}$

$$
\therefore \quad f\left(x_{0}\right)=f\left(\frac{\pi}{4}\right)=-6.2857
$$

and $\quad f^{\prime}\left(x_{0}\right)=f^{\prime}\left(\frac{\pi}{4}\right)=8.0711$
$\therefore$ By Newton-Raphson method,

$$
\begin{array}{cc}
x_{1}=x_{0}-\frac{f\left(x_{0}\right)}{f^{\prime}\left(x_{0}\right)}=\frac{\pi}{4}-\frac{(-6.2857)}{(8.0711)} \\
\therefore & x_{1}=1.5642
\end{array}
$$

Hence, the correct answer is 1.56 .

Question Number: 16
A rigid ball of weight 100 N is suspended with the help of a string. The ball is pulled by a horizontal force $F$ such that the string makes an angle of $30^{\circ}$ with the vertical. The magnitude of force $F($ in N$)$ is $\qquad$
Solution: Consider the freebody diagram given below:
Weight of rigid ball $=100 \mathrm{~N}$

Question Type: NAT



Angle that string makes with vertical $=30^{\circ}$
From the free-body diagram, we get
$T \cos 30=100$
$T=115.47$
$T \sin 30=F$

$$
F=57.73
$$

Hence, the correct answer is $57.5-58.0$.
Question Number: 17
Question Type: MCQ
A point mass $M$ is released from rest and slides down a spherical bowl (of radius $R$ ) from a height $H$ as shown in the figure below. The surface of the bowl is smooth (no friction). The velocity of the mass at the bottom of the bowl is

(A) $\sqrt{g H}$
(B) $\sqrt{2 g R}$
(C) $\sqrt{2 g H}$
(D) 0

Solution: At the bottom of the bowl
Potential energy $=$ Kinetic energy

$$
\begin{aligned}
M g H & =\frac{1}{2} M v^{2} \\
v^{2} & =2 g H
\end{aligned}
$$

$$
\text { or, } \quad v=\sqrt{2 g H}
$$

Hence, the correct option is (C).
Question Number: 18
Question Type: MCQ
The cross-sections of two hollow bars made of the same material are concentric circles as shown in the figure. It is given that $r_{3}>r_{1}$ and $r_{4}>r_{2}$, and that the areas of the crosssections are the same. $J_{1}$ and $J_{2}$ are the torsional rigidities of the bars on the left and right, respectively. The ratio $J_{2} / J_{1}$ is
(A) $>1$
(B) $<0.5$
(C) $=1$
(D) between 0.5 and 1


Solution: We are given that $r_{4}>r_{2} ; r_{3}>r_{1}$

$$
\begin{aligned}
& \text { Also } \\
& \qquad \begin{array}{c}
\left(r_{4}^{2}-r_{3}^{2}\right)=\left(r_{2}^{2}-r_{1}^{2}\right) \\
J_{1}
\end{array}=\frac{r_{4}^{4}-r_{3}^{4}}{r_{2}^{4}-r_{1}^{4}}=\frac{r_{4}^{2}+r_{3}^{2}}{r_{2}^{2}+r_{1}^{2}} \times \frac{r_{4}^{2}-r_{3}^{2}}{r_{2}^{2}-r_{1}^{2}} \\
& =\frac{\left(r_{2}+x\right)^{2}+\left(r_{1}+y\right)^{2}}{r_{2}^{2}+r_{1}^{2}}\left(r_{4}=r_{2}+x, r_{3}=r_{1}+y\right) \\
& \therefore \frac{J_{2}}{J_{1}}>1
\end{aligned} .
$$

Hence, the correct option is (A).
Question Number: 19
Question Type: MCQ
A cantilever beam having square cross-section of side $a$ is subjected to an end load. If $a$ is increased by $19 \%$, the tip deflection decreases approximately by
(A) $19 \%$
(B) $29 \%$
(C) $41 \%$
(D) $50 \%$

Solution: Deflection can be calculated using

$$
\begin{equation*}
y=\frac{W l^{3}}{3 E I} \tag{1}
\end{equation*}
$$

where

$$
\begin{equation*}
I=\frac{b d^{3}}{12}=\frac{a^{4}}{12}(b=d=a) \tag{2}
\end{equation*}
$$

Substituting Eq. (2) in Eq. (1), we get

$$
\therefore y=\frac{12 W l^{3}}{3 E(a)^{4}}
$$

When $a$ is increased by $19 \%$, then we get the new $a$ as $a^{1}=$ $1.19 a$

$$
\left(a^{1}\right)^{4}=2 a^{4}
$$

Thus,

$$
y^{1}=\frac{12 W l^{2}}{3 E\left(2 a^{4}\right)}=\frac{y}{2}
$$

Hence, the correct option is (D).

## Question Number: 20

Question Type: NAT
A car is moving on a curved horizontal road of radius 100 m with a speed of $20 \mathrm{~m} / \mathrm{s}$. The rotating masses of the engine have an angular speed of $100 \mathrm{rad} / \mathrm{s}$ in the clockwise direction when viewed from the front of the car. The combined moment of inertia of the rotating masses is 10 $\mathrm{kg}-\mathrm{m}^{2}$. The magnitude of the gyroscopic moment (in $\mathrm{N}-\mathrm{m}$ ) is $\qquad$ _.

Solution: Radius of curved road $R=100 \mathrm{~m}$

$$
\begin{aligned}
\text { Speed of car } v & =20 \mathrm{~m} / \mathrm{s} \\
\text { Angular speed } \omega & =100 \mathrm{rad} / \mathrm{s}
\end{aligned}
$$

Combined moment of inertia of the rotating masses $I=$ $10 \mathrm{~kg}-\mathrm{m}^{2}$

$$
\omega_{\mathrm{p}}=v / R=\frac{20}{100}=0.2 \mathrm{rad} / \mathrm{s}
$$

The magnitude of the gyroscopic moment will be

$$
T_{\mathrm{C}}=I \omega \omega_{\mathrm{p}}=10 \times 100 \times 0.2=200 \mathrm{~N}-\mathrm{m}
$$

Hence, the correct answer is 200.
Question Number: 21
Question Type: NAT
A single degree of freedom spring mass system with viscous damping has a spring constant of $10 \mathrm{kN} / \mathrm{m}$. The system is excited by a sinusoidal force of amplitude 100 N . If the damping factor (ratio) is 0.25 , the amplitude of steadystate oscillation at resonance is $\qquad$ mm .

Solution: Spring constant $K=10 \mathrm{kN} / \mathrm{m}$
Amplitude of force $F_{0}=100 \mathrm{~N}$
Damping factor (ratio) is 0.25
The amplitude of steady-state oscillation can be calculated using

$$
X_{0}=\frac{F_{0} / K}{\sqrt{\left(1-\left(\frac{\omega}{\omega_{n}}\right)^{2}\right)^{2}+\left(2 \xi \frac{\omega}{\omega_{n}}\right)^{2}}}
$$

At resonance, $\omega=\omega_{n}$. Therefore, the above relation becomes

$$
\begin{aligned}
X_{0} & =\frac{100 / 10^{4}}{2 \times 0.25} \\
& =0.02 \mathrm{~m}=20 \mathrm{~mm}
\end{aligned}
$$

Hence, the correct answer is 20 .
Question Number: 12
Question Type: MCQ
The spring constant of a helical compression spring DOES NOT depend on
(A) coil diameter
(B) material strength
(C) number of active turns
(D) wire diameter

Solution: Spring constant $k$ of a helical compression spring does not depend on material strength.
Hence, the correct option is (B).

## Question Number: 23

Question Type: MCQ
The instantaneous stream-wise velocity of a turbulent flow is given as follows:

$$
u(x, y, z, t)=\bar{u}(x, y, z)+u^{\prime}(x, y, z, t)
$$

The time-average of the fluctuating velocity $u^{\prime}(\mathrm{x}, \mathrm{y}, \mathrm{z}, \mathrm{t})$ is
(A) $u^{\prime} / 2$
(B) $-\bar{u} / 2$
(C) zero
(D) $\bar{u} / 2$

Solution: Time-average is always zero for fluctuating velocity.

Hence, the correct option is (C).

## Question Number: 24

## Question Type: MCQ

For a floating body, buoyant force acts at the
(A) centroid of the floating body
(B) center of gravity of the body
(C) centroid of the fluid vertically below the body
(D) centroid of the displaced fluid

Solution: Buoyant force acts at the center of buoyance for a floating body, which is the centroid of liquid displaced.
Hence, the correct option is (D).
Question Number: 25
Question Type: MCQ
A plastic sleeve of outer radius $r_{0}=1 \mathrm{~mm}$ covers a wire (radius $r=0.5 \mathrm{~mm}$ ) carrying electric current. Thermal conductivity of the plastic is $0.15 \mathrm{~W} / \mathrm{m}-\mathrm{K}$. The heat transfer coefficient on the outer surface of the sleeve exposed to air is $25 \mathrm{~W} / \mathrm{m}^{2}-\mathrm{K}$. Due to the addition of the plastic cover, the heat transfer from the wire to the ambient will
(A) increase
(B) remain the same
(C) decrease
(D) be zero

Solution: Outer radius $r_{0}=1 \mathrm{~mm}$
Radius of wire $r=0.5 \mathrm{~mm}$
Thermal conductivity of the plastic $k=0.15 \mathrm{~W} / \mathrm{m}-\mathrm{K}$
Heat transfer coefficient $h_{0}=25 \mathrm{~W} / \mathrm{m}^{2}-\mathrm{K}$

$$
r_{\mathrm{cr}}=\frac{k}{h_{0}}=\frac{0.5}{25} \times 1000=6 \mathrm{~mm}
$$

Since $r_{0}<r_{\text {cr }}$
Thus, heat transfer will increase due to the addition of plastic cover.
Hence, the correct option is (A).
Question Number: 26
Question Type: MCQ
Which of the following statements are TRUE with respect to heat and work?
(i) They are boundary phenomena
(ii) They are exact differentials
(iii) They are path functions
(A) both (i) and (ii)
(B) both (i) and (iii)
(C) both (ii) and (iii)
(D) only (iii)

Solution: We know that work and heat are path function, boundary phenomena, and inexact differential.
Hence, the correct option is (B).
Question Number: 27
Question Type: NAT
Propane $\left(\mathrm{C}_{3} \mathrm{H}_{8}\right)$ is burned in an oxygen atmosphere with $10 \%$ deficit oxygen with respect to the stoichiometric requirement. Assuming no hydrocarbons in the products, the volume percentage of CO in the products is $\qquad$
Solution: The following reaction will take place when propane $\left(\mathrm{C}_{3} \mathrm{H}_{8}\right)$ is burned in an oxygen atmosphere with $10 \%$ deficit oxygen with respect to the stoichiometric requirement:

$$
\mathrm{C}_{3} \mathrm{H}_{8}+4.5 \mathrm{O}_{2} \rightarrow \mathrm{CO}+2 \mathrm{CO}_{2}+4 \mathrm{H}_{2} \mathrm{O}
$$

Volume $\%$ of CO in products $=\frac{1}{7} \times 100$

$$
=14.28 \%
$$

Hence, the correct answer is 14.2 to 14.3 .

## Question Number: 28

Question Type: NAT
Consider two hydraulic turbines having identical specific speed and effective head at the inlet. If the speed ratio ( $N_{1} /$ $N_{2}$ ) of the two turbines is 2 , then the respectively power ratio $\left(P_{1} / P_{2}\right)$ is $\qquad$
Solution: We know that

$$
\begin{aligned}
& \qquad N_{\mathrm{S}}=\frac{N \sqrt{P}}{H^{5 / 4}} \\
& N_{\mathrm{s}} H^{5 / 4}=N \sqrt{P} \\
& \text { We are given that } N_{\mathrm{S} 1}=N_{\mathrm{S} 2}, H_{1}=H_{2} \\
& \therefore \quad(N \sqrt{P})_{1}=(N \sqrt{P})_{2} \\
& \therefore \frac{N_{1}}{N_{2}}=\sqrt{\frac{P_{2}}{P_{1}}} \\
& \frac{P_{1}}{P_{2}}=\left(\frac{N_{2}}{N_{1}}\right)^{2}=\left(\frac{1}{2}\right)^{2}=0.25
\end{aligned}
$$

Hence, the correct answer is 0.25 .
Question Number: 29
Question Type: MCQ
The INCORRECT statement about regeneration in vapor power cycle is that
(A) it increases the irreversibility by adding the liquid with higher energy content to the steam generator
(B) heat is exchanged between the expanding fluid in the turbine and the compressed fluid before heat addition
(C) the principle is similar to the principle of Stirling gas cycle
(D) it is practically implemented by providing feed water heaters

Solution: We know that adiabatic mixing of two fluids is always irreversible and two fluids are mixed in the regenerative heat exchanger. So, it increases the irreversibility.
Hence, the correct option is (A).

## Question Number: 30

Question Type: MCQ
The 'Jominy test' is used to find
(A) Young's modulus
(B) hardenability
(C) yield strength
(D) thermal conductivity

Solution: The correct option is (B).

## Question Number: 31

Question Type: MCQ
Under optimal conditions of the process, the temperatures experienced by a copper work piece in fusion welding, brazing, and soldering are such that
(A) $T_{\text {welding }}>T_{\text {solddering }}>T_{\text {brazing }}$
(B) $T_{\text {soldering }}>T_{\text {welding }}>T_{\text {brazing }}$
(C) $T_{\text {brazing }}>T_{\text {welding }}^{\text {win }}>T_{\text {soldering }}$
(D) $T_{\text {welding }}>T_{\text {brazing }}>T_{\text {soldering }}$

Solution: $T_{\text {welding }}>T_{\text {Brazing }}>T_{\text {welding }}$.
Hence, the correct option is (D).

## Question Number: 32

Question Type: MCQ
The part of a gating system which regulates the rate of pouring of molten metal is
(A) pouring basin
(B) runner
(C) choke
(D) ingate

Solution: We know that the rate of pouring of molten metal is regulated by choke.
Hence, the correct option is (C).
Question Number: 33
Question Type: MCQ
The non-traditional machining process that essentially required vacuum is
(A) electron beam machining
(B) electrochemical machining
(C) electrochemical discharge machining
(D) electro discharge machining

Solution: We know that vacuum is required by electron beam machining.
Hence, the correct option is (A).

## Question Number: $34 \quad$ Question Type: NAT

In an orthogonal cutting process, the tool used has rake angle of zero degree. The measured cutting force and thrust force are 500 N and 250 N , respectively. The coefficient of friction between the tool and the chip is $\qquad$ —.

Solution: Thrust force $F_{\mathrm{t}}=250 \mathrm{~N}$
Cutting force $F_{\mathrm{c}}=500 \mathrm{~N}$
We know that

$$
\begin{aligned}
\tan (\beta-\alpha) & =\frac{F_{\mathrm{t}}}{F_{\mathrm{c}}} \\
\alpha & =0 \\
\therefore \quad \tan \beta & =\frac{F_{\mathrm{t}}}{F_{\mathrm{c}}}=\frac{250}{500}=0.5
\end{aligned}
$$

The coefficient of friction between the tool and the chip is

$$
\mu=\tan \beta=0.5
$$

Hence, the correct answer is 0.5 .
Question Number: 35
Question Type: MCQ
Match the following

| P. Feeler gauge | I. Radius of an object |
| :---: | :---: |
| Q. Fillet gauge | II. Diameter within limits by comparison |
| R. Snap gauge | III. Clearance or gap between components |
| S. Cylindrical plug gauge | IV. Inside diameter of straight hole |
| (A) P-III, Q-I, | II, S-IV (B) P-III, Q-II, R-I |
| (C) P-IV, Q-II, | -I, S-III (D) P-IV, Q-I, R-II, |

Solution: The correct mapping is
P. Feeler gauge III. Clearance between components
Q. Fillet gauge
I. Radius of an object
R. Snap gauge II. $\begin{aligned} & \text { Diameter within limits by } \\ & \text { comparison }\end{aligned}$
S. Cylindrical IV. Inside diameter of gauge plug straight hole
Hence, the correct option is (A).
Q. 26 to Q. 55 carry two marks each.

Question Number: 36
Question Type: NAT
26. Consider the function $f(x)=2 x^{3}-3 x^{2}$ in the domain $[-1,2]$. The global minimum of $f(x)$ is $\qquad$
Solution: Given $f(x)=2 x^{3}-3 x^{2}$ in the domain $[-1,2]$.

$$
\begin{array}{rlrl}
f(x) & =6 x^{2}-6 x \\
f(x) & =0 \Rightarrow 6 x^{2}-6 x=0 \Rightarrow x^{2}-x=0 \\
\Rightarrow \quad \xi(x-1) & =0 \\
\Rightarrow \quad \xi & & =0, x=1
\end{array}
$$

$\therefore$ The stationary values of $f(x)$ are $x=0$ and $x=1$.
$\therefore$ The global minimum of $f(x)$ in $[-1,2]$

$$
\begin{aligned}
& =\operatorname{Min}\{f(-1), f(0), f(1), f(2)\} \\
& =\operatorname{Min}\{-5,0,-1,4\}=-5
\end{aligned}
$$

Hence, the correct answer is -5 .
Question Number: 37
Question Type: NAT
If $y=f(x)$ satisfies the boundary value problem
$y^{\prime \prime}+9 y=0, y(0)=0, y(\pi / 2)=\sqrt{2}$, then $y(\pi / 4)$ is $\qquad$
Solution: Given boundary value problem is
$y^{\prime \prime}+9 y=0$
where $y(0)=0$ and $y\left(\frac{\pi}{2}\right)=\sqrt{2} y$
The auxiliary equation of (1) is

$$
D^{2}+9=0 \Rightarrow D= \pm 3 i
$$

$\therefore$ The general solution of $(1)$ is

$$
\begin{equation*}
y=c_{1} \cos 3 x+c_{2} \sin 3 x \tag{3}
\end{equation*}
$$

Given $y(0)=0$

$$
\begin{array}{cc}
\Rightarrow & 0=c_{1} \cos (3 \times 0)+c_{2} \sin (3 \times 0) \\
\Rightarrow & \chi_{1}=0
\end{array}
$$

Also, given

$$
y\left(\frac{\pi}{2}\right)=\sqrt{2}
$$

$\therefore$ From Eq. (3),

$$
\begin{aligned}
& \sqrt{2}=c_{1} \cos \left(3 \times \frac{\pi}{2}\right)+c_{2} \sin \left(3 \times \frac{\pi}{2}\right) \\
& \Rightarrow \sqrt{2}=c_{1} \times 0+c_{2} \times(-1) \\
& \Rightarrow c_{2}=-\sqrt{2}
\end{aligned}
$$

Substituting the values of $c_{1}$ and $c_{2}$ in Eq. (3), we get

$$
\begin{aligned}
y & =-\sqrt{2} \sin 3 x \\
\therefore y\left(\frac{\pi}{4}\right) & =-\sqrt{2} \sin \left(3 \times \frac{\pi}{4}\right) \\
& =-\sqrt{2} \sin \left(\frac{3 \pi}{4}\right)=-\sqrt{2} \times \frac{1}{\sqrt{2}} \\
\therefore y\left(\frac{\pi}{4}\right) & =-1
\end{aligned}
$$

Hence, the correct answer is -1 .
Question Number: 38 Question Type: MCQ
The value of the integral $\int_{-\infty}^{\infty} \frac{\sin x}{x^{2}+2 x+2} d x$ evaluated using contour integration and the residue theorem is
(A) $-\pi \sin (1) / e$
(B) $-\pi \cos (1) / e$
(C) $\sin (1) / e$
(D) $\cos (1) / e$

Solution: We have to evaluate $\int_{-\infty}^{\infty} \frac{\sin x}{x^{2}+2 x+2} d x$

Let

$$
f(z)=\frac{e^{i z}}{z^{2}+2 z+2}
$$

Consider the contour integral $\oint_{C} f(z) d z$

where $C$ is the contour consisting of the semi-circle $C_{R}$ : $|z|=R$, together with the diameter that closes it.
The singularities of $f(z)=\frac{e^{i z}}{z^{2}+2 z+2}$ are

$$
-1+i \text { and }-1-i
$$

Let $z_{1}=-1+i$ and $z_{2}=1-i$
Clearly $z_{1}$ lies inside the semi-circle but $z_{2}$ does not lie.
$\therefore$ By Cauchy's residue theorem, we have

$$
\begin{gather*}
\oint_{C} f(\mathrm{z}) d z=2 \pi i\left(\operatorname{Res}_{z=z_{1}}(f(z))\right)  \tag{1}\\
\operatorname{Res}(f(z))=\operatorname{Lt}_{z \rightarrow z_{1}}^{\operatorname{Lt}}\left[\left(z-z_{1}\right) f(z)\right] \\
\operatorname{Lt}_{z \rightarrow z_{1}}\left[\left(z-z_{1}\right) \frac{e^{i z}}{z^{2}+2 z+2}\right] \\
\therefore \operatorname{Res}_{z=z_{1}}[f(z)]=\frac{e^{-1-i}}{2 i}
\end{gather*}
$$

$\therefore$ From Eq. (1), we have

$$
\oint_{C} f(z) d z=\int_{C_{R}} f(z) d z+\int_{-R}^{R} f(x) d x=2 \pi i \times \frac{e^{-1-i}}{2 i}
$$

As $R \rightarrow \infty$, we have

$$
\int_{-\infty}^{\infty} f(x) d x=\pi\left(e^{-1-i}\right)
$$

$$
\Rightarrow \int_{-\infty}^{\infty} \frac{e^{i x}}{x^{2}+2 x+2} d x=\pi\left[e^{-1}\left(e^{-i}\right)\right]
$$

$$
\begin{gathered}
\Rightarrow \int_{-\infty}^{\infty} \frac{\cos x+i \sin x}{x^{2}+2 x+2} d x=\frac{\pi}{e}[\cos 1-i \sin 1] \\
\Rightarrow \int_{-\infty}^{\infty} \frac{\cos x}{x^{2}+2 x+2} d x+i \int_{-\infty}^{\infty} \frac{\sin x}{x^{2}+2 x+2} d x \\
=\frac{\pi}{e}[\cos 1-i \sin 1]
\end{gathered}
$$

Comparing the imaginary parts on both sides,

$$
\int_{-\infty}^{\infty} \frac{\sin x}{x^{2}+2 x+2} d x=-\frac{\pi}{e} \sin (1)
$$

Hence, the correct option is (A).
Question Number: 39
Question Type: NAT
Gauss-Seidel method is used to solve the following equations (as per the given order):

$$
\begin{aligned}
& x_{1}+2 x_{2}+3 x_{3}=5 \\
& 2 x_{1}+3 x_{2}+x_{3}=1 \\
& 3 x_{1}+2 x_{2}+x_{3}=3
\end{aligned}
$$

Assuming initial guess as $x_{1}=x_{2}=x_{3}=0$, the value of $x_{3}$ after the first iteration is $\qquad$
Solution: Given system of equations is

$$
\begin{aligned}
& x_{1}+2 x_{2}+3 x_{3}=5 \\
& 2 x_{1}+3 x_{2}+x_{3}=1 \\
& 3 x_{1}+2 x_{2}+x_{3}=3
\end{aligned}
$$

As the Gauss-Seidal method is used to solve the above equations as per the given order, we have to solve the first equation for $x_{1}$, the second equation for $x_{2}$, and the third equation for $x_{3}$.

$$
\therefore \quad \begin{align*}
& x_{1}=5-2 x_{2}-3 x_{3}  \tag{1}\\
& x_{2}=x_{2}=\frac{1}{3}-\frac{2}{3} x_{1}-\frac{x_{3}}{3} \tag{2}
\end{align*}
$$

and

$$
\begin{equation*}
x_{3}=3-3 x_{1}-2 x_{2} \tag{3}
\end{equation*}
$$

Given the initial guess values are $x_{1}=x_{2}=x_{3}=0$

$$
\text { i.e., } \quad x_{1}^{(0)}=0, x_{2}^{(0)}=0 \text { and } x_{3}^{(0)}=0
$$

$\therefore$ From Eq. (1), we have

$$
\begin{aligned}
x_{1}^{(1)}= & 5-2 x_{2}^{(0)}-3 x_{3}^{(0)} \\
= & 5-2 \times 0-3 \times 0 \\
& \therefore x_{1}^{(1)}=5
\end{aligned}
$$

From Eq. (2), we have

$$
\begin{gathered}
x_{2}^{(1)}=\frac{1}{3}-\frac{2}{3} x_{1}^{(1)}-\frac{1}{3} x_{3}^{(0)}=\frac{1}{3}-\frac{2}{3} \times 5-\frac{1}{3} \times 0 \\
\therefore x_{2}^{(1)}=-3
\end{gathered}
$$

From Eq. (3), we have

$$
\begin{aligned}
x_{3}^{(1)} & =3-3 x_{1}^{(1)}-2 x_{2}^{(1)} \\
& =3-3 \times 5-2 \times(-3) \\
& =3-15+6 \\
\therefore x_{3}^{(1)} & =-6
\end{aligned}
$$

Hence, the value of $x_{3}$ after the first iteration is $x_{3}^{(1)}=-6$ Hence, the correct answer is 6 .

## Question Number: 40

Question Type: NAT
A block of mass $m$ rests on an inclined plane and is attached by a string to the wall as shown in the figure. The coefficient of static friction between the plane and the block is 0.25 . The string can withstand a maximum force of 20 N . The maximum value of the mass $(m)$ for which the string will not break and the block will be in static equilibrium is _ kg.
Take $\cos \theta=0.8$ and $\sin \theta=0.6$
Acceleration due to gravity $g=10 \mathbf{m} / \mathbf{s}^{2}$


Solution: Coefficient of friction $\mu=0.25$
Maximum force $T_{\max }=20 \mathrm{~N}$

$$
\cos \theta=0.8
$$

$\sin \theta=0.6$


From the above figure, we conclude that

$$
\begin{aligned}
T+\mu m g \cos \theta & =m g \sin \theta \\
20+0.25 g(0.8 m) & =m g 0.6 \\
20 & =m g[0.6-0.25 \times 0.8] \\
\Rightarrow \quad m & =5 \mathrm{~kg}
\end{aligned}
$$

Hence, the correct answer is 5 .

## Question Number: 41

Question Type: MCQ A two-member truss $P Q R$ is supporting a load $W$. The axial forces in members $P Q$ and $Q R$ are respectively.

(A) $2 W$ tensile and $\sqrt{2} W$ compressive
(B) $\sqrt{3} W$ tensile and $2 W$ compressive
(C) $\sqrt{3} W$ compressive and $2 W$ tensile
(D) $2 W$ compressive and $\sqrt{3} W$ tensile

Solution: In the free-body diagram given below, consider the equilibrium of Pin Joint $Q$.


For equilibrium, we will have

$$
\begin{aligned}
T_{\mathrm{PQ}}+T_{\mathrm{QR}} \cos \theta & =0 \\
W+T_{\mathrm{QR}} \sin \theta & =0 \\
T_{\mathrm{QR}} & =\frac{-W}{\sin \theta}=\frac{-W}{0.5}=-2 W
\end{aligned}
$$

(Assume directions are opposite)
Aslo

$$
\begin{aligned}
T_{\mathrm{PQ}} & =-T_{\mathrm{QR}} \cos \theta \\
& =2 W \times \frac{\sqrt{3}}{2}=\sqrt{3} \mathrm{~W} \\
\therefore \quad & \\
T_{\mathrm{PQ}} & =\sqrt{3} \mathrm{~W} \text { tensile } \\
T_{\mathrm{QR}} & =2 W \text { Compressive. }
\end{aligned}
$$

Hence, the correct option is (B).
Question Number: 42
Question Type: NAT
A horizontal bar with a constant cross-section is subjected to loading as shown in the figure. The Young's moduli for the sections $A B$ and $B C$ are $3 E$ and $E$, respectively.


For the deflection at $C$ to be zero, the ratio $P / F$ is

Solution: For equilibrium of forces, we have

$$
\begin{gather*}
R_{\mathrm{A}}+P=F  \tag{1}\\
\delta_{\mathrm{C}}=0 \\
\frac{R_{\mathrm{A}} L}{A(3 E)}+\frac{\left(R_{\mathrm{A}}+P\right) L}{A E}=0  \tag{2}\\
\frac{R_{\mathrm{A}}}{3}+\frac{\left(R_{\mathrm{A}}+P\right)}{1}=0 \\
4 R_{\mathrm{A}}=-3 P \\
R_{\mathrm{A}}=\frac{-3}{4} P \tag{3}
\end{gather*}
$$

Substituting Eq. (3) in Eq. (1), we get

$$
\begin{aligned}
\frac{1}{4} P & =F \\
\frac{P}{F} & =4
\end{aligned}
$$

Hence, the correct answer is 4.
Question Number: 43
Question Type: NAT
The figure shows cross-section of a beam subjected to bending. The area moment of inertia (in $\mathrm{mm}^{4}$ ) of this crosssection about its base is $\qquad$ —.


Solution: Using Parallel Axis theorem, we get

$$
\begin{aligned}
A_{\mathrm{I}} & =\left(\frac{10 \times 10^{3}}{12}+10 \times 10 \times 5^{2}\right)-\left[\left(\frac{\pi}{64} \times 8^{4}\right)+\pi \times 4^{2} \times 5^{2}\right] \\
& =3333.33-1457.69 \\
& =1875.63 \mathrm{~mm}^{4}
\end{aligned}
$$

Hence, the correct answer is 1873 to 1879 .
Question Number: 44
Question Type: NAT
A simply-supported beam of length $3 L$ is subjected to the loading shown in the figure.
It is given that $P=1 \mathrm{~N}, L=1 \mathrm{~m}$, and Young's modulus $E$ $=200 \mathrm{GPa}$. The cross-section is a square with dimension
$10 \mathrm{~mm} \times 10 \mathrm{~mm}$. The bending stress (in Pa ) at the point $A$ located at the top surface of the beam at a distance of 1.5 L from the left end is $\qquad$


Solution: Young's modulus $E=200 \mathrm{GPa}$
Force $P=1 \mathrm{~N}$
Length $L=1 \mathrm{~m}$


For moment at point $B$,

$$
\begin{gathered}
R_{\mathrm{C}} \times 3 L-P \times 2 L+P \times L=0 \\
R_{\mathrm{C}}=\frac{P}{3} \\
R_{\mathrm{B}}=\frac{-P}{3}
\end{gathered}
$$



For moment at $A=-R_{\mathrm{B}}(1.5 L)+P \times 0.5 L$

$$
=-\frac{P}{3}(1.5 L)+P \times 0.5 L=0 \mathrm{~N}-\mathrm{m}
$$

Therefore, bending stress at point $A$ is zero.
Hence, the correct answer is 0 .
Question Number: 45
Question Type: NAT
A slider crank mechanism with crank radius 200 mm and connecting rod length 800 mm is shown. The crank is rotating at 600 rpm in the counterclockwise direction. In the configuration shown, the crank makes an angle of $90^{\circ}$ with the sliding direction of the slider, and a force of 5 kN is
acting on the slider. Neglecting the inertia forces, the turning moment on the crank (in $\mathrm{kN}-\mathrm{m}$ ) is $\qquad$ —.


## Solution:



Crank radius $r=200 \mathrm{~mm}$
Rod length $L=800 \mathrm{~mm}$
Force $F=5 \mathrm{KN}=5 \times 10^{3} \mathrm{~N}$
Turning moment on crank

$$
\begin{aligned}
& =\frac{F}{\cos \theta} \sin (90+\theta) r \\
& =\frac{F \cos \theta}{\cos \theta}(r) \\
& =F r=5 \times 10^{3} \times 0.2 \\
& =1000 \mathrm{~N}-\mathrm{m} \approx 1 \mathrm{kNm}
\end{aligned}
$$

Hence, the correct answer is 1 .
Question Number: 46
Question Type: NAT
In the gear train shown, gear 3 is carried on arm 5. Gear 3 meshes with gear 2 and gear 4 . The number of teeth on gears 2,3 , and 4 are 60,20 , and 100 , respectively. If gear 2 is fixed and gear 4 rotates with an angular velocity of 100 rpm in the counterclockwise direction, the angular speed of arm 5 (in rpm) is

(A) 166.7 counterclockwise
(B) 166.7 clockwise
(C) 62.5 counterclockwise
(D) 62.5 clockwise

## Solution:



We know that
$N_{4}=100 \mathrm{rpm} \mathrm{C.C.W}$
$N_{5}=$ ?
Let the angular speed of gear 2 be $x$ and $y$ be the angular speed of arm. Consider the table given below:

| Gear | 2 | 3 | 4 |
| :--- | :--- | :--- | :--- |
| Teeth | 60 | 20 | 100 |
| Without <br> arm speed | 1 | $\frac{-60}{20}$ | $\frac{-60}{20} \times \frac{20}{100}$ |
| $x$ rev | $x$ | $-x\left(\frac{60}{20}\right)$ | $\frac{(x) 60}{20} \times \frac{20}{100}$ |
| With arm <br> speed | $y+x$ | $y-3 x$ | $y-\frac{6 x}{10}$ |
|  | $y+x=0$ |  |  |
| $y-\frac{3}{5} x=100$ |  |  |  |

From Eqs. (1) and (2), we get

$$
\begin{aligned}
y+\frac{3}{5} y & =100 \\
8 y & =500 \\
y & =62.5 \\
x & =-62.5
\end{aligned}
$$

and
Thus, speed of arm $\left(N_{5}\right)=62.5$ C.C.W.
Hence, the correct option is (C).
Question Number: 47
Question Type: MCQ
A solid disc with radius $a$ is connected to a spring at a point $d$ above the center of the disc. The other end of the spring is fixed to the vertical wall. The disc is free to roll without
slipping on the ground. The mass of the disc is $M$ and the spring constant is $K$. The polar moment of inertia for the disc about its center is $J=M a^{2} / 2$.


The natural frequency of this system in rad/s is given by
(A) $\sqrt{\frac{2 K(a+d)^{2}}{3 M a^{2}}}$
(B) $\sqrt{\frac{2 K}{3 M}}$
(C) $\sqrt{\frac{2 K(a+d)^{2}}{M a^{2}}}$
(D) $\sqrt{\frac{K(a+d)^{2}}{M a^{2}}}$

Solution: Consider the figure given below:


If $\theta$ be an angular deflection of disc

$$
x=r \theta \text { and } r=a+d
$$

Moment of inertia of disc

$$
I=\frac{m a^{2}}{2}+m a^{2}=\frac{3}{2} m a^{2}
$$

Applying torque equation,

$$
\begin{gathered}
I \ddot{\theta}+k x y=0 \\
\frac{3}{2} m a^{2} \ddot{\theta}+k(a+d)^{2} \theta=0 \\
\omega_{\mathrm{n}}=\sqrt{\frac{2 k(a+d)^{2}}{3 m a^{2}}}
\end{gathered}
$$

Hence, the correct option is (A).
Question Number: 48
Question Type: MCQ
The principal stresses at a point inside a solid object are $\sigma_{1}=100 \mathrm{MPa}, \sigma_{2}=100 \mathrm{MPa}$, and $\sigma_{3}=0 \mathrm{MPa}$. The yield
strength of the material is 200 MPa . The factor of safety calculated using Tresca (maximum shear stress) theory is $n_{\mathrm{T}}$ and the factor of safety calculated using von Mises (maximum distortional energy) theory is $n_{\mathrm{v}}$. Which one of the following relations is TRUE?
(A) $n_{\mathrm{T}}=(\sqrt{3} / 2) n_{\mathrm{V}}$
(B) $n_{\mathrm{T}}=(\sqrt{3}) n_{\mathrm{V}}$
(C) $n_{\mathrm{T}}=n_{\mathrm{V}}$
(D) $n_{\mathrm{V}}=(\sqrt{3}) n_{\mathrm{T}}$

Solution: Principal stresses $\sigma_{1}=100 \mathrm{MPa}$
Principal stresses $\sigma_{2}=100 \mathrm{MPa}$
Principal stresses $\sigma_{3}=0$
Yield strength $S_{\mathrm{yT}}=200 \mathrm{MPa}$
According to Tresca's theory,

$$
\begin{align*}
\sigma_{1} & \leq S_{\mathrm{yT}} / N \\
100 & \leq 200 / n_{\mathrm{T}} \tag{1}
\end{align*}
$$

According to Von Mises theory,

$$
\begin{align*}
\left(\sigma_{1}\right. & \left.-\sigma_{2}\right)^{2}+\left(\sigma_{2}-\sigma_{3}\right)^{2}+\left(\sigma_{3}-\sigma_{1}\right)^{2} \\
& =2\left(\frac{S_{\mathrm{yT}}}{n_{\mathrm{V}}}\right)^{2}=2\left(\sigma_{1}\right)^{2}=2\left(\frac{S_{\mathrm{yT}}}{n_{\mathrm{V}}}\right)^{2} \\
& =\sigma_{1}=\frac{S_{\mathrm{yT}}}{n_{\mathrm{V}}} \tag{2}
\end{align*}
$$

From Eqs. (1) and (2), we get

$$
n_{\mathrm{T}}=n_{\mathrm{V}}
$$

Hence, the correct option is (C).
Question Number: 49
Question Type: NAT
An inverted U-tube manometer is used to measure the pressure difference between two pipes $A$ and $B$, as shown in the figure. Pipe $A$ is carrying oil (specific gravity $=0.8$ ) and pipe $B$ is carrying water. The densities of air and water are $1.16 \mathrm{~kg} / \mathrm{m}^{3}$, respectively. The pressure difference between pipes $A$ and $B$ is $\qquad$ kPa .
Acceleration due to gravity $g=10 \mathrm{~m} / \mathrm{s}^{2}$.


Solution:
Density of oil $\rho_{\text {oil }}=800 \mathrm{~kg} / \mathrm{m}^{3}$
Density of water $\rho_{\text {water }}=1000 \mathrm{~kg} / \mathrm{m}^{3}$
Density of air $\rho_{\text {air }}=1.16 \mathrm{~kg} / \mathrm{m}^{3}$
Acceleration due to gravity $g=10 \mathrm{~m} / \mathrm{s}^{2}$


$$
\begin{array}{cc} 
& P_{\mathrm{A}}-\rho_{\text {oil }} g(0.2)-\rho_{\text {air }} g(0.08)+\rho_{\text {water }} \\
& g(0.38)=P_{\mathrm{B}} \\
\therefore \quad & P_{\mathrm{B}}-P_{\mathrm{A}}=2.199 \mathrm{kPa}
\end{array}
$$

Hence, the correct answer is -2.2 kPa .
Question Number: 50
Question Type: NAT
40. Oil (kinematic viscosity, $v_{\text {oil }}=1.0 \times 10^{-5} \mathrm{~m}^{2} / \mathrm{s}$ ) flows through a pipe of 0.5 m diameter with a velocity of $10 \mathrm{~m} / \mathrm{s}$. Water (kinematic viscosity, $v_{\mathrm{w}}=0.89 \times 10^{-6} \mathrm{~m}^{2} / \mathrm{s}$ ) is flowing through a model pipe of diameter 20 mm . For satisfying the dynamic similarity, the velocity of water (in $\mathrm{m} / \mathrm{s}$ ) is

Solution: Kinematic viscosity of oil $v_{\text {oil }}=1 \times 10^{-5} \mathrm{~m}^{2} / \mathrm{s}$
Diameter of pipe $d_{\text {oil }}=0.5 \mathrm{~m}$
Velocity $V_{\text {oil }}=10 \mathrm{~m} / \mathrm{s}$
Kinematic viscosity of water $v_{\text {water }}=0.89 \times 10^{-6} \mathrm{~m}^{2} / \mathrm{s}$
Diameter of pipe $d_{\text {water }}=0.02 \mathrm{~m}$
Equating Reynolds number, we get

$$
\begin{gathered}
\frac{V_{\text {oil }} d_{\text {oil }}}{v_{\text {oil }}}=\frac{V_{\text {water }} d_{\text {water }}}{v_{\text {water }}} \\
\Rightarrow \frac{0.5 \times 10}{10^{-5}}=\frac{V_{\text {water }} \times 0.02}{0.89 \times 10^{-6}} \\
V_{\text {water }}=22.25 \mathrm{~m} / \mathrm{s}
\end{gathered}
$$

Hence, the correct answer is 22 to 22.5 .
Question Number: 51
Question Type: NAT
A steady laminar boundary layer is formed over a flat plate as shown in the figure. The free stream velocity of the fluid
is $U_{0}$. The velocity profile at the inlet $a-b$ is uniform, while that at the downstream location $c-d$ given by

$$
u=U_{\mathrm{o}}\left[2\left(\frac{y}{\delta}\right)-\left(\frac{y}{\delta}\right)^{2}\right]
$$



The ratio of the mass flow rate, $\dot{m}_{b d}$, leaving through the horizontal section $b-d$ to that entering through the vertical section $a-b$ is $\qquad$
Solution: From the above figure, we get


$$
\begin{gathered}
\dot{m}_{a b}=\rho u_{\mathrm{o}} \delta \\
\dot{m}_{c d}=\int_{0}^{\delta} \rho u_{o}\left[2\left(\frac{y}{\delta}\right)-\left(\frac{y}{\delta}\right)^{2}\right] d y \\
=\rho u_{\mathrm{o}}\left[\delta-\frac{\delta}{3}\right] \\
=\frac{2}{3} \rho u_{\mathrm{o}} \delta \\
\therefore \dot{m}_{b d}=\frac{1}{3} \rho u_{\mathrm{o}} \delta
\end{gathered}
$$

As

$$
\begin{gathered}
\dot{m}_{a b}=\dot{m}_{c d}+\dot{m}_{b d} \\
\therefore \frac{\dot{m}_{b d}}{\dot{m}_{a b}}=\frac{1}{3} \approx 0.333
\end{gathered}
$$

Hence, the correct answer is $0.32-0.34$.
Question Number: 52
Question Type: NAT
42. A steel ball of 10 mm diameter at 1000 K is required to be cooled to 350 K by immersing it in a water environment at 300 K . The convective heat transfer coefficient is $1000 \mathrm{~W} /$ $\mathrm{m}^{2}-\mathrm{K}$. Thermal conductivity of steel is $40 \mathrm{~W} / \mathrm{mK}$. The time
constant for the cooling process $\tau$ is 16 s . The time required (in s) to reach the final temperature is $\qquad$
Solution: Diameter of steel ball $d=10 \mathrm{~mm}$
Initial temperature $T_{0}=1000 \mathrm{~K}$
Final temperature $T_{1}=350 \mathrm{~K}$
Temperature of water $T_{\infty}=300 \mathrm{~K}$
Convective heat transfer coefficient $h=1000 \mathrm{~W} / \mathrm{m}^{2}-\mathrm{K}$
Thermal conductivity of steel $k=40 \mathrm{~W} / \mathrm{m}-\mathrm{K}$
Time constant $\tau=16 \mathrm{~s}$
We know that the time constant can be expressed as

$$
\tau=\frac{\rho V C}{h A}=16 \mathrm{~s}
$$

$$
\begin{aligned}
& \frac{T_{0}-T_{\infty}}{T_{1}-T_{\infty}}=e^{(h A / \rho V C)} t \\
& \Rightarrow \frac{1000-300}{350-300}=e^{t / 16} \\
& \quad t=42.22 \mathrm{~s}
\end{aligned}
$$

Hence, the correct answer is $42-43$.
Question Number: 53
Question Type: MCQ
An infinitely long furnace of $0.5 \mathrm{~m} \times 0.4 \mathrm{~m}$ cross-section is shown in the figure below. Consider all surfaces of the furnace to be black. The top and bottom walls are maintained at temperature $T_{1}=T_{3}=927^{\circ} \mathrm{C}$ while the side walls are at temperature $T_{2}=T_{4}=527^{\circ} \mathrm{C}$. The view factor, $F_{12}$ is 0.26 . The net radiation heat loss or gain on side 1 is $\qquad$ W/m.
Stefan-Boltzmann constant $=5.67 \times 10^{-8} \mathrm{~W} / \mathrm{m}^{2}-\mathrm{K}^{4}$


Solution: Temperature $T_{1}=T_{3}=1200 \mathrm{~K}$
Temperature $T_{2}=T_{4}=800 \mathrm{~K}$
View factor $F_{12}=0.26$
For symmetry, $F_{12}=F_{14}=0.26$

Therefore, the flow is laminar.
Hydrodynamic boundary layer at 0.5 will be

$$
\begin{gathered}
=\frac{5 x}{\sqrt{R_{\mathrm{ex}}}}=\frac{5 \times 0.5}{\sqrt{1.6667 \times 10^{5}}} \approx 6.12 \times 10^{-3} \mathrm{~m} \\
\delta_{t h}=6.12 \mathrm{~mm}
\end{gathered}
$$

Hence, the correct answer is 6 to 6.25
Question Number: 55
Question Type: NAT
45. For water at $25^{\circ} \mathrm{C}, d p_{\mathrm{s}} / d T_{\mathrm{s}}=0.189 \mathrm{kPa} / \mathrm{K}\left(P_{\mathrm{s}}\right.$ is the saturation pressure in kPa and $T_{\mathrm{s}}$ is the saturation temperature in K ) and the specific volume of dry saturated vapor is $43.38 \mathrm{~m}^{3} / \mathrm{kg}$. Assume that the specific volume of liquid is negligible in comparison with that of vapor. Using the Clausius-Clapeyron equation, an estimate of the enthalpy of evaporation of water at $25^{\circ} \mathrm{C}$ (in $\mathrm{kJ} / \mathrm{kg}$ ) is $\qquad$
Solution: Temperature of water $T_{\text {sat }}=25^{\circ} \mathrm{C}$

$$
\frac{d P_{\mathrm{s}}}{d T_{\mathrm{s}}}=0.189 \mathrm{k} \mathrm{~Pa} / \mathrm{K}
$$

Specific volume of dry saturated vapor $v=43.38 \mathrm{~m}^{3} / \mathrm{kg}$

$$
\begin{aligned}
\frac{d P_{\mathrm{s}}}{d T_{\mathrm{s}}} & =\frac{h_{\mathrm{fg}}}{T_{\mathrm{sat}} V_{\mathrm{g}}}\left(\text { Here }, v_{\mathrm{g}}=v_{\mathrm{fg}}\right) \\
0.189 \times 10^{3} & =\frac{h_{\mathrm{fg}}}{298 \times 43.38} \\
h_{\mathrm{fg}} & =2443.24 \mathrm{~kJ} / \mathrm{kg}
\end{aligned}
$$

Hence, the correct answer is $2400-2500$.

## Question Number: 56

Question Type: NAT
An ideal gas undergoes a reversible process in which the pressure varies linearly with volume. The conditions at the start (subscript 1) and at the end (subscript 2) of the process with usual notation are: $p_{1}=100 \mathrm{kPa}, V_{1}=0.2 \mathrm{~m}^{3}$, and $p_{2}$ $=200 \mathrm{kPa}, V_{2}=0.1 \mathrm{~m}^{3}$ and the gas constant, $R=0.275 \mathrm{~kJ} /$ $\mathrm{kg}-\mathrm{K}$. The magnitude of the work required for the process (in kJ ) is $\qquad$


Solution: Gas constant $R=0.275 \mathrm{~kJ} / \mathrm{kg}-\mathrm{K}$
Pressure $p_{1}=100 \mathrm{kPa}$

Volume $V_{1}=0.2 \mathrm{~m}^{3}$
Pressure $p_{2}=200 \mathrm{kPa}$
Volume $V_{2}=0.1 \mathrm{~m}^{3}$
Work done $=$ area under the curve 1,2

$$
=\frac{1}{2} \times 100 \times 10^{3} \times 0.1+100 \times 10^{3} \times 0.1=15,000 \mathrm{~J}
$$

Hence, the correct answer is 15 .

## Question Number: 57

## Question Type: NAT

In a steam power plant operating on an ideal Rankine cycle, superheated steam enters the turbine at 3 MPa and $350^{\circ} \mathrm{C}$. The condenser pressure is 75 kPa . The thermal efficiency of the cycle is $\qquad$ percent.
Given data:
For saturated liquid, at $P=75 \mathrm{kPa}, h_{\mathrm{f}}$

$$
\begin{aligned}
& =384.39 \mathrm{~kJ} / \mathrm{kg}, v_{\mathrm{f}}=0.001037 \mathrm{~m}^{3} / \mathrm{kg}, s_{\mathrm{f}} \\
& =1.213 \mathrm{~kJ} / \mathrm{kg}-\mathrm{K} \\
h_{\mathrm{fg}} & =2278.6 \mathrm{~kJ} / \mathrm{kg}, \\
s_{\mathrm{fg}} & =6.2434 \mathrm{~kJ} / \mathrm{kg}-\mathrm{K}
\end{aligned}
$$

At $75 \mathrm{kPa}, \quad h_{\mathrm{fg}}=2278.6 \mathrm{~kJ} / \mathrm{kg}$,

At $P=3 \mathrm{MPa}$ and $T=350^{\circ} \mathrm{C}$ (superheated steam), $h=$ $3115.3 \mathrm{~kJ} / \mathrm{kg}$,

$$
s=6.7428 \mathrm{~kJ} / \mathrm{kg}-\mathrm{K}
$$

## Solution:



From the figure we conclude that process $1-2$ is isentropic process; therefore,

$$
\begin{aligned}
S_{2} & =S_{1} \\
S_{2} & =S_{f_{1}}+\mathrm{x} \mathrm{~S}_{\mathrm{fg}} \\
x & =0.886 \\
h_{1} & =h_{f_{1}}+x h_{f g_{1}} \\
& =384.39+0.886 \times 2278.6 \\
& =2403.23 \mathrm{~kJ} / \mathrm{kg}
\end{aligned}
$$

The thermal efficiency of the cycle is

$$
\eta=\frac{h_{2}-h_{1}}{h_{2}-h_{5}}=\frac{3115.3-2403.23}{3115.3-384.39}=0.2607
$$

$$
\eta=26.07 \%
$$

Hence, the correct answer is 25.8 to 26.
Question Number: 58
Question Type: MCQ
A hypothetical engineering stress-strain curve shown in the figure has three straight lines $P Q, Q R, \mathrm{RS}$ with coordinates $P(0,0), Q(0.2,100), R(0.6,100), R(0.6,140)$, and $S(0.8,130)$. ' $Q$ ' is the yield point, ' $R$ ' is the UTS point and, ' $S$ ' the fracture point.


The toughness of the material (in $\mathrm{MJ} / \mathrm{m}^{3}$ ) is $\qquad$
Solution: Consider the figure given below:


Toughness $=$ Area under the curve

$$
\begin{aligned}
& 1+2+3+4+5 \\
& =\frac{1}{2} \times \frac{0.2}{100} \times 100+\frac{0.4}{100} \times 100+\frac{0.2}{100} \times 130 \\
& =\frac{1}{2} \times 10 \times \frac{0.2}{100}+\frac{1}{2} \times 40 \times \frac{0.4}{100}=0.85 \mathrm{MJ} / \mathrm{m}^{3}
\end{aligned}
$$

Hence, the correct answer is 0.85 .
Question Number: 59
Question Type: NAT
Heat is removed from a molten metal of mass 2 kg at a constant rate of 10 kW till it is completely solidified. The cooling curve is shown in the figure.

Assuming uniform temperature throughout the volume of the metal during solidification, the latent heat of fusion of the metal (in $\mathrm{kJ} / \mathrm{kg}$ ) is $\qquad$


Solution: Mass of molten metal $m=2 \mathrm{~kg}$
Constant rate $\dot{Q}=10 \mathrm{~kW}$
Total heat removed during phase change

$$
=10 \times 10=100 \mathrm{~kJ}
$$

$$
\text { Time }=10 \mathrm{~s}
$$

Latented heat $\frac{\mathrm{kJ}}{\mathrm{kg}}=\frac{100}{2}=50 \frac{\mathrm{~kJ}}{\mathrm{~kg}}$
Hence, the correct answer is 50 .
Question Number: 60
Question Type: MCQ
The tool life equation for HSS tool is $V T^{0.14} f^{0.7} d^{0.4}=$ constant. The tool life $(T)$ of 30 min is obtained using the following cutting conditions:

$$
V=45 \mathrm{~m} / \mathrm{min}, f=0.35 \mathrm{~mm}, d=2.0 \mathrm{~mm}
$$

If speed $(V)$, feed $(f)$ and depth of cut $(d)$ are increased individually by $25 \%$, the tool life (in min ) is
(A) 0.15
(B) 1.06
(C) 22.50
(D) 30.0

Solution: The cutting conditions are given as

$$
\begin{aligned}
V_{1} & =45 \mathrm{~m} / \mathrm{min} \\
f_{1} & =0.35 \mathrm{~mm} \\
d_{1} & =2 \mathrm{~mm} \\
T_{1} & =30 \mathrm{~min}
\end{aligned}
$$

When the speed is increased individually by $25 \%$, then we have cutting conditions as given below:

$$
\begin{aligned}
V_{2} & =1.25 \mathrm{~V}_{1} \\
f_{2} & =1.25 \mathrm{f}_{1} \\
d_{1} & =1.25 \mathrm{~d}_{1} \\
T_{2} & =?
\end{aligned}
$$

Now we have $V T^{0.14} f^{0.7} d^{0.4}=C$

$$
\begin{aligned}
& V_{1} T_{1}^{0.14} f_{1}^{0.7} d_{1}^{0.4}=V_{2} T_{2}^{0.14} f_{2}^{0.7} d_{2}^{0.4} \\
& \Rightarrow T_{1}^{0.14}=1.25 \times 1.25^{0.7} \times 1.25^{0.4} T_{2}^{0.14}
\end{aligned}
$$

$$
\therefore T_{2}=\frac{30}{(1.25)^{2.1 / 0.14}} \approx 1.055 \approx 1.06
$$

Hence, the correct option is (B).
Question Number: 61
Question Type: MCQ
A cylindrical job with diameter of 200 mm and height of 100 mm is to be cast using modulus method of riser design. Assume that the bottom surface of cylindrical riser does not contribute as cooling surface. If the diameter of the riser is equal to its height, then the height of the riser (in mm ) is
(A) 150
(B) 200
(C) 100
(D) 125

Solution: Diameter of cylindrical job $d_{\text {casting }}=200 \mathrm{~mm}$ Height of cylindrical job $h_{\text {casting }}=100 \mathrm{~mm}$ We know that $d_{\text {riser }}=h_{\text {riser }}$

$$
\left(\frac{V}{S . A}\right)_{\text {riser }}=1.2\left(\frac{V}{S . A}\right)_{\text {casting }}
$$

$$
\begin{gathered}
=\frac{\left(\frac{\pi}{4} d_{\text {riser }}^{2} h_{\text {riser }}\right)}{\pi d_{\text {riser }} h_{\text {riser }}+\frac{\pi}{4} d_{\text {riser }}^{2}} \\
=1.2 \frac{\frac{\pi}{4} d_{\text {casting }}^{2} h_{\text {casting }}}{\pi d_{\text {casting }} h_{\text {casting }}+2 \frac{\pi}{4} d_{\text {casting }}^{2}} \\
\frac{4 h_{\text {riser }}^{3}}{4 \pi h_{\text {riser }}^{2}+\pi h_{\text {riser }}^{2}}=\frac{1.2 \times 200^{2} \times 100 \times 2}{2 \pi 200 \times 100+\pi \times 200^{2}} \\
\Rightarrow \frac{4 h_{\text {riser }}}{5}=\frac{1.2 \times 200^{2} \times 100 \times 2}{2 \times 200 \times 100+200^{2}} \\
h_{\text {riser }}=150 \mathrm{~mm}
\end{gathered}
$$

Hence, the correct option is (A).
Question Number: 62
Question Type: NAT
A 300 mm thick slab is being cold rolled using roll of 600 mm diameter. If the coefficient of friction is 0.08 , the maximum possible reduction (in mm ) is $\qquad$
Solution: Thickness of slab $t_{1}=300 \mathrm{~mm}$
Diameter of roll $r=300 \mathrm{~mm}$
Coefficient of friction $\mu=0.08$

$$
t_{2}=?
$$

Now we know that

$$
\begin{aligned}
t_{1}-t_{2} & =\mu^{2} R=\text { max. possible reduction } \\
& =0.08^{2} \times 300=1.92 \mathrm{~mm}
\end{aligned}
$$

Hence, the correct answer is $1.9-1.94 \mathrm{~mm}$.

## Question Number: 63

Question Type: MCQ
The figure below represents a triangle $P Q R$ with initial coordinates of the vertices as $P(1,3), Q(4,5)$, and $R(5,3$, 5). The triangle is rotated in the $X-Y$ plane about the vertex $P$ by angle $\theta$ in the clockwise direction. If $\sin \theta=0.6$ and $\cos \theta=0.8$, then new coordinates of the vertex $Q$ are

(A) $(4.6,2.8)$
(B) $(3.2,406)$
(C) $7.9,5.5)$
(D) $(5.5,7.9)$

Solution: If we translate the point $P$ to origin
New coordinates at $P$ are $(0,0)$
New coordinates of $Q$ are $(3,2)$
Now rotating $P Q$ about $P$ by $\theta$
New coordinates of $Q$ are

$$
\begin{aligned}
X^{1} & =x_{\mathrm{Q}} \cos \theta+y_{\mathrm{Q}} \sin \theta \\
& =3 \times 0.8+2 \times 0.6=3.6 \\
Y^{1} & =-X_{\mathrm{Q}} \sin \theta+y_{\mathrm{Q}} \cos \theta \\
& =-3 \times 0.6+2 \times 0.8=-0.2
\end{aligned}
$$

Now, again translating $P$ to original condition,
New coordinates of the vertex $Q$ becomes (3.6 $+1,3-0.2$ ) $=(4.6,2.8)$
Hence, the correct option is (A).
Question Number: 64
Question Type: NAT
The annual demand for an item is 10,000 units. The unit cost is Rs. 100 and inventory carrying charges are $14.4 \%$ of the unit cost per annum. The cost of one procurement is Rs. 2000. The time between two consecutive orders to meet the above demand is $\qquad$ month(s).

Solution: Annual demand of item $D=10,000$
Unit cost of item $C=$ Rs. 100
Inventory carrying charges $C_{\mathrm{h}}=0.144 \times 10$
Cost of one procurement $C_{\mathrm{o}}=$ Rs. 2000

Now using the relation

$$
\begin{aligned}
E O Q & =\sqrt{2 D \frac{C_{\mathrm{o}}}{C_{\mathrm{h}}}}=\sqrt{\frac{2 \times 10,000 \times 2000}{0.144 \times 100}} \\
& =1666.6667
\end{aligned}
$$

No. of orders $=\frac{10,000}{1666.6667}=6$
Time between orders

$$
=\frac{\text { No. of month in a year }}{\text { No.of orders }}=\frac{12}{6}=2
$$

Hence, the correct answer is 2 .
Question Number: 65
Question Type: MCQ
Maximize $Z=15 X_{1}+20 X_{2}$ subject to

$$
\begin{array}{r}
12 X_{1}+4 X_{2} \geq 36 \\
12 X_{1}-6 X_{2} \leq 24 \\
X_{1}, X_{2} \geq 0
\end{array}
$$

The above linear programming problem has
(A) infeasible solution
(B) unbounded solution
(C) alternative optimum solutions
(D) degenerate solution

Solution: Maximize $Z=15 X_{1}+20 X_{2}$

$$
\begin{array}{r}
\text { Subject to } 12 X_{1}+4 X_{2} \geq 36 \\
12 X_{1}+6 X_{2} \geq 36
\end{array}
$$


$\therefore$ The region is unbounded.
Hence, the correct option is (B).

# GATE 2016 Solved Paper ME: Mechanical Engineering Set - 2 

## Number of Questions: 65

Total Marks:100.0

Wrong answer for MCQ will result in negative marks, (-1/3) for 1 Mark Questions and (-2/3) for 2 Marks Questions.

## General Aptitude

## Number of Questions: 10

Section Marks: 15.0

## Q. 1 to Q. 5 carry 2 mark each and Q. 6 to Q. 10 carry 1 marks each.

## Question Number: 1 <br> Question Type: MCQ

Students taking an exam are divided into two groups, $P$ and $Q$ such that each group has the same number of students. The performance of each of the students in a test was evaluated out of 200 marks. It was observed that the mean of group $P$ was 105 , while that of group $Q$ was 85 . The standard deviation of group $P$ was 25 , while that of group $Q$ was 5 . Assuming that the marks were distributed on a normal distribution, which of the following statements will have the highest probability of being TRUE?
(A) No student in group $Q$ scored fewer marks than any student in group $P$.
(B) No student in group $P$ scored fewer marks than any student in group $Q$.
(C) Most students of group $Q$ scored marks in a narrower range than students in group $P$.
(D) The median of the marks of group $P$ is 100 .

Solution: Consider the table given below:

|  | $Q$ | $P$ |
| :--- | :--- | :--- |
| Mean | 85 | 105 |
| Standard <br> deviation | 5 | 25 |

For $Q, 68 \%$ of the scores are in the interval $80-90$ and $99.7 \%$ of the scores are in the interval $70-100$.
For $P, 68 \%$ of the scores are in the interval $80-130$. $99.7 \%$ of the scores are in the interval $30-180$.

For option (A), the probability is 0 .
For options (B, C), (B) starts with 'No student' while (C) begins with 'most students'. Even before reading the options completely, we can say that (C) is more probable than (B). But we do need to read the options completely. When we do that, we do not see anything to change our conclusion.

For option (D), the median of $P$ is 100 . For a normal distribution, the mean, median, and mode coincide, i.e. the median is actually 85 . Therefore, the probability of (D) is 0 .

Hence, the correct option is (C).
Question Number: 2
Question Type: MCQ
A smart city integrates all modes of transport, uses clean energy, and promotes sustainable use of resources. It also uses technology to ensure safety and security of the city, something which critics argue, will lead to a surveillance state.
Which of the following can be logically inferred from the above paragraph?
(i) All smart cities encourage the formation of surveillance states.
(ii) Surveillance is an integral part of smart city.
(iii) Sustainability and surveillance go hand-in-hand in a smart city.
(iv) There is a perception that smart cities promote surveillance.
(A) (i) and (iv) only
(B) (ii) and (iii) only
(C) (iv) only
(D) (i) only

Solution: We know that a smart city would lead to a surveillance state is merely the perception of critics; therefore, it cannot be said that sustainability and surveillance go hand-in-hand or surveillance is an integral part of a smart city. All smart cities encourage the formation of surveillance state; a surveillance state could be a possible outcome of a smart city. Hence, options (i), (ii), and (iii) are incorrect.

Hence, the correct option is (C).
Question Number: 3
Question Type: MCQ
Find the missing sequence in the letter series.
B, FH, LNP, $\qquad$
(A) SUWY
(B) TUVW
(C) TVXZ
(D) TWXZ

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## Solution:



So, it is 'TVXZ'
Hence, the correct option is (C).

## Question Number: 4

Question Type: MCQ
The binary operation o is defined $a$ o $b=a b+(a+b)$, where $a$ and $b$ are any two real numbers. The value of the identity element of this operation, defined as the number $x$ such that $a$ o $x=a$, for any $a$, is
(A) 0
(B) 1
(C) 2
(D) 10

Solution: The binary operation o is defined as$b=a b+(a+b)$
Let $x$ be the identity element
i.e., $a \square x=a x+(a+x)=a$ for all values of $a$.
$\Rightarrow x(a+1)=0 \Rightarrow x=0$ [The equality holds for all values of $a$ and not just for $a=-1$.]
The identify element for this operation is 0 .
Hence, the correct option is (A).
Question Number: 5
Question Type: MCQ
Which of the following curves represent the function $y=$ $\ln \left(\left|e^{[\sin (x x)] \mid}\right|\right)$ for $|x|<2 \pi$ ? Here, $x$ represents the abscissa and $y$ represents the ordinate.

(A)


(C)

(D)

Solution: We have to identify the graph of

$$
y=\ln \left(\left|e^{[|\sin | x| |]}\right|\right)
$$

for $|x|<2 \pi$.
(1) If $x=0$, then $y=0$. We can reject (D).
(2) Also $y$ is an even function, i.e. $y(-a)=y(a)$. We can reject (B).
(3) We see that $y$ depends on $|\sin | x|\mid($ rather than $\sin |x|)$. Therefore, for $x=\frac{3 \pi}{4}$ and $\frac{5 \pi}{4}$ (for example), $y$ should have the same value. We can reject (A).
Hence, the correct option is (C).
Question Number: 6
Question Type: MCQ
The volume of a sphere of diameter 1 unit is $\qquad$ than the volume of a cube of side 1 unit.
(A) least
(B) less
(C) lesser
(D) low

Solution: The correct option is (B).
Question Number: 7
Question Type: MCQ
The unruly crowd demanded that the accused be $\qquad$ without trial.
(A) hanged
(B) hanging
(C) hankering
(D) hung

Solution: The correct option is (A).
(B)

## Question Number: 8 <br> Question Type: MCQ

Choose the statement(s) where the underlined word is used correctly:
(i) A prone is a dried plum.
(ii) He was lying prone on the floor.
(iii) People who eat a lot of fat are prone to heart disease.
(A) (i) and (iii) only
(B) (iii) only
(C) (i) and (ii) only
(D) (ii) and (iii) only

Solution: A dried plum is called a prune. Hence, statement (i) is incorrect. To lie prone is to lie prostrate. To be prone to a disease is to be liable or likely to suffer from. Hence, options (ii) and (iii) are correct.
Hence, the correct option is (D).
Question Number: 9
Question Type: MCQ
Fact: If it rains, then the field is wet.
Read the following statements:
(i) It rains.
(ii) The field is not wet.
(iii) The field is wet.
(iv) It did not rain.

Which one of the options given below is NOT logically possible, based on the given fact?
(A) If (iii), then (iv).
(B) If (i), then (iii).
(C) If (i), then (ii).
(D) If (ii), then (iv).

Solution: We know that
If $P \Rightarrow q$

$$
p=\text { it rains }
$$

then $\sim q \Rightarrow \sim \mathrm{P} \quad q=$ the field is wet
(A) If $q$, then $p$ (we are not sure)
(B) If $p$, then $q$ (TRUE)
(C) If $p$, then $\sim q$ (NOT POSSIBLE)
(D) If $\sim q$, then $\sim p$ (TRUE)

Hence, the correct option is (C).
Question Number: 10
Question Type: MCQ
A window is made up of a square portion and an equilateral triangle portion above it. The base of the triangular portion coincides with the upper side of the square. If the perimeter of the window is 6 m , the area of the window in $\mathrm{m}^{2}$ is
(A) 1.43
(B) 2.06
(C) 2.68
(D) 2.88

Solution: The window is shown in the figure below.


The boundary of the window is a pentagon, in which all the 5 sides are equal. As the perimeter is 6 m , each of these sides is 1.2 m . The area is $(1.2)^{2}+\frac{\sqrt{3}}{4}(1.2)^{2}$.

$$
=(1.44)\left(1+\frac{1.73}{4}\right)=(0.36)(5.73) \approx 2.06
$$

Hence, the correct option is (B).

## Mechanical Engineering

## Number of Questions: 55

Section marks: 85.0
Q. 11 to Q. 35 carry 1 mark each and Q. 36 to Q. 65 carry 2 marks each.
Question Number: $11 \quad$ Question Type: MCQ
The condition for which the eigenvalues of the matrix are positive is $A=\left[\begin{array}{ll}2 & 1 \\ 1 & k\end{array}\right]$
(A) $k>1 / 2$
(B) $k>-2$
(C) $k>0$
(D) $k<-1 / 2$

Solution: Given matrix is $A=\left[\begin{array}{ll}2 & 1 \\ 1 & k\end{array}\right]$
The eigenvalues of $A$ are positive.
$\Rightarrow$ Sum of the eigenvalues of $A$ is positive and the product of the eigenvalues of $A$ is positive.
$\Rightarrow \operatorname{Trace}(A)>0$ and $\operatorname{Det}(A)>0$

$$
\begin{array}{lc}
\Rightarrow & 2+k>0 \text { and } 2 k-1>0 \\
\Rightarrow & k>-2 \text { and } k>\frac{1}{2} \\
\Rightarrow & k>
\end{array}
$$

Hence, the correct option is (A).
Question Number: 12
Question Type: MCQ
The values of $x$ for which the function is NOT continuous are $f(x)=\frac{x^{2}-3 x-4}{x^{2}+3 x-4}$.
(A) 4 and -1
(B) 4 and 1
(C) -4 and 1
(D) -4 and -1

Solution: Given $f(x)=f(x)=\frac{x^{2}-3 x-4}{x^{2}+3 x-4}$.
As the numerator and the denominator of $f(x)$ are polynomials in $x, f(x)$ is not continuous at those values of $x$, where the denominator is zero.

$$
\begin{array}{rlrl}
\therefore & x^{2}+3 x-4 & =0 \\
\Rightarrow & & (x+4)(x-1) & =0 \\
\Rightarrow & x & =-4, x=1
\end{array}
$$

$\therefore f(x)$ is not continuous at $x=-4$ and $x=1$
Hence, the correct option is (C).

## Question Number: 13

Question Type: MCQ
Laplace transform of $\cos (\omega t)$ is
(A) $\frac{S}{S^{2}+\omega^{2}}$
(B) $\frac{\omega}{S^{2}+\omega^{2}}$
(C) $\frac{S}{S^{2}-\omega^{2}}$
(D) $\frac{\omega}{S^{2}-\omega^{2}}$

Solution: Standard result
Hence, the correct option is (A).
Question Number: 14
Question Type: NAT
A function $f$ of the complex variable $z=x+i y$, is given as $f(x, y)=u(x, y)+i v(x, y)$, where $u(x, y)=2 k x y$ and $v(x, y)$ $=x^{2}-y^{2}$. The value of $k$, for which the function is analytic, is $\qquad$
Solution: Given $f(x, y)=u(x, y)+i v(x, y)$
where $u(x, y)=2 k x y$ and $v(x, y)=x^{2}-y^{2}$

$$
\Rightarrow \frac{\partial u}{\partial x}=2 k y ; \quad \frac{\partial u}{\partial y}=2 k x
$$

and

$$
\frac{\partial v}{\partial x}=2 x ; \quad \frac{\partial v}{\partial y}=-2 y
$$

$f(x, y)$ is analytic $\Rightarrow f(x, y)$ satisfies Cauchy-Riemann equations.

$$
\begin{aligned}
\Rightarrow & \frac{\partial u}{\partial x}=\frac{\partial v}{\partial v} \text { and } \frac{\partial v}{\partial x}=-\frac{\partial u}{\partial y} \\
\Rightarrow & 2 k y=-2 y \text { and } 2 x=-2 k x \\
\Rightarrow & \kappa=-1
\end{aligned}
$$

Hence, the correct answer is -1 .

## Question Number: 15

Question Type: MCQ
Numerical integration using trapezoidal rule gives the best result for a single variable function, which is
(A) linear
(B) parabolic
(C) logarithmic
(D) hyperbolic

Solution: Standard result is linear.
Hence, the correct option is (A).
Question Number: 16
Question Type: MCQ
A point mass having mass $M$ is moving with a velocity $V$ at an angle $\theta$ to the wall as shown in the figure. The mass
undergoes a perfectly elastic collision with the smooth wall and rebounds. The total change (final minus initial) in the momentum of the mass is

(A) $-2 M V \cos \theta \hat{j}$
(B) $2 M V \sin \theta \hat{j}$
(C) $2 M V \cos \theta \hat{j}$
(D) $-2 M V \sin \theta \hat{j}$

Solution:



Velocity of approach $=V \sin \theta$
If ' $u$ ' is the rebound velocity of the point mass making an angle ' $\alpha$ ' with the vertical axis, then,

$$
u \cos \alpha=-V \sin \theta
$$



Total change in momentum, can be calculated using

$$
\begin{aligned}
\Delta P & =M\left(V_{2}-V_{1}\right) \\
& =M(-V \sin \theta-V \sin \theta) \\
& =-2 M V \sin \theta
\end{aligned}
$$

Hence, the correct option is (D).
Question Number: 17
Question Type: MCQ
A shaft with a circular cross-section is subjected to pure twisting moment. The ratio of the maximum shear stress to the largest principal stress is
(A) 2.0
(B) 1.0
(C) 0.5
(D) 0

Solution: As shaft is subjected to pure twisting moment, thus $\sigma_{x}=\sigma_{y}=0$.
Now we have

$$
\begin{aligned}
& \sigma_{1,2}=\frac{\sigma_{x}+\sigma_{y}}{2} \pm \sqrt{\left(\frac{\sigma_{x}-\sigma_{y}}{2}\right)^{2}+\left(\tau_{x y}\right)^{2}} \\
& \sigma_{1,2}= \pm \tau_{x y}
\end{aligned}
$$

$$
\begin{gathered}
\tau_{\max }=\frac{\sigma_{1}-\sigma_{2}}{2}=\frac{\tau_{x y}-\left(-\tau_{x y}\right)}{2}=\tau_{x y} \\
\therefore \frac{\tau_{\max }}{\sigma_{1}}=\frac{\tau_{x y}}{\tau_{x y}}=1
\end{gathered}
$$

Hence, the correct option is (B).

## Question Number: 18

Question Type: MCQ
A thin cylindrical pressure vessel with closed ends is subjected to internal pressure. The ratio of circumferential (hoop) stress to the longitudinal stress is
(A) 0.25
(B) 0.50
(C) 1.0
(D) 2.0

Solution: We know that for a thin cylindrical pressure vessel, ratio of circumferential (hoop) stress to the longitudinal stress is 2 , that is

$$
\frac{\sigma_{\mathrm{h}}}{\sigma_{1}}=2 .
$$

Hence, the correct option is (D).

## Question Number: 19

Question Type: NAT
The forces $F_{1}$ and $F_{2}$ in a brake band and the direction of rotation of the drum are as shown in the figure. The coefficient of friction is 0.25 . The angle of wrap is $3 \pi / 2$ radians. It is given that $R=1 \mathrm{~m}$ and $F_{2}=1 \mathrm{~N}$. The torque (in $\mathrm{N}-\mathrm{m}$ ) exerted on the drum is $\qquad$

## Solution:



Radius $R=1 \mathrm{~m}$
Force $F_{2}=1 \mathrm{~N}$
Angle of wrap $\theta=3 \pi / 2$,
Coefficient of friction $\mu=0.25$
Now using the relation

$$
\begin{gathered}
\frac{F_{1}}{F_{2}}=e^{\mu \theta} \\
\therefore \frac{F_{1}}{F_{2}}=e^{\left(0.5 \times \frac{3 \pi}{2}\right)}=3.246
\end{gathered}
$$

$$
\begin{array}{ll} 
& F_{2}=1 \mathrm{~N} \\
\therefore & F_{1}=3.246 \mathrm{~N}
\end{array}
$$

The torque (in $\mathrm{N}-\mathrm{m}$ ) exerted on the drum will be

$$
\begin{aligned}
T & =\left(F_{1}-F_{2}\right) R \\
& =(3.246-1) \times 1 \\
& =2.246 \mathrm{Nm}
\end{aligned}
$$

Hence, the correct answer is 2.2 to 2.3.

## Question Number: 20

Question Type: MCQ
A single degree of freedom mass-spring-viscous damper system with mass $m$, spring constant $k$, and viscous damping coefficient $q$ is critically damped. The correct relation among $m, k$, and $q$ is
(A) $q=\sqrt{2 k m}$
(B) $q=2 \sqrt{k m}$
(C) $q=\sqrt{\frac{2 k}{m}}$
(D) $q=2 \sqrt{\frac{k}{m}}$

Solution: We know that for critically damped system $q$ relation among $m, k$, and $q$ is

$$
q_{c}=2 \sqrt{\mathrm{~km}}
$$

Hence, the correct option is (C).
Question Number: 21
Question Type: MCQ
A machine element $X Y$, fixed at end $X$, is subjected to an axial load $P$, transverse load $F$, and a twisting moment $T$ at its free and $Y$. The most critical point from the strength point of view is

(A) a point on the circumference at location $Y$
(B) a point at the center at location $Y$
(C) a point on the circumference at location $X$
(D) a point at the center at location $X$

Solution: Outer fibers of the element XY will be at severe stress, location $X$ will be stressed more than location $Y$ as movement of $X$ is restricted.
Hence, the correct option is (C).
Question Number: 22
Question Type: MCQ
For the brake shown in the figure, which one of the following is TRUE?
(A) Self energizing for clockwise rotation of the drum
(B) Self energizing for anti-clockwise rotation of the drum

(C) Self energizing for rotation in either direction of the drum
(D) Not of the self energizing type

Solution: Brake is said to be self-energizing when the moment of frictional force $(\mu \mathrm{N})$ is in the same direction as that of the applied force $(F)$.
Therefore, in the given case, the brake will be self energizing when the rotation of the drum is clockwise.
Hence, the correct option is (C).
Question Number: 23
Question Type: MCQ
The volumetric flow rate (per unit depth) between two streamlines having stream functions $\Psi_{1}$ and $\Psi_{2}$ is
(A) $\left|\Psi_{1}+\Psi_{2}\right|$
(B) $\Psi_{1} \Psi_{2}$
(C) $\Psi_{1} / \Psi_{2}$
(D) $\left|\Psi_{1}-\Psi_{2}\right|$

Solution: We know that the difference between two stream functions give volumetric flow rate, i.e.,

$$
\dot{q}=\left|\Psi_{1}-\Psi_{2}\right| .
$$

Hence, the correct option is (C).
Question Number: 24
Question Type: MCQ
Assuming constant temperature condition and air to be an ideal gas, the variation in atmospheric pressure with height calculated from fluid static is
(A) linear
(B) exponential
(C) quadratic
(D) cubic

Solution: For an ideal gas at isothermal condition the variation in atmospheric pressure with height calculated from fluid static is exponential.
Hence, the correct option is (B).
Question Number: 25
Question Type: MCQ
A hollow cylinder has length $L$, inner radius $r_{1}$, outer radius $r_{2}$, and thermal conductivity $k$. The thermal resistance of the cylinder for radial conduction is
(A) $\frac{\ln \left(r_{2} / r_{1}\right)}{2 \pi k L}$
(B) $\frac{\ln \left(r_{1} / r_{2}\right)}{2 \pi k L}$
(C) $\frac{2 \pi k L}{\ln \left(r_{2} / r_{1}\right)}$
(D) $\frac{2 \pi k L}{\ln \left(r_{1} / r_{2}\right)}$

Solution: The thermal resistance of the cylinder for radial conduction is

$$
R_{\mathrm{th}}=\frac{\ln \left(r_{2} / r_{1}\right)}{2 \pi k L}
$$

Hence, the correct option is (A).
Question Number: 26
Question Type: MCQ
Consider the radiation heat exchange inside an annulus between two very long concentric cylinders. The radius of the outer cylinder is $R_{\mathrm{o}}$ and that of the inner cylinder is $R_{\mathrm{i}}$. The radiation view factor of the outer cylinder onto itself is
(A) $1-\sqrt{\frac{R_{\mathrm{i}}}{R_{\mathrm{o}}}}$
(B) $\sqrt{1-\frac{R_{\mathrm{i}}}{R_{\mathrm{o}}}}$
(C) $1-\left(\frac{R_{\mathrm{i}}}{R_{\mathrm{o}}}\right)^{1 / 3}$
(D) $1-\frac{R_{\mathrm{i}}}{R_{\mathrm{o}}}$

Solution: Radius of the outer cylinder $=R_{\text {o }}$ Radius of the inner cylinder $=R_{\mathrm{i}}$.
We know that

$$
\begin{aligned}
F_{11} & =0 \\
F_{11}+F_{12} & =1 \\
F_{12} & =1 \\
\therefore \quad F_{12} A_{1} & =F_{21} A_{2} \\
\text { Reciprocity theorem } & =\frac{A_{1}}{A_{2}}=\frac{2 \pi R_{\mathrm{i}} L}{2 \pi R_{\mathrm{o}} L}=\frac{R_{\mathrm{i}}}{R_{\mathrm{o}}} \\
F_{22} & =1-F_{21}=1-\frac{R_{\mathrm{i}}}{R_{\mathrm{o}}}
\end{aligned}
$$

Hence, the correct option is (D).
Question Number: 27
Question Type: MCQ
The internal energy of an ideal gas is a function of
(A) temperature and pressure
(B) volume and pressure
(C) entropy and pressure
(D) temperature only

Solution: The internal energy of an ideal gas is a function of temperature. Thus,

$$
u=f(T) .
$$

Hence, the correct option is (D).
Question Number: 28
Question Type: NAT
The heat removal rate from a refrigerated space and the power input to the compressor are 7.2 kW and 1.8 kW , respectively. The coefficient of performance (COP) of the refrigerator is $\qquad$ -.
Solution: Heat removal rate, $Q_{2}=7.2 \mathrm{~kW}$


Power input to the compressor $W=1.8 \mathrm{~kW}$
The coefficient of performance (COP) of the refrigerator can be calculated using the relation

$$
\begin{aligned}
(C O P)_{\mathrm{R}} & =\frac{Q_{2}}{Q_{2}-Q_{1}}=\frac{Q_{2}}{W} \\
& =\frac{7.2}{1.8}=4
\end{aligned}
$$

Hence, the correct answer is 4.0 .
Question Number: 29

## Question Type: NAT

Consider a simple gas turbine (Brayton) cycle and a gas turbine cycle with perfect regeneration. In both the cycles, the pressure ratio is 6 and the ratio of the specific heats of the working medium is 1.4. The ratio of minimum to maximum temperatures is 0.3 (with temperatures expressed in K ) in the regenerative cycle. The ratio of the thermal efficiency of the simple cycle to that of the regenerative cycle is $\qquad$
Solution: Pressure ratio $r=6$,
Without regeneration
Ratio of the specific heat of the working medium $\gamma=1.4$
Now using the relation

$$
\begin{aligned}
\eta_{\text {Brayton }} & =1-\frac{1}{(r)^{\frac{\gamma-1}{\gamma}}} \\
& =1-\frac{1}{(6)^{\frac{1.4-1}{1.4}}}=0.4
\end{aligned}
$$

With regeneration,

$$
\text { Ratio } \frac{T_{1}}{T_{3}}=0.3
$$

Now using the relation

$$
\begin{aligned}
\eta_{\text {Brayton }} & =1-\frac{T_{1}}{T_{3}} r^{\frac{\gamma-1}{\gamma}} \\
& =1-(0.3) 6^{\frac{1.4-1}{1.4}}=0.5
\end{aligned}
$$

Thus, the required ratio $=\frac{0.4}{0.5}=0.8$.
Hence, the correct answer is 0.8 .

## Question Number: 30

Question Type: NAT
In a single-channel queuing model, the customer arrival rate is 12 per hour and the serving rate is 24 per hour. The expected time that a customer is in queue is $\qquad$ minutes.

Solution: Customer arrival rate $\lambda=12 /$ hour
Serving rate $\mu=24 /$ hour
Now using the relation given below, expected time that a customer is in queue can be calculated as

$$
\begin{aligned}
W_{\mathrm{q}} & =\frac{\lambda}{\mu(\mu-\lambda)} \\
& =\frac{12}{24(24-12)}=\frac{1}{24} \text { hour } \\
& =\frac{1}{24} \times 60=2.5 \mathrm{~min}
\end{aligned}
$$

Hence, the correct answer is 2.5 .
Question Number: 31
Question Type: MCQ
In the phase diagram shown in the figure, four samples of the same composition are heated to temperatures marked by $a, b, c$, and $d$.


At which temperature will a sample get solutionized the fastest?
(A) a
(B) b
(C) c
(D) d

Solution: Maximum solubility occurs at point c.
Hence, the correct option is (C).
Question Number: 32
Question Type: MCQ
The welding process which uses a blanket of fusible granular flux is
(A) tungsten inert gas welding
(B) submerged arc welding
(C) electroslag welding
(D) thermit welding

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Solution: Submerged arc welding (SAW) uses blanket of fusible granular flux.
Hence, the correct option is (B).
Question Number: 33
Question Type: MCQ
The value of true strain produced in compressing a cylinder to half its original length is
(A) 0.69
(B) -0.69
(C) 0.5
(D) -0.5

Solution: If $L_{\mathrm{o}}$ is the original length, it is given that

$$
L=\frac{1}{2} L_{\mathrm{o}}
$$

Value of true strain produced in compressing will be

$$
\begin{aligned}
\epsilon_{\mathrm{T}} & =\ln \frac{L}{L_{\mathrm{o}}} \text { (for tension) } \\
\therefore \quad \epsilon_{\mathrm{T}} & =\ln \left(\frac{1}{2}\right)=-\ln 2 \\
& =-0.69
\end{aligned}
$$

Hence, the correct option is (B).
Question Number: 34
Question Type: NAT
The following data is applicable for a turning operation. The length of job is 900 mm , diameter of job is 200 mm , feed rate is $0.25 \mathrm{~mm} / \mathrm{rev}$, and optimum cutting speed is 300 $\mathrm{m} / \mathrm{min}$. The machining time (in min) is $\qquad$ -.

Solution: Length of job $L=900 \mathrm{~mm}$
Diameter of job $D=200 \mathrm{~mm}$
Feed rate $f=0.25 \mathrm{~mm}^{2} / \mathrm{rev}$
Optimum cutting speed $V=300 \mathrm{~m} / \mathrm{min}$
The machining time can be calculated using

$$
\begin{aligned}
t & =\frac{\pi D L}{1000 \mathrm{fV}} \\
& =\frac{\pi \times 200 \times 900}{1000 \times 0.25 \times 300}=7.536 \mathrm{~min}
\end{aligned}
$$

Hence, the correct answer is 7.5 to 7.6.
Question Number: 35
Question Type: MCQ
In an ultrasonic machining (USM) process, the material removal rate (MRR) is plotted as a function of the feed force of the USM tool. With increasing feed force, the MRR exhibits the following behavior:
(A) increases linearly
(B) decreases linearly
(C) does not change
(D) first increases and then decreases

Solution: With increasing feed force, the MRR first increases and then decreases as shown in the figure.


Hence, the correct option is (D).
Question Number: 36
Question Type: NAT A scalar potential $\phi$ has the following gradient: $\nabla \phi=y z \hat{i}$ $+x z \hat{j}+x y \hat{k}$. Consider the integral $\int_{C} \nabla \phi \cdot d \vec{r}$ on the curve $\vec{r}=x \hat{i}+y \hat{j}+z \hat{k}$.
The curve $C$ is parameterized as follows:

$$
\left\{\begin{array}{l}
x=t \\
y=t^{2} \\
z=3 t^{2}
\end{array} \text { and } \quad 1 \leq t \leq 3\right.
$$

The value of the integral is $\qquad$ -
Solution: Gradient of scalar potential is

$$
\nabla \varphi=y z \bar{i}+x z \bar{j}+x y \bar{k}
$$

And the given curve is

$$
\begin{gathered}
\bar{r}=x \bar{i}+y \bar{j}+z \bar{k} \\
\Rightarrow d \bar{r}=d x \bar{i}+d y \bar{j}+d z \bar{k}
\end{gathered}
$$

The parametric form of the given curve $C$ is
$x=t, y=t^{2}$, and $z=3 t^{2} ; 1 \leq t \leq 3$
$d x=d t, d y=2 t d t$, and $d z=6 t d t$

$$
\begin{aligned}
& \therefore \int_{C} \\
& \nabla \varphi \cdot d \bar{r}=\oint_{C}(y z \bar{i}+x z \bar{j}+x y \bar{k}) \cdot(d x \bar{i}+d y \bar{j}+d z \bar{k}) \\
&=\oint_{C}[y z d x+x z d y+x y d z] \\
&=\int_{t=1}^{3}\left[\left(t^{2} \times 3 t^{2}\right) d t+\left(t \times 3 t^{2}\right) 2 t d t+\left(t \times t^{2}\right) 6 t d t\right] \\
&\left.=\int_{t=1}^{3} 15 t^{4} d t=15 \times \frac{t^{5}}{5}\right]_{t=1}^{3}=726
\end{aligned}
$$

Hence, the correct answer is 726 .
Question Number: 37

## Question Type: MCQ

The value of $\oint_{r} \frac{3 z-5}{(z-1)(z-2)} d z$ along a closed path $\Gamma$ is equal to $(4 \pi i)$, where $z=x+i y$ and $i=\sqrt{-1}$. The correct path $\Gamma$ is

(B)

(C)

(D)

Solution: Consider the integral

$$
I=\oint_{\Gamma} \frac{3 z-5}{(z-1)(z-2)} d z
$$

If

$$
g(z)=\frac{3 z-5}{(z-1)(z-2)}
$$

$z=1$ and $z=2$ are the singularities of $g(z)$.
For option (A):
From the figure, we conclude that here both the singularities are inside the closed path $\Gamma$.


$$
\begin{aligned}
\therefore I & =\oint_{\Gamma} \frac{3 z-5}{(z-1)(z-2)} d z \\
& =\oint_{\Gamma}\left[\frac{2}{z-1}+\frac{1}{z-2}\right] d z \oint_{\Gamma} \frac{2}{(z-1)} d z+\oint_{\Gamma} \frac{1}{z-2} d z \\
& =2 \pi i \times 2+2 \pi i \times 1
\end{aligned}
$$

(By Cauchy's integral formula)

$$
\therefore I=5 \pi i \neq 4 \pi i
$$

For Option (B):
From the figure given below, we conclude that here the singularity $z=1$ only lies inside $\Gamma$.

$$
\begin{aligned}
& \therefore \oint_{\Gamma} \frac{3 z-5}{(z-1)(z-2)} d z=\oint_{\Gamma} \frac{(3 z-5) /(z-2)}{(z-1)} d z \\
& =2 \pi i \cdot f\left(z_{0}\right), \text { where } f(z)=\frac{3 z-5}{z-1}
\end{aligned}
$$

and $z_{0}=1$
(By Cauchy's integral formula)

$$
\therefore I=4 \pi i
$$

Hence, the correct option is (B).
Question Number: 38
Question Type: MCQ
The probability that a screw manufactured by a company is defective is 0.1 . The company sells screws in packets
containing 5 screws and gives a guarantee of replacement if one or more screws in the packet are found to be defective. The probability that a packet would have to be replaced is
$\qquad$ .

Solution: If random variable $X$ is defined by 'The number of defective screws in a packet of $5^{\prime}$.
Then clearly $X$ follows binomial distribution with 'Drawing a defective screw' as success.
Probability that a screw manufactured by a company is defective $p=0.1$

$$
q=1-p=0.9
$$

Probability that a packet of 5 screws would have to be replaced $=$ Probability that the packet contains one or more defective screws.

$$
\begin{aligned}
& =P(\mathrm{X}<1)=1-P(X<1) \\
& =1-\mathrm{P}(\mathrm{X}=0)=1-P(0 ; 5,0.1) \\
& =1-5 \mathrm{C}_{\mathrm{o}}(0.1)^{0}(0.9)^{5}=0.4095
\end{aligned}
$$

Hence, the correct option is 0.4095 .

## Question Number: 39

Question Type: NAT
The error in numerically computing the integral
$\int_{0}^{\pi}(\sin x+\cos x) d x$ using the trapezoidal rule with three intervals of equal length between 0 and $\pi$ is $\qquad$ -
Solution: Let $I=\int_{0}^{\pi}(\sin x+\cos x) d x$
Exact value of $I$ :
(Value of $I$ by the analytical method)

$$
\begin{align*}
I & \left.=\int_{0}^{\pi}(\sin x+\cos x) d x=-\cos x+\sin x\right]_{0}^{\pi} \\
& =(-\cos \pi)-(-\cos 0+\sin 0) \\
\therefore & I=\int_{0}^{\pi}(\sin x+\cos x) d x=2 \tag{1}
\end{align*}
$$

Value of $I$ by the trapezoidal rule
Here $a=0$ and $b=\pi$
The number of sub-intervals $=n=3$

$$
\therefore h=\frac{b-a}{n}=\frac{\pi}{3}
$$

Let $y=f(x)=\sin x+\cos x$

| $x$ | 0 | $\frac{\pi}{3}$ | $\frac{2 \pi}{3}$ | $\pi$ |
| :---: | :---: | :---: | :---: | :---: |
| $y=f(x)$ | 1 | 1.3660 | 0.3660 | -1 |

By trapezoidal rule, we have

$$
\begin{gather*}
\int_{\sigma}^{b} y d x=\frac{h}{2}\left[\left(y_{0}+y_{3}\right)+2\left(y_{1}+y_{2}\right)\right] \\
\therefore I=\int_{0}^{\pi}(\sin x+\cos x) d x \\
=\frac{(\pi / 3)}{2}[(1+(-1))+2(1.3660+0.3660)]=1.8137 \\
\therefore I=\int_{0}^{\pi}(\sin x+\cos x) d x=1.8137 \tag{2}
\end{gather*}
$$

$\therefore$ From Eqs. (1) and (2),

$$
\begin{aligned}
\text { Error } & =2-1.8137 \\
& =0.1863
\end{aligned}
$$

Hence, the correct answer is 0.1863 .
Question Number: 40
Question Type: NAT
A mass of 2000 kg is currently being lowered at a velocity of $2 \mathrm{~m} / \mathrm{s}$ from the drum as shown in the figure. The mass moment of inertia of the drum is $150 \mathrm{~kg}-\mathrm{m}^{2}$. On applying the brake, the mass is brought to rest in a distance of 0.5 m . The energy absorbed by the brake (in kJ ) is $\qquad$ —.


Solution: Mass $m=20,000 \mathrm{~kg}$
Velocity $v=2 \mathrm{~m} / \mathrm{s}$
Mass moment of inertia of drum $I=150 \mathrm{~kg}-\mathrm{m}^{2}$
Distance $h=0.5 \mathrm{~m}$
Angular speed

$$
\omega=\frac{v}{r}=\frac{2}{1}=2 \mathrm{rad} / \mathrm{s},
$$

The energy absorbed by the brake (in kJ ) will be

$$
\begin{aligned}
E & =m g h+\frac{1}{2} m v^{2}+\frac{1}{2} I \omega^{2} \\
& =14.1 \mathrm{~kJ}
\end{aligned}
$$

Hence, the correct answer is 14.1 to 14.3 .

## Question Number: 41

Question Type: MCQ
A system of particles in motion has mass center $G$ as shown in the figure. The particle $i$ has mass $m_{\mathrm{i}}$ and its position with respect to a fixed point $O$ is given by the position vector $r_{\mathrm{i}}$. The position of the particle with respect to $G$ is given by the vector $\rho_{\mathrm{i}}$. The time rate of change of the angular momentum of the system of particles about $G$ is
(The quantity $\ddot{\rho}_{i}$ indicates second derivative of $\rho_{\mathrm{i}}$ with respect to time and likewise for $r_{\mathrm{i}}$ ).

(A) $\Sigma_{\mathrm{i}} r_{\mathrm{i}} \times m_{\mathrm{i}} \ddot{\rho}_{i}$
(B) $\Sigma_{i} \rho_{i} \times m_{i} \ddot{r}_{i}$
(C) $\Sigma_{i} r_{i} \times m_{i} \ddot{r}_{i}$
(D) $\Sigma_{i} \rho_{i} \times m_{i} \ddot{\rho}_{i}$

Solution: Rate of change of angular momentum of system about

$$
\begin{aligned}
G & =\Sigma_{i} \frac{d}{d t}(m v r)=\Sigma_{i} \frac{d}{d t}\left(m_{\mathrm{i}} \dot{r}_{i} \rho_{\mathrm{i}}\right) \\
& =\Sigma_{i} m_{i} \ddot{r}_{i} \rho_{i}
\end{aligned}
$$

Hence, the correct option is (B).
Question Number: 42
Question Type: MCQ
A rigid horizontal rod of length $2 L$ is fixed to a circular cylinder of radius $R$ as shown in the figure. Vertical forces of magnitude $P$ are applied at the two ends as shown in the figure. The shear modulus for the cylinder is $G$ and the Young's modulus is $E$.


The vertical deflection at point $A$ is
(A) $P L^{3} /\left(\pi R^{4} G\right)$
(B) $P L^{3} /\left(\pi R^{4} E\right)$
(C) $2 P L^{3} /\left(\pi R^{4} E\right)$
(D) $4 P L^{3} /\left(\pi R^{4} G\right)$

Solution: We know that

$$
\begin{aligned}
\text { Torque } \begin{aligned}
(\tau) & =P \times 2 L \\
\frac{T}{J} & =\frac{G \theta}{L} \\
\theta & =\frac{T L}{J G} \\
\text { Deflection } & =L \theta \\
\text { Deflection } & =\frac{T L^{2}}{J G} \\
& =\frac{(P \times 2 L) L^{2}}{\frac{\pi}{32}(2 R)^{4} \times G}=\frac{4 P L^{3}}{\pi R^{4} G}
\end{aligned}
\end{aligned}
$$

Hence, the correct option is (D).

## Question Number: 43

Question Type: MCQ
A simply supported beam of length $2 L$ is subjected to a moment $M$ at the mid-point $x=0$ as shown in the figure. The deflection in the domain $0 \leq x \leq L$ is given by

$$
W=\frac{-M x}{12 E I L}(L-x)(x+c)
$$

where $E$ is the Young's modulus, $I$ is the area momentum of inertia, and $c$ is a constant (to be determined).


The slope at the center $x=0$ is
(A) $M L /(2 E I)$
(B) $M L /(3 E I)$
(C) $M L /(6 E I)$
(D) $M L /(12 E I)$

Solution: $R_{A}=\frac{M}{2 L}, R_{B}=-\frac{M}{2 L}$
Bending moment at section $X-X$ at $x$, from center $C$ is

$$
\begin{gathered}
M_{\mathrm{x}}=R_{\mathrm{B}}(L-x)=\frac{-M}{2 L}(L-x) \\
\frac{d^{2} y}{d x^{2}}=\frac{M_{x}}{E I}=\frac{-M}{2 E I L}(L-x) \\
\frac{d y}{d x}=\frac{-M}{2 E I L} \frac{(L-x)^{2}}{2}(-1)+C_{1}
\end{gathered}
$$

$$
\begin{aligned}
y & =\frac{M}{4 E I L} \frac{(L-x)^{3}}{(-3)}+C_{1} x+C_{2} \\
& =\frac{-M(L-x)^{3}}{12 E I L}+C_{1} x+C_{2}
\end{aligned}
$$

Boundary conditions:

1. $x=0, y=0$
2. $x=L, y=0$

$$
y_{(x=0)}=\frac{-M L^{3}}{12 E I L}+C_{2}=0
$$

$\therefore \quad C_{2}=\frac{M L^{2}}{12 E I}$
and
$\therefore \quad C_{1}=\frac{-C_{2}}{L}=\frac{-M L}{12 E I}$
Thus,

$$
\begin{aligned}
& y=\frac{-M}{12 E I L}(L-x)^{3}-\frac{M L(x)}{12 E I}+\frac{M L^{2}}{12 E I} \\
& =\frac{-M}{12 E I L}\left(L^{3}-3 L^{2} x+3 L x^{2}-x^{3}+L^{2} x-L^{3}\right) \\
& =\frac{-M}{12 E I L}\left(-2 L^{2} x+3 L x^{2}-x^{3}\right) \\
& =\frac{-M x}{12 E I L}\left(-2 L^{2}+3 L x-x^{2}\right) \\
& =\frac{-M x}{12 E I L}\left\{\left(-2 L^{2}+2 L x+L x-x^{2}\right)\right\} \\
& =\frac{-M x}{12 E I L}\{-2 L(L-x)+x(L-x)\} \\
& =\frac{-M x}{12 E I L}(L-x)(x-2 L)
\end{aligned}
$$

Comparing the above equation with the expression for deflection,

$$
w=\frac{-M x}{12 E I L}(L-x)(x+c)
$$

We get $C=-2 L$

$$
\begin{aligned}
& \text { Slope } \theta=\frac{d w}{d x} \\
= & \frac{d}{d x}\left\{\frac{-M x}{12 E I L}\left(L x+L c-x^{2}-x c\right)\right\} \\
= & \frac{-M x}{12 E I L}(L-2 x-c)+\left(L x+L c-x^{2}-x c\right)\left(\frac{-M}{12 E I L}\right)
\end{aligned}
$$

$$
\begin{aligned}
\theta_{(x=0)} & =0+(L c)\left(\frac{-M}{12 E I L}\right) \\
& =\frac{-M c}{12 E I}=\frac{M L}{6 E I}
\end{aligned}
$$

Hence, the correct option is (C).
Question Number: 44
Question Type: NAT
In the figure, the load $P=1 N$, length $L=1 \mathrm{~m}$, Young's modulus $E=70 \mathrm{GPa}$, and the cross-section of the links is a square with dimension $10 \mathrm{~mm} \times 10 \mathrm{~mm}$. All joints are pin joints.


The stress (in Pa) in the link $A B$ is $\qquad$ -.
(Indicate compressive stress by a negative sign and tensile stress by a positive sign.)

Solution: Consider the free-body diagram, given below:


$$
\begin{aligned}
T_{1} \cos 45 & =P \cos 45 \\
T_{1} & =P=1 \mathrm{~N} \\
T_{1} \sin 45 & =P \sin 45-T_{2} \\
T_{2} & =0
\end{aligned}
$$

Since tension in link $A B\left(T_{2}\right)$ is zero, hence stress will also be zero.
Hence, the correct answer is 0 .
Question Number: 45
Question Type: NAT
A circular metallic rod of length 250 mm is placed between two rigid immovable walls as shown in the figure. The rod is in perfect contact with the wall on the left side and there
is a gap of 0.2 mm between the rod and the wall on the right side. If the temperature of the rod is increased by $200^{\circ} \mathrm{C}$, the axial stress developed in the rod is $\qquad$ MPa.
Young's modulus of the material of the rod is 200 GPa and the coefficient of thermal expansion is $10^{-5}$ per ${ }^{\circ} \mathrm{C}$.


Solution: Length of metallic $\operatorname{rod} L=250 \mathrm{~mm}$,
Change in temperature $\Delta T=200^{\circ} \mathrm{C}$,
Coefficient of thermal expansion $\alpha=10^{-5} /{ }^{\circ} \mathrm{C}$
Young's modulus of the material of the rod $=200 \mathrm{GPa}$
Now we know that

$$
\begin{aligned}
\delta l & =\alpha \Delta T L=0.5 \mathrm{~mm} \\
(\delta l)_{\mathrm{net}} & =0.5-0.2=0.3 \\
\Sigma_{\mathrm{T}} & =\frac{(\delta l)_{\mathrm{net}}}{L}=\frac{0.3}{250} \\
\sigma_{\mathrm{T}} & =E \epsilon_{\mathrm{T}} \\
& =200 \times 10^{3} \times \frac{0.3}{250} \\
& =240 \mathrm{MPa}
\end{aligned}
$$

Hence, the correct answer is 240 .

## Question Number: 46

Question Type: NAT
The $\operatorname{rod} A B$, of length 1 m , shown in the figure is connected to two sliders at each end through pins. The sliders can slide along $Q P$ and $Q R$. If the velocity $V_{\mathrm{A}}$ of the slider at $A$ is $2 \mathrm{~m} / \mathrm{s}$, the velocity of the midpoint of the rod at this instant is $\qquad$ $\mathrm{m} / \mathrm{s}$.


Solution: Hence, the correct answer is 0.95 to 1.05 .
Question Number: 47
Question Type: NAT
The system shown in the figure consists of block $A$ of mass 5 kg connected to a spring through a mass less rope passing
over pulley $B$ of radius $r$ and mass 20 kg . The spring constant $k$ is $1500 \mathrm{~N} / \mathrm{m}$. If there is no slipping of the rope over the pulley, the natural frequency of the system is $\qquad$ rad/s.


Solution: Using energy method

$$
\frac{d E}{d t}=0
$$

$$
\begin{aligned}
& E=\frac{1}{2} I_{0} \omega^{2}+\frac{1}{2} m v^{2}+\frac{1}{2} k x^{2} \\
& =\frac{1}{2}\left(\frac{M r^{2}}{2}\right)-\dot{\theta}^{2}+\frac{1}{2} m(r \dot{\theta})^{2}+\frac{1}{2} k(r \theta)^{2} \\
& \frac{d E}{d t}=\frac{1}{2}\left(\frac{M r^{2}}{2} \times 2 \ddot{\theta} \ddot{\theta}+m r^{2}(2 \dot{\theta} \ddot{\theta})+k r^{2}(2 \theta \dot{\theta})\right)=0
\end{aligned}
$$

$$
\frac{M r^{2}}{2} \ddot{\theta}+m r^{2} \ddot{\theta}+k r^{2} \theta=0
$$

$$
\left(m+\frac{M}{2}\right) \ddot{\theta}+k \theta=0
$$

$$
\omega_{n}=\sqrt{\frac{k}{m+M / 2}}=\sqrt{\frac{1500}{5+10}}=10 \mathrm{rad} / \mathrm{s}
$$

Hence, the correct answer is 10 .
Question Number: 48
Question Type: MCQ
In a structural member under fatigue loading, the minimum and maximum stresses developed at the critical point are 50 MPa and 150 MPa , respectively. The endurance, yield, and the ultimate strengths of the material are $200 \mathrm{MPa}, 300$ MPa , and 400 MPa , respectively. The factor of safety using modified Goodman criterion is
(A) $\frac{3}{2}$
(B) $\frac{8}{5}$
(C) $\frac{12}{7}$
(D) 2

Solution: Minimum stresses $\sigma_{\text {min }}=50 \mathrm{MPa}$


Maximum stresses $\sigma_{\text {max }}=150 \mathrm{MPa}$

$$
\begin{aligned}
& \sigma_{\mathrm{a}}=\frac{150-50}{2}=50 \mathrm{MPa} \\
& \sigma_{\mathrm{m}}=\frac{150+50}{2}=100 \mathrm{MPa}
\end{aligned}
$$

Endurance strength $S_{e}=200 \mathrm{MPa}$,
Yeild strength $S_{\mathrm{yt}}=300 \mathrm{MPa}$
Ultimate strength $S_{\mathrm{ut}}=400 \mathrm{MPa}$

$$
\tan \theta=\frac{\sigma_{\mathrm{a}}}{\sigma_{\mathrm{m}}}=0.5
$$

Equation of line $A B$,

$$
\begin{align*}
\frac{S_{\mathrm{a}}}{S_{\mathrm{e}}}+\frac{S_{\mathrm{m}}}{S_{\mathrm{ut}}} & =1 \\
\frac{S_{\mathrm{a}}}{200}+\frac{S_{\mathrm{m}}}{400} & =1 \\
2 S_{\mathrm{a}}+S_{\mathrm{m}} & =400 \tag{1}
\end{align*}
$$

Equation of line OX,

$$
\begin{equation*}
\frac{S_{\mathrm{a}}}{S_{\mathrm{m}}}=\tan \theta=0.5 \tag{2}
\end{equation*}
$$

Solving Eqs. (1) and (2), we get

$$
S_{\mathrm{a}}=100 \mathrm{MPa}, S_{\mathrm{m}}=200 \mathrm{MPa}
$$

Now,

$$
\begin{aligned}
& \sigma_{\mathrm{a}}=\frac{S_{\mathrm{a}}}{(f o s)} \\
& f o s=\frac{S_{\mathrm{a}}}{\sigma_{\mathrm{a}}}=\frac{100}{50}=2
\end{aligned}
$$

Hence, the correct option is (D).
Question Number: 49
Question Type: NAT
The large vessel shown in the figure contains oil and water. A body is submerged at the interface of oil and water such that $45 \%$ of its volume is in oil while the rest is in water. The density of the body is $\qquad$ $\mathrm{kg} / \mathrm{m}^{3}$.

The specific gravity of oil is 0.7 and the density of water is $1000 \mathrm{~kg} / \mathrm{m}^{3}$.

## Acceleration due to gravity $g=10 \mathrm{~m} / \mathrm{s}^{2}$



Specific gravity of oil $=0.7$
Density of water $=1000 \mathrm{~kg} / \mathrm{m}^{3}$
Solution: Let the density of the body be $\rho_{\mathrm{b}}$.
Now using the relation

$$
\begin{aligned}
(\rho V)_{\text {body }} & =(\rho V)_{\text {water }}+(\rho V)_{\text {oil }} \\
\rho_{\mathrm{b}} V & =(1000)(0.55) V+700 \times(0.45) V \\
\rho_{\mathrm{b}} & =550+315 \\
& =865 \mathrm{~kg} / \mathrm{m}^{3}
\end{aligned}
$$

Hence, the correct answer is 865 .
Question Number: 50
Question Type: NAT
Consider fluid flow between two infinite horizontal plates which are parallel (the gap between them being 50 mm ). The top plate is sliding parallel to the stationary bottom plate at a speed of $3 \mathrm{~m} / \mathrm{s}$. The flow between the plates is solely due to the motion of the top plate. The force per unit area (magnitude) required to maintain the bottom plate stationary is $\qquad$ $\mathrm{N} / \mathrm{m}^{2}$.
Viscosity of the fluid $\mu=0.44 \mathrm{~kg} / \mathrm{m}-\mathrm{s}$ and density $\rho=88$ $\mathrm{kg} / \mathrm{m}^{3}$.

## Solution:



Viscosity of the fluid $\mu=0.44 \mathrm{~kg} / \mathrm{m} \cdot \mathrm{s}$
Density of fluid $\rho=88 \mathrm{~kg} / \mathrm{m}^{3}$
The force per unit area (magnitude) required to maintain the bottom plate stationary will be

$$
\begin{aligned}
\frac{F}{A} & =\tau=\frac{\mu V}{y}=\frac{0.44 \times 3}{0.05} \\
& =26.4 \mathrm{~N} / \mathrm{m}^{2}
\end{aligned}
$$

Hence, the correct answer is 26.4.

## Question Number: 51

Question Type: MCQ
Consider a frictionless, mass less, and leak-proof plug blocking a rectangular hole of dimensions $2 R \times L$ at the bottom of an open tank as shown in the figure. The head of the plug has the shape of a semi-cylinder of radius $R$. The tank is filled with a liquid of density $\rho$ up to the tip of the plug. The gravitational acceleration is $g$. Neglect the effect of the atmospheric pressure.


The force $F$ required to hold the plug in its position is
(A) $2 \rho R^{2} g L\left(1-\frac{\pi}{4}\right)$
(B) $2 \rho R^{2} g L\left(1+\frac{\pi}{4}\right)$
(C) $\pi R^{2} \rho g L$
(D) $\frac{\pi}{2} \rho R^{2} g L$

Solution: Consider the figure given below:


Neglecting mass of plug,
Net Buoyancy force $=S_{\mathrm{g}}\left(V_{\text {net }}\right)$
Now using the relation

$$
\begin{aligned}
V_{\text {net }} & =(2 R \times L) R-\frac{\pi R^{2} L}{2} \\
& =2 R^{2} L\left(1-\frac{\pi}{4}\right)
\end{aligned}
$$

The force $F$ required to hold the plug in its position is

$$
\begin{aligned}
& \therefore \quad F_{\mathrm{B}}=2 R^{2} L \rho_{\mathrm{g}}\left(1-\frac{\pi}{4}\right) \\
& \qquad F_{B}=F=2 \rho R^{2} g L\left(1-\frac{\pi}{4}\right) \\
& \text { Hence, the correct option is (A). }
\end{aligned}
$$

## Question Number: 52

Question Type: NAT
Consider a parallel-flow heat exchanger with area $A_{\mathrm{p}}$ and a counter-flow heat exchanger with area $A_{c}$. In both the heat exchangers, the hot stream flowing at $1 \mathrm{~kg} / \mathrm{s}$ cools from $80^{\circ} \mathrm{C}$ to $50^{\circ} \mathrm{C}$. For the cold stream in both the heat exchangers, the flow rate and the inlet temperature are $2 \mathrm{~kg} / \mathrm{s}$ and $10^{\circ} \mathrm{C}$, respectively. The hot and cold streams in both the heat exchanges are of the same fluid. Also, both the heat exchangers have the same overall heat transfer coefficient. The ratio $A_{\mathrm{c}} / A_{\mathrm{p}}$ is $\qquad$ _.

Solution: Flow rate of hot stream $=1 \mathrm{~kg} / \mathrm{s}$
Change in temperature of hot stream $T_{\mathrm{h} 1}-T_{\mathrm{h} 2}=80-50=$ $30^{\circ} \mathrm{C}$

Flow rate of cold stream $=2 \mathrm{~kg} / \mathrm{s}$
Change in temperature of cold stream $T_{\mathrm{h} 1}-T_{\mathrm{c} 2}=80-50$ $=30^{\circ} \mathrm{C}$
As specific heat of cold and hot stream is same, therefore $\left(\epsilon_{\mathrm{P}}=\epsilon_{\mathrm{C}}=\epsilon_{\mathrm{P}, \mathrm{C}}\right)$

$$
\begin{align*}
& R=\frac{C_{\min }}{C_{\max }}=\frac{1}{2}=0.5 \\
& \epsilon_{\mathrm{p}}=\frac{1-\exp (-N T U(1+R))}{1+R} \tag{1}
\end{align*}
$$

Also, for cold

$$
\begin{align*}
& \epsilon_{\mathrm{P} \cdot \mathrm{C}}=\frac{T_{\mathrm{h}_{1}}-T_{\mathrm{h}_{2}}}{T_{\mathrm{h}_{1}}-T_{\mathrm{c}_{1}}}=\frac{80-50}{80-10} \\
& \epsilon_{\mathrm{P}, \mathrm{C}}=\frac{3}{7} \tag{2}
\end{align*}
$$

Form Eqs. (1) and (2)

$$
\frac{3}{7}=\frac{1-\exp (-N T U \times 1.5)}{1.5}
$$

$$
\begin{aligned}
(N T U)_{\mathrm{P}} & =0.686 \\
\epsilon_{\mathrm{c}} & =\frac{1-\exp (-N T U(1-R))}{1-R \exp (-N T U(1-R))} \\
\frac{3}{7} & =\frac{1-\exp (-0.5 N T U)}{1-0.5 \exp (-0.5 N T U)}
\end{aligned}
$$

$$
\begin{aligned}
(N T U)_{\mathrm{C}} & =0.637 \\
N T U & =\frac{U A}{C_{\min }}
\end{aligned}
$$

Now we have

$$
U_{\mathrm{C}}=U_{\mathrm{P}}
$$

$$
\begin{gathered}
\therefore \frac{(N T U)_{\mathrm{C}}}{(N T U)_{\mathrm{P}}}=\frac{A_{\mathrm{C}}}{A_{\mathrm{P}}} \\
\Rightarrow \frac{A_{\mathrm{C}}}{A_{\mathrm{P}}}=\frac{0.637}{0.686}=0.928=0.93
\end{gathered}
$$

Hence, the correct answer is 0.93 .

## Question Number: 53

Question Type: NAT
Two cylindrical shafts $A$ and $B$ at the same initial temperature are simultaneously placed in a furnace. The surfaces of the shafts remain at the furnace gas temperature at all times after they are introduced into the furnace. The temperature variation in the axial direction of the shafts can be assumed to be negligible. The data related to shafts $A$ and $B$ is given in the following table.

| Quantity | Shaft A | Shaft B |
| :--- | :---: | :---: |
| Diameter (m) | 0.4 | 0.1 |
| Thermal conductivity <br> (W/m-K) | 40 | 20 |
| Volumetric heat capacity <br> $\left(\mathrm{J} / \mathrm{m}^{3}-\mathrm{K}\right)$ | $2 \times 10^{6}$ | $2 \times 10^{7}$ |

The temperature at the centerline of the shaft $A$ reaches $400^{\circ} \mathrm{C}$ after two hours. The time required (in hours) for the centerline of the shaft $B$ to attain the temperature of $400^{\circ} \mathrm{C}$ is $\qquad$ —.
Solution: Volumetric heat capacity $\left(\mathrm{J} / \mathrm{m}^{3}-\mathrm{K}\right)$ for shaft $A$ $(\rho C)_{\mathrm{A}}=2 \times 10^{6} \mathrm{~J} / \mathrm{m}^{3} \mathrm{~K}$.
Volumetric heat capacity $\left(\mathrm{J} / \mathrm{m}^{3}-\mathrm{K}\right)$ for shaft $B(\rho C)_{\mathrm{B}}=2 \times$ $10^{7} \mathrm{~J} / \mathrm{m}^{3} \mathrm{~K}$
At the center of cylinder of radius $r$, temperature is

$$
\frac{\theta}{\theta_{i}}=\operatorname{erf}\left(\frac{\alpha \tau}{r^{2}}\right)
$$

In the above relation, $\alpha=\frac{k}{\rho C}$
We know that $\frac{\theta}{\theta_{i}}$ is constant for both shafts.

$$
\begin{aligned}
\therefore \quad\left(\frac{\alpha \tau}{r^{2}}\right)_{1} & =\left(\frac{\alpha \tau}{r^{2}}\right)_{2} \\
\tau_{2} & =2.5 \text { hours }
\end{aligned}
$$

Hence, the correct answer is 2.5 .
Question Number: 54
Question Type: NAT
A piston-cylinder device initially contains $0.4 \mathrm{~m}^{3}$ of air (to be treated as an ideal gas) at 100 kPa and $80^{\circ} \mathrm{C}$. The air is now isothermally compressed to $0.1 \mathrm{~m}^{3}$. The work done during this process is $\qquad$ kJ.
(Take the sign convention such that work done on the system is negative)

Solution: Initial volume of air $V_{1}=0.4 \mathrm{~m}^{3}$ Initial volume of air $V_{2}=0.1 \mathrm{~m}^{3}$
We know that for isothermal process, work done,

$$
\begin{aligned}
W & =P_{1} V_{1} \ln \frac{V_{2}}{V_{1}} \\
& =100 \times 0.4 \ln \frac{0.1}{0.4} \mathrm{~kJ}=-55.45 \mathrm{~kJ}
\end{aligned}
$$

Hence, the correct answer is -55.6 to -55.4 .
Question Number: 55
Question Type: NAT
A reversible cycle receives 40 kJ of heat from one heat source at a temperature of $127^{\circ} \mathrm{C}$ and 37 kJ from another heat source at $97^{\circ} \mathrm{C}$. The heat rejected (in kJ ) to the heat $\sin$ $k$ at $47^{\circ} \mathrm{C}$ is $\qquad$ -.
Solution:


Heat $Q_{1}=40 \mathrm{~kJ}$
Heat $Q_{2}=37 \mathrm{~kJ}$
Temperature $T_{1}=127+273=400 \mathrm{~K}$
Temperature $T_{2}=97+273=370 \mathrm{~K}$
Temperature $T_{3}=47+273=320 \mathrm{~K}$
For reversible cycle

$$
\begin{aligned}
& \frac{Q_{1}}{T_{1}}+\frac{Q_{2}}{T_{2}}=\frac{Q_{3}}{T_{3}} \\
& \frac{40}{400}+\frac{37}{370}=\frac{Q_{3}}{320} \\
& \frac{Q_{3}}{320}=0.2 \\
& Q_{3}
\end{aligned}=320 \times 0.22 \mathrm{~kJ} .
$$

Hence, the correct answer is 64 .
Question Number: 56
Question Type: NAT
A refrigerator uses $\mathrm{R}-134 \mathrm{a}$ as its refrigerant and operates on an ideal vapor-compression refrigeration cycle between 0.14 MPa and 0.8 MPa . If the mass flow rate of
the refrigerant is $0.05 \mathrm{~kg} / \mathrm{s}$, the rate of heat rejection to the environment is $\qquad$ kW .


Given data:
At $P=0.14 \mathrm{MPa}, h=236.04 \mathrm{~kJ} / \mathrm{kg}, s=0.9322 \mathrm{~kJ} / \mathrm{kg}-\mathrm{K}$
At $P=0.8 \mathrm{MPa}, h=272.05 \mathrm{~kJ} / \mathrm{kg}$ (superheated vapor)
At $P=0.8 \mathrm{MPa}, h=93.42 \mathrm{~kJ} / \mathrm{kg}$ (saturated liquid)
Solution: Mass flow rate of the refrigerant $\dot{m}=0.05 \mathrm{~kg} / \mathrm{s}$
For superheated vapor $h_{2}=272.05 \mathrm{~kJ} / \mathrm{kg}$
For saturated liquid $h_{3}=93.42 \mathrm{~kJ} / \mathrm{kg}$
Now heat rejected to atmosphere

$$
\begin{aligned}
Q & =\dot{m}\left(h_{2}-h_{3}\right) \\
& =0.05(272.05-93.42) \\
& =8.93 \mathrm{~kW}
\end{aligned}
$$

Hence, the correct answer is 8.9 to 8.95 .

## Question Number: 57

Question Type: NAT
The partial pressure of water vapor in a moist air sample of relative humidity $70 \%$ is 1.6 kPa , the total pressure being 101.325 kPa . Moist air may be treated as an ideal gas mixture of water vapor and dry air. The relation between saturation temperature ( $T_{\mathrm{s}}$ in K ) and saturation pressure ( $p_{\mathrm{s}}$ in kPa ) for water is given by $\ln \left(p_{\mathrm{s}} / p_{\mathrm{o}}\right)=14.317-5304 / T_{\mathrm{s}}$, where $p_{o}=101.325 \mathrm{kPa}$. The dry bulb temperature of the moist air sample (in ${ }^{\circ} \mathrm{C}$ ) is $\qquad$ _.
Solution: Relative humidity $\Phi=0.7$
Now we know that

$$
\begin{aligned}
\Phi & =\frac{P_{\mathrm{v}}}{P_{\mathrm{vs}}} \\
P_{\mathrm{vs}} & =P_{\mathrm{v}} / \Phi=2.285 \mathrm{kPa}
\end{aligned}
$$

Now using the relation

$$
\begin{array}{r}
\ln \frac{P_{\mathrm{s}}}{P_{\mathrm{o}}}=14.317-\frac{5304}{T_{\mathrm{s}}} \\
\ln \frac{2.285}{101.325}=14.317-\frac{5304}{T_{\mathrm{s}}}
\end{array}
$$

The dry bulb temperature of the moist air sample will be

$$
T_{\mathrm{s}}=292.9 \mathrm{~K}=19.9^{\circ} \mathrm{C}
$$

Hence, the correct answer is 19.9.

Question Number: 58
Question Type: MCQ
In a binary system of $A$ and $B$, a liquid of $20 \% A(80 \% B)$ is coexisting with a solid of $70 \% A(30 \% B)$. For an overall composition having $40 \% A$, the fraction of solid is
(A) 0.40
(B) 0.50
(C) 0.60
(D) 0.75

Solution: Using lever rule,

$$
\begin{align*}
W_{\alpha}+W_{\mathrm{L}} & =1  \tag{1}\\
\mathrm{~W}_{\alpha} \mathrm{C}_{\alpha}+W_{\mathrm{L}} C_{\mathrm{L}} & =C_{\mathrm{o}} \\
W_{\alpha}(0.7)+W_{\mathrm{L}}(0.2) & =0.4 \tag{2}
\end{align*}
$$

Solving Eqs. (1) and (2), we get

$$
W_{\alpha}=0.4, W_{\mathrm{L}}=0.6
$$

Thus, fraction of solid $=0.4$
where $W_{\alpha}$ is fraction of solid and $W_{\mathrm{L}}$ is fraction of liquid, $C$ is composition.
Hence, the correct option is (A).
Question Number: 59
Question Type: NAT
Gray cast iron blocks of size $100 \mathrm{~mm} \times 50 \mathrm{~mm} \times 10 \mathrm{~mm}$ with a central spherical cavity of diameter 4 mm are sand cast. The shrinkage allowance for the pattern is $3 \%$. The ratio of the volume of the patter to the volume of the casting is $\qquad$ —.
Solution: Shrinkage allowance $=3 \%=0.03$
Volume of casting
$V_{\mathrm{c}}=(100 \times 50 \times 10)-\left(\frac{4}{3} \pi \times 2^{3}\right)$
$=49966.5 \mathrm{~mm}^{3}$
Volume of pattern

$$
\begin{aligned}
& V_{\mathrm{p}}=(100 \times 1.03) \times(50 \times 1.03) \times(10 \times 1.03)-\frac{4}{3} \pi(2 \times \\
&1.03)^{3} \\
&= 54599.65 \mathrm{~mm}^{3} \\
& \therefore \frac{\text { Volume of pattern }}{\text { Volume of casting }}=V_{\mathrm{p}} / V_{\mathrm{s}}=(54599.65) /(49966.5) \\
&=1.09
\end{aligned}
$$

Hence, the correct answer is 1.08 to 1.10 .
Question Number: 60
Question Type: NAT
The voltage-length characteristic of a direct current arc in an arc welding process is $V=(100+40 l)$, where $l$ is the length of the arc in mm and $V$ is the arc voltage in volts. During a welding operation, the arc length varies between 1 and 2 mm and the welding current is in the range 200-250 A. Assuming a linear power source, the short circuit current is $\qquad$ A.

Solution: Length $l_{1}=1 \mathrm{~mm}$,
Welding current $I_{1}=250 \mathrm{~A}$

Length $l_{2}=2 \mathrm{~mm}$,
Welding current $I_{2}=200 \mathrm{~A}$
Voltage $V_{1}=140 \mathrm{~V}$,
Voltage $V_{2}=180 \mathrm{~V}$
The voltage-length characteristic of a direct current arc in an arc welding process is

$$
\begin{aligned}
V & =100+40 l \\
\frac{V}{V_{\mathrm{oc}}}+\frac{I}{I_{\mathrm{sc}}} & =1 \\
V & =V_{\mathrm{oc}}\left(1-\frac{I}{I_{\mathrm{sc}}}\right)
\end{aligned}
$$

We get,

$$
\begin{align*}
& 140=V_{\mathrm{oc}}\left(1-\frac{250}{I_{\mathrm{sc}}}\right)  \tag{1}\\
& 180=V_{\mathrm{oc}}\left(1-\frac{200}{I_{\mathrm{sc}}}\right) \tag{2}
\end{align*}
$$

On solving Eqs. (1) and (2), we get

$$
\begin{aligned}
0.777\left(1-\frac{200}{I_{\mathrm{sc}}}\right) & =1-\frac{250}{I_{\mathrm{sc}}} \\
I_{\mathrm{sc}} & =424 \mathrm{~A}
\end{aligned}
$$

Hence, the correct answer is 423 to 428 .
Question Number: 61
Question Type: NAT
For a certain job, the cost of metal cutting in Rs. $18 \mathrm{C} / \mathrm{V}$ and the cost of tooling is Rs. $2701 C /(T V)$, where $C$ is a constant, $V$ is the cutting speed in $\mathrm{m} / \mathrm{min}$, and $T$ is the tool life in minutes. The Taylor's tool life equation is $V T^{0.25}=$ 150. The cutting speed (in $\mathrm{m} / \mathrm{min}$ ) for the minimum total cost is $\qquad$ -.

## Solution:

Taylor's tool life equation is $V T^{0.25}=150$
Cost of metal cutting $C_{\mathrm{t}}=\frac{18 C}{V}$
Cost of tooling $C_{\mathrm{m}}=\frac{270 C}{T V}$
Let the cutting speed (in $\mathrm{m} / \mathrm{min}$ ) for the minimum total cost be $V$

$$
\begin{aligned}
\text { Total cost } & =C_{\mathrm{t}}+C_{\mathrm{m}} \\
& =\frac{18 \times 150}{V}+\frac{270 \times 150}{\frac{(150)^{4}}{V^{4}} \times V} \\
& =\frac{2700}{V}+\left(8 \times 10^{-5}\right) V^{3}
\end{aligned}
$$

Now we have

$$
\begin{gathered}
\frac{d C_{\text {Total }}}{d V}=0 \\
\frac{-2700}{V^{2}}+3 \times 8 \times 10^{-5} V^{2}=0
\end{gathered}
$$

$$
V=57.91 \mathrm{~m} / \mathrm{min}
$$

Hence, the correct answer is 57.8 to 58 .
Question Number: 62
Question Type: NAT
The surface irregularities of electrodes used in an electrochemical machining (ECM) process are $3 \mu \mathrm{~m}$ and $6 \mu \mathrm{~m}$ as shown in the figure. If the work-piece is of pure iron and 12 V DC is applied between the electrodes, the largest feed rate is $\qquad$ $\mathrm{mm} / \mathrm{min}$.

| Conductivity of the electrolyte | $0.02 \mathrm{ohm}^{-1} \mathrm{~mm}^{-1}$ |
| :--- | :--- |
| Over-potential voltage | 1.5 V |
| Density of iron | $7860 \mathrm{~kg} / \mathrm{m}^{3}$ |
| Atomic weight of iron | 55.85 gm |

Assume the iron to be dissolved as $\mathrm{Fe}^{+2}$ and the Faraday constant to be 96,500 Coulomb.


Work-piece (Iron)

Solution: Density of iron $\rho=7860 \mathrm{~kg} / \mathrm{m}^{3}=7.86 \mathrm{~g} / \mathrm{cm}^{3}$
Conductivity of the electrolyte $C=0.02 \mathrm{ohm}^{-1} \mathrm{~mm}^{-1}=0.2$ ohm ${ }^{-1} \mathrm{~cm}^{-1}$

Over-potential voltage $\Delta V=1.5 \mathrm{~V}$
Voltage applied between the electrodes $V=12 \mathrm{~V}$
Atomic weight of iron $A=55.85 \mathrm{gm}$
$Z=2$, Faraday constant $F=96,500$ Coulomb Maximum feed rate can be calculated as

$$
\begin{aligned}
f_{\max } & =\frac{C A(V-\Delta V)}{\rho Z F y_{\mathrm{e}}} \mathrm{~m} / \mathrm{s} \\
& =\frac{0.2 \times 55.85 \times(12-1.5)}{7.86 \times 2 \times 96500 \times 9 \times 10^{-4}} \\
& =0.0852 \mathrm{~cm} / \mathrm{s}=51.12 \mathrm{~mm} / \mathrm{min}
\end{aligned}
$$

Hence, the correct answer is 51 to 52 .

$$
\begin{aligned}
& \text { Total gap }\left(y_{c}\right)=3 \times 10^{-6}+6 \times 10^{-6} \\
& =9 \times 10^{-6} \mathrm{~m}=9 \times 10^{-4} \mathrm{~cm}
\end{aligned}
$$

Question Number: 63
Question Type: MCQ
For the situation shown in the figure below, the expression for $H$ in terms of $r, R$, and $D$ is

(A) $H=D+\sqrt{r^{2}+R^{2}}$
(B) $H=(R+r)+(D+r)$
(C) $H=(R+r)+\sqrt{D^{2}-R^{2}}$
(D) $H=(R+r)+\sqrt{2 D(R+r)-D^{2}}$


Solution: From the above figure, we get

$$
\begin{aligned}
x & =D-(R+r) \\
y & =\sqrt{(R+r)^{2}-(D-(R+r))^{2}} \\
& =\sqrt{2 D(R+r)-D^{2}}
\end{aligned}
$$

Now we have

$$
\begin{aligned}
& H=R+r+y \\
& H=(R+r)+\sqrt{2 D(R+r)-D^{2}}
\end{aligned}
$$

Hence, the correct option is (D).

Question Number: 64
Question Type: NAT
A food processing company uses $25,000 \mathrm{~kg}$ of corn flour every year. The quantity-discount price of corn flour is provided in the table below:

| Quantity (kg) | Unit price (Rs./kg) |
| :---: | :---: |
| $1-749$ | 70 |
| $750-1499$ | 65 |
| 1500 and above | 60 |

The order processing charges are Rs. 500/order. The handling plus carry-over charge on an annual basis is $20 \%$ of the purchase price of the corn flour per kg. The optimal order quantity (in kg ) is $\qquad$ —.

Solution: Corn flour used every year $D=25,000 \mathrm{~kg}$
Order processing charges $C_{\mathrm{o}}=$ Rs. $500 /$ order
Handling plus carry-over charge on an annual basis $i=$ $20 \%=0.2$

Now using the relation

$$
E O Q=\sqrt{\frac{2 D C_{\mathrm{o}}}{C_{\mathrm{u}} \times i}}
$$

$C_{\mathrm{u}}$ is unit price
For $\quad C_{\mathrm{u}}=$ Rs. $70 / \mathrm{kg}$

$$
(E O Q)_{70}=\sqrt{\frac{2 \times 25,000 \times 500}{70 \times 0.2}}=1336.30 \mathrm{~kg}
$$

For $\quad C_{\mathrm{u}}=$ Rs. $65 / \mathrm{kg}$

$$
\begin{aligned}
(E O Q)_{65} & =\sqrt{\frac{2 \times 25,000 \times 500}{65 \times 0.2}} \\
& =1386.75 \mathrm{~kg} \\
C_{\mathrm{u}} & =\text { Rs. } 60 / \mathrm{kg} \\
(E O Q)_{60} & =\sqrt{\frac{2 \times 25,000 \times 500}{60 \times 0.2}}=1443.37 \mathrm{~kg}
\end{aligned}
$$

On for $C_{\mathrm{u}}=$ Rs. $65, E O Q$ is lying within the given quantity band,

$$
\begin{aligned}
& (T V C)_{65}=\sqrt{2 D C_{0} C_{\mathrm{u}} i}=\text { Rs. } 18,027.75 \\
& (T V C)_{60}=\left(\frac{D}{Q}\right) 6+\frac{Q}{2} \times\left(C_{\mathrm{u}} i\right)=\text { Rs. } 17,333.33
\end{aligned}
$$

where,

$$
\begin{aligned}
Q & =1500 \\
(T V C)_{60} & <(T V C)_{65}
\end{aligned}
$$

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$\therefore \quad E O Q=1500$
Hence, the correct answer is 1500 .
Solution: Critical path is $1-2-4-7-9-11-12$

Critical time $=19$ days
Latest finish time for node $10=19-(2+3)=14$ days
Hence, the correct answer is 14 .


# GATE 2016 Solved Paper ME: Mechanical Engineering Set - 3 

## Number of Questions: 65

Total Marks:100.0

Wrong answer for MCQ will result in negative marks, (-1/3) for 1 Mark Questions and (-2/3) for 2 Marks Questions.

## General Aptitude

## Number of Questions: 10

Section Marks: 15.0
Q. 1 to Q. 5 carry 1 mark each and Q. 6 to Q. 10 carry 2 marks each.

## Question Number: $1 \quad$ Question Type: MCQ

Based on the given statements, select the appropriate option with respect to grammar and usage.

## Statements

(i) The height of Mr . $\mathbf{X}$ is 6 feet.
(ii) The height of Mr. $\mathbf{Y}$ is 5 feet.
(A) Mr. $\mathbf{X}$ is longer than Mr. $\mathbf{Y}$.
(B) $\mathrm{Mr} . \mathbf{X}$ is more elongated than $\mathrm{Mr} . \mathbf{Y}$.
(C) $\mathrm{Mr} . \mathbf{X}$ is taller than $\mathrm{Mr} . \mathbf{Y}$.
(D) $\mathrm{Mr} . \mathbf{X}$ is lengthier than $\mathrm{Mr} . \mathbf{Y}$.

Solution: 'tall' is used to denote the height of a person. As the comparison is made between the heights of two people, the comparative adjective 'taller' is correct to use here.
Hence, the correct option is (C).

## Question Number: 2

Question Type: MCQ
The students $\qquad$ the teacher on teachers' day for twenty years of dedicated teaching.
(A) facilitated
(B) felicitated
(C) fantasized
(D) facillitated

Solution: 'Felicitated' means to compliment upon a happy event or to congratulate. The word facilitate is to aid or help. The word fantasize is envision or daydream.
Hence, the correct option is (B).
Question Number: 3
Question Type: MCQ
After India's cricket world cup victory in 1985, Shrotria who was playing both tennis and cricket till then, decided to concentrate only on cricket. And the rest is history.
What does the underlined phrase mean in this context?
(A) history will rest in peace
(B) rest is recorded in history books
(C) rest is well known
(D) rest is archaic

Solution: The correct option is (C).

Question Number: 4
Question Type: MCQ
Given $(9 \text { inches })^{1 / 2}=(0.25 \text { yards })^{1 / 2}$, which one of the following statements is TRUE?
(A) 3 inches $=0.5$ yards
(B) 9 inches $=1.5$ yards
(C) 9 inches $=0.25$ yards
(D) 81 inches $=0.0625$ yards

Solution: $(9 \text { inches })^{1 / 2}=(0.25 \text { yards })^{1 / 2}$
Squaring both sides, we get
9 inches $=0.25$ yards.
Hence, the correct option is (C).
Question Number: 5
Question Type: MCQ
$\boldsymbol{S}, \boldsymbol{M}, \boldsymbol{E}$, and $\boldsymbol{F}$ are working in shifts in a team to finish a project. $\boldsymbol{M}$ works with twice the efficiency of others but for half as many days as $\boldsymbol{E}$ worked. $\boldsymbol{S}$ and $\boldsymbol{M}$ have 6-hour shifts in a day, whereas $\boldsymbol{E}$ and $\boldsymbol{F}$ have 12-hour shifts. What is the ratio of contribution of $\boldsymbol{M}$ to contribution of $\boldsymbol{E}$ in the project?
(A) $1: 1$
(B) $1: 2$
(C) $1: 4$
(D) $2: 1$

Solution: Consider the table given below:

|  | $\boldsymbol{S}$ | $\boldsymbol{M}$ | $\boldsymbol{E}$ | $\boldsymbol{F}$ |
| :--- | :--- | :--- | :--- | :--- |
| Efficiency | 1 | 2 | 1 | 1 |
| Hours/day | 6 | 6 | 12 | 12 |
| Days |  | $n$ | $2 n$ |  |

From the above data, the ratio of the work done by $M$ and $E$ is

$$
W_{\mathrm{M}} / W_{\mathrm{E}}=\frac{12 n}{12(2 n)}=\frac{1}{2} .
$$

Hence, the correct option is (B).
Question Number: 6
Question Type: MCQ
The Venn diagram shows the preference of the student population for leisure activities.

From the data given, the number of students who like to read books or play sports is $\qquad$ _.

(A) 44
(B) 51
(C) 79
(D) 108

Solution: Consider the Venn diagram given below:


The number of students who read books or play sports is $n(R U S)$

$$
\begin{aligned}
& =(13+12+7+44)+(17+15) \\
& =76+32=108
\end{aligned}
$$

Hence, the correct option is (D).
Question Number: 7
Question Type: MCQ
Social science disciplines were in existence in an amorphous form until the colonial period when they were institutionalized. In varying degrees, they were intended to further the colonial interest. In the time of globalization and the economic rise of postcolonial countries like India, conventional ways of knowledge production have become obsolete.
Which of the following can be logically inferred from the above statements?
(i) Social science disciplines have become obsolete.
(ii) Social science disciplines had a pre-colonial origin.
(iii) Social science disciplines always promote colonialism.
(iv) Social science must maintain disciplinary boundaries.
(A) (ii) only
(B) (i) and (iii) only
(C) (ii) and (iv) only
(D) (iii) and (iv) only

Solution: Hence options (i) and (iii) are incorrect. Statement (iv) is not stated. Statement (ii) can be understood
from the first sentence of the passage.
Hence, the correct option is (A).
Question Number: 8
Question Type: MCQ
Two and a quarter hours back, when seen in a mirror, the reflection of a wall clock without number markings seemed to show 1:30. What is the actual current time shown by the clock?

(A) $8: 15$
(B) $11: 15$
(C) $12: 15$
(D) $12: 45$

Solution: Figure below shows the actual positions of the hands of the clock and the reflection
The actual time, two and a quarter hours back, was 10:30. Now it is 12:45.

Hence, the correct option is (D).

## Question Number: 9

Question Type: MCQ
$M$ and $N$ start from the same location. $M$ travels 10 km East and then 10 km North-East. $N$ travels 5 km South and then 4 km South-East. What is the shortest distance (in km) between $M$ and $N$ at the end of their travel?
(A) 18.60
(B) 22.50
(C) 20.61
(D) 25.00

## Solution:



The starting point for both M and N is 0 , the origin. The final positions are

$$
\begin{aligned}
& M=(10+5 \sqrt{2}, 5 \sqrt{2}) \text { and } N=(2 \sqrt{2},-5-2 \sqrt{2}) \\
& \left.\therefore \quad \begin{array}{rl}
\therefore N^{2} & =(10+3 \sqrt{2})^{2}+(5+7 \sqrt{2})^{2} \\
& =(118+60 \sqrt{2})+(123+70 \sqrt{2}) \\
\Rightarrow & \\
& =241+130 \sqrt{2} \\
\Rightarrow & M N
\end{array}\right)=20.61 .
\end{aligned}
$$

Hence, the correct option is (C).
Question Number: $10 \quad$ Question Type: MCQ
A wire of length 340 mm is to be cut into two parts. One of the parts is to be made into a square and the other into a rectangle where sides are in the ratio of $1: 2$. What is the length of the side of the square (in mm ) such that the combined area of the square and the rectangle is a MINIMUM?
(A) 30
(B) 40
(C) 120
(D) 180

Solution: Length of the wire $=340 \mathrm{~m}$

If the side of the square be $a$, let the breadth and length of the rectangle be $b$ and $2 b$, respectively.

$$
\begin{array}{r}
4 a+2(b+2 b)=340 \\
2 a+3 b=170
\end{array}
$$

We need the value of $a$ for which $y=a^{2}+2 b^{2}$ has the minimum possible value

$$
\begin{array}{rlrl} 
& & y & =a^{2}+2\left(\frac{170-2 a}{3}\right)^{2} \\
& \Rightarrow & 9 y & =9 a^{2}+8(85-a)^{2}  \tag{1}\\
9 y & =17 a^{2}-8(17 a)+8(85)^{2} \\
\therefore & & \frac{d}{d a}(9 y) & =34 a-8(170)=0 \\
\Rightarrow & a & =40
\end{array}
$$

Form Eq. (1), we conclude that the graph of $9 y$ vesus $a$ is a parabola and $y$ has the minimum value at $a=40$.
Hence, the correct option is (B).

## Mechanical Engineering

## Number of Questions: 55

Q. 11 to Q. 35 carry 1 mark each and Q. 36 to Q. 65 carry 2 marks each.

## Question Number: 11

Question Type: MCQ
A real square matrix $A$ is called skew-symmetric if
(A) $A^{\mathrm{T}}=A$
(B) $A^{\mathrm{T}}=A^{-1}$
(C) $A^{\mathrm{T}}=-A$
(D) $A^{\mathrm{T}}=A+A^{-1}$

Solution: A real square matrix $A$ is called skew-symmetric if $A^{\mathrm{T}}=-A$
Hence, the correct option is (C).

## Question Number: 12

## Question Type: MCQ

$\operatorname{Lt}_{x \rightarrow 0} \frac{\log _{e}(1+4 x)}{e^{3 x}-1}$ is equal to
(A) 0
(B) $\frac{1}{2}$
(C) $\frac{4}{3}$
(D) 1

## Solution:

$$
\operatorname{Lt}_{x \rightarrow 0} \frac{\log _{e}(1+4 x)}{e^{3 x}-1}=\underset{x \rightarrow 0}{\operatorname{Lt}} \frac{\left(\frac{4}{1+4 x}\right)}{3 e^{3 x}}
$$

(By using L'Hospital's rule)

$$
=\frac{4}{3}
$$

Hence, the correct option is (C).

Section marks: 85.0

## Question Number: 13

Question Type: MCQ
Solutions of Laplace's equation having continuous secondorder partial derivatives are called
(A) biharmonic functions
(B) harmonic functions
(C) conjugate harmonic functions
(D) error functions

Solution: Solutions of Laplace's equation having continuous second-order partial derivatives are called harmonic functions.

Hence, the correct option is (B).
Question Number: 14

## Question Type: NAT

The area (in percentage) under standard normal distribution curve of random variable $Z$ within limits from -3 to +3 is $\qquad$ -.
Solution: The area (in percentage) under standard normal distribution curve of random variable $Z$ within limits from -3 to +3 is 99.74 .
Hence, the correct answer is 99.74 .
Question Number: 15
Question Type: MCQ
The root of the function $f(x)=x^{3}+x-1$ obtained after the first iteration on application of Newton-Raphson scheme suing an initial guess of $x_{0}=1$ is
(A) 0.682
(B) 0.686
(C) 0.750
(D) 1.000
c | GATE 2016 Solved Paper ME: Set - 3

Solution: The given function is

$$
f(x)=x^{3}+x-1
$$

Derivative of the function will be

$$
F^{\prime}(\mathrm{x})=3 \mathrm{x}^{2}+1
$$

At $x_{0}=1$, the function and its derivative will be $f\left(x_{0}\right)=f(1)$
$=1$ and $f^{\prime}\left(x_{0}\right)=f^{\prime}(1)=4$
By Newton-Raphson method,

$$
\begin{aligned}
x_{1} & =x_{0}-\frac{f\left(x_{0}\right)}{f^{\prime}\left(x_{0}\right)} \\
& =1-\frac{1}{4}=0.75
\end{aligned}
$$

Hence, the correct option is (C).
Question Number: 16
Question Type: MCQ
A force $F$ is acting on a bent bar which is clamped at one end as shown in the figure.


The CORRECT free-body diagram is

(A)

(B)

(C)

(D)

Solution: The required free-body diagram is given below:


All supports have to be isolated in F.B.D.
Hence, the correct option is (A).
Question Number: 17
Question Type: MCQ
7.The cross-sections of two solid bars made of the same material are shown in the figure. The square cross-section has flexural (bending) rigidity $I_{1}$, while the circular crosssection has flexural rigidity $I_{2}$. Both sections have the same cross-sectional area. The ratio $I_{1} / I_{2}$ is

(A) $1 / \pi$
(B) $2 / \pi$
(C) $\pi / 3$
(D) $\pi / 6$

Solution: Let the side of square be $a \mathrm{~cm}$ and circular diameter be $d \mathrm{~cm}$.
Now area of square $=a^{2}$
and

$$
\text { area of circle }=\frac{\pi}{4} d^{2}
$$

Since square and circle have equal areas

$$
a^{2}=\frac{\pi}{4} d^{2}
$$

Now using the relation

$$
\frac{I_{1}}{I_{2}}=\frac{a^{4}}{12 \times \frac{\pi d^{4}}{64}}=\frac{64}{12 \pi} \times \frac{\pi^{2}}{16}=\frac{\pi}{3}
$$

Hence, the correct option is (C).
Question Number: 18
Question Type: MCQ
The state of stress at a point on an element is shown in figure (a). The same state of stress is shown in another coordinate system in figure (b).

(a)

(b)

The components $\left(\tau_{x x}, \tau_{y y}, \tau_{x y}\right)$ are given by
(A) $(p / \sqrt{2},-p / \sqrt{2}, 0)$
(B) $(0,0, p)$
(C) $(p,-p, p / \sqrt{2})$
(D) $(0,0, p / \sqrt{2})$

## Solution:



Given principal stresses are equal and opposite and are acting at a plane inclined at $\theta=45^{\circ}$ with respect to $x$-axis.

Which is a pure shear state of stress? So, $\tau_{x y}=p, \tau_{y y}=$ $\tau_{x x}=0$.
Hence, the correct option is (B).
Question Number: 19
Question Type: MCQ
A rigid link $P Q$ is undergoing plane motion as shown in the figure ( $V_{\mathrm{P}}$ and $V_{\mathrm{Q}}$ are non-zero). $V_{\mathrm{QP}}$ is the relative velocity of point $Q$ with respect to point $P$.


Which one of the following is TRUE?
(A) $V_{\mathrm{QP}}$ has components along and perpendicular to $P Q$.
(B) $V_{\mathrm{QP}}$ has only one component directed from $P$ to $Q$.
(C) $V_{\mathrm{QP}}$ has only one component directed from $Q$ to $P$.
(D) $V_{\mathrm{QP}}$ has only one component perpendicular to $P Q$.

Solution: $V_{\mathrm{QP}}$ will have one component perpendicular to $P Q$.
Hence, the correct option is (D).
Question Number: 20
Question Type: MCQ
The number of degrees of freedom in a planer having $n$ links and $j$ simple hinge joints is
(A) $3(n-3)-2 j$
(B) $3(n-1)-2 j$
(C) $3 n-2 j$
(D) $2 j-3 n+4$

Solution: The number of degrees of freedom in a planer having $n$ links and $j$ simple hinge joints is $3(n-1)-2 j$.
Hence, the correct option is (B).
Question Number: 21
Question Type: NAT
The static deflection of a spring under gravity, when a mass of 1 kg is suspended from it, is 1 mm . Assume the acceleration due to gravity $g=10 \mathrm{~m} / \mathrm{s}^{2}$. The natural frequency of this spring-mass system (in rad/s) is $\qquad$ -.
Solution: Mass $m=1 \mathrm{~kg}$
Static deflection of spring $\delta=1 \mathrm{~mm}$
Acceleration due to gravity $g=10 \mathrm{~m} / \mathrm{s}^{2}$
Let the natural frequency of this spring-mass system $\omega_{n}=$ ?
Now using the relation

$$
\omega_{n}=\sqrt{g / \delta}=\sqrt{\frac{10}{1 \times 10^{-3}}}=100 \mathrm{rad} / \mathrm{s}
$$

Hence, the correct answer is 100 .

## Question Number: 22

Question Type: MCQ
Which of the bearings given below SHOULD NOT be subjected to a thrust load?
(A) Deep groove ball bearing
(B) Angular contact ball bearing
(C) Cylindrical (straight) roller bearing
(D) Single row tapered roller bearing

Solution: Cylindrical roller bearing should not be subjected to thrust loads.
Hence, the correct option is (C).
Question Number: 23
Question Type: NAT
A channel of width 450 mm branches into two sub-channels having width 300 mm and 200 mm as shown in the figure. If the volumetric flow rate (taking unit depth) of an incompressible flow through the main channel is $0.9 \mathrm{~m}^{3} / \mathrm{s}$ and the velocity in the sub-channel of width 200 mm is 3 $\mathrm{m} / \mathrm{s}$, the velocity in the sub-channel of width 300 mm is
$\qquad$ $\mathrm{m} / \mathrm{s}$.
Assume both inlet and outlet to be at the same elevation.


## Solution:



From the above figure, equating mass flow rate, we get

$$
\begin{aligned}
0.9 & =0.3 V+0.2 \times 3 \\
0.9 & =0.3 V+0.6 \\
0.3 V & =0.3 \\
V & =1 \mathrm{~m} / \mathrm{s}
\end{aligned}
$$

Hence, the correct answer is 1 .

## Question Number: 24

Question Type: MCQ
For a certain two-dimensional incompressible flow, velocity field is given by $2 x y \hat{i}-y^{2} \hat{j}$. The streamlines for this flow are given by the family of curves
(A) $x^{2} y^{2}=\mathrm{constant}$
(B) $x y^{2}=$ constant
(C) $2 x y-y^{2}=$ constant
(D) $x y=$ constant

Solution: Given that velocity field is $\bar{V}=2 x y \hat{i}-y^{2} \hat{j}$
$\Psi=$ ?

$$
\begin{aligned}
-u=\frac{\delta \psi}{\delta y} & , V=-\frac{\delta \psi}{\delta x} \\
-2 x y d y & =d \Psi \\
\Psi & =-x y^{2}+F(x) \\
\frac{\delta \psi}{\delta x} & =f^{\prime}(x)-y^{2} \\
F^{\prime}(x) & =0 \Rightarrow F(x)=\text { Constant } \\
\Rightarrow \quad \Psi & =x y^{2}=\text { constant. }
\end{aligned}
$$

Hence, the correct option is (B).
Question Number: 25
Question Type: NAT
Steady one-dimensional heat conduction takes place across the faces 1 and 3 of a composite slab consisting of slabs $A$ and $B$ in perfect contact as shown in the figure, where $k_{\mathrm{A}}$, $k_{\mathrm{B}}$ denote the respective thermal conductivities. Using the data given in the figure, the interface temperature $T_{2}\left(\right.$ in $\left.{ }^{\circ} \mathrm{C}\right)$ is $\qquad$ —.


Solution: Now using the relation

$$
\begin{aligned}
Q_{13} & =Q_{12}=Q_{23} \\
\frac{130-30}{\frac{0.1}{20}+\frac{0.3}{100}} & =\frac{130-T_{2}}{\frac{0.1}{20}} \\
\Rightarrow \quad T_{2} & =67.5^{\circ} \mathrm{C}
\end{aligned}
$$

Hence, the correct answer is 67.5 .
Question Number: 26
Question Type: MCQ
Grash of number signifies the ratio of
(A) inertia force to viscous force
(B) buoyancy force to viscous force
(C) buoyancy force to inertia force
(D) inertia force to surface tension force

Solution: Grashof number is the ratio of buoyancy force to viscous force and is expressed as

$$
G_{\mathrm{r}}=\frac{g \beta \Delta T L^{3}}{v^{2}} .
$$

Hence, the correct option is (B).

## Question Number: 27

Question Type: MCQ
The INCORRECT statement about the characteristics of critical point of a pure substance is that
(A) there is no constant temperature vaporization process
(B) it has point of inflection with zero slope
(C) the ice directly converts from solid phase to vapor phase
(D) saturated liquid and saturated vapor states are identical

Solution: The ice directly converts from solid phase to vapor phase, this statement is false. Critical point is used to specifically denote the vapor-liquid critical point of a material, above which distinct liquid and gas phases do not exist.
Hence, the correct option is (C).
Question Number: 28
Question Type: MCQ
For a heat exchanger, $\Delta T_{\max }$ is the maximum temperature difference and $\Delta T_{\text {min }}$ is the minimum temperature difference between the two fluids. $L M T D$ is the $\log$ mean temperature difference. $C_{\text {min }}$ and $C_{\text {max }}$ are the minimum and the maximum heat capacity rates. The maximum possible heat transfer $\left(Q_{\max }\right)$ between the two fluids is
(A) $C_{\text {min }} L M T D$
(B) $C_{\text {min }} \Delta T_{\text {max }}$
(C) $C_{\max } \Delta T_{\text {max }}$
(D) $C_{\text {max }} \Delta T_{\text {min }}$

Solution: The maximum possible heat transfer $\left(Q_{\max }\right)$ between the two fluids is

$$
Q_{\max }=(\Delta T)_{\max } C_{\min } .
$$

Hence, the correct option is (B).
Question Number: 29
Question Type: NAT
The blade and fluid velocities for an axial turbine are as shown in the figure.


The magnitude of absolute velocity at entry is $300 \mathrm{~m} / \mathrm{s}$ at an angle of $65^{\circ}$ to the axial direction, while the magnitude of the absolute velocity at exit is $150 \mathrm{~m} / \mathrm{s}$. The exit velocity vector has a component in the downward direction. Given that the axial (horizontal) velocity is the same at entry and exit, the specific work (in $\mathrm{kJ} / \mathrm{kg}$ ) is $\qquad$ -.

Solution: Entry angle $\alpha=65^{\circ}$,
Absolute $u=150 \mathrm{~m} / \mathrm{s}$

Magnitude of the absolute velocity at entry $V_{1}=300 \mathrm{~m} / \mathrm{s}$,


Magnitude of the absolute velocity at exit $V_{2}=150 \mathrm{~m} / \mathrm{s}$ We know that

$$
\begin{aligned}
V_{\mathrm{F}_{1}} & =V_{\mathrm{F}_{2}} \\
V_{1} \cos \alpha & =V_{2} \cos \theta \\
300 \cos 65 & =150 \cos \theta \\
\theta & =32.3^{\circ}
\end{aligned}
$$

Now

$$
\begin{aligned}
& V_{\omega_{1}}=V_{1} \sin \alpha=300 \sin 65=272 \mathrm{~m} / \mathrm{s} \\
& V_{\omega_{2}}=V_{2} \sin \theta=150 \sin 32.3=80.15 \mathrm{~m} / \mathrm{s}
\end{aligned}
$$

The specific work (in $\mathrm{kJ} / \mathrm{kg}$ )

$$
\begin{aligned}
W & =\left(V_{\omega_{1}}+V_{\omega_{2}}\right) u \\
& =(272+80.15) 150=52.82 \mathrm{~kJ} / \mathrm{kg}
\end{aligned}
$$

Hence, the correct answer is $50-54$.
Question Number: 30
Question Type: MCQ
Engineering strain of a mild steel sample is recorded as $0.100 \%$. The true strain is
(A) $0.010 \%$
(B) $0.055 \%$
(C) $0.099 \%$
(D) $0.101 \%$

Solution:
Engineering strain $e=0.1 \%=\frac{0.1}{100}$
True strain $\varepsilon_{\mathrm{T}}=$ ?
The true strain can be calculated using

$$
\begin{aligned}
\varepsilon_{\mathrm{T}} & =\ln (1+e)=\ln (1+0.001)=\ln (1.001) \\
& =9.99 \times 10^{-4}=9.99 \times 10^{-2} \%=0.099 \%
\end{aligned}
$$

Hence, the correct option is (C).
Question Number: 31
Question Type: MCQ
Equal amounts of a liquid metal at the same temperature are poured into three moulds made of steel, copper, and aluminum. The shape of the cavity is a cylinder with 15 mm diameter. The sizes of the moulds are such that the
outside temperatures of the moulds do not increase appreciably beyond the atmospheric temperature during solidification. The sequence of solidification in the mould from the fastest to slowest is
(Thermal conductivities of steel, copper, and aluminum are $60.5,401$, and $237 \mathrm{~W} / \mathrm{m}-\mathrm{K}$, respectively Specific heats of steel, copper, and aluminum are 434,385 , and $903 \mathrm{~J} / \mathrm{kg}-\mathrm{K}$, respectively. Densities of steel, copper, and aluminum are 7854,8933 , and $2700 \mathrm{~kg} / \mathrm{m}^{3}$, respectively.)
(A) Copper - Steel - Aluminum
(B) Aluminum - Steel - Copper
(C) Copper - Aluminum - Steel
(D) Steel - Copper - Aluminum

Solution: Thermal diffusivity $\alpha=\frac{k}{\rho C}$
Thermal conductivity is $k$
Density $=\rho$
Specific heat $=C$
For copper

$$
\begin{aligned}
\alpha_{\mathrm{copper}} & =\left(\frac{k}{\rho C}\right)_{\mathrm{copper}}=\frac{401}{3439} \times 10^{-3} \\
& \approx 0.1166 \times 10^{-3} \mathrm{~m}^{2} / \mathrm{s}
\end{aligned}
$$

For steel

$$
\begin{aligned}
\alpha_{\text {steel }} & =\left(\frac{k}{\rho C}\right)_{\text {steel }}=\frac{60.5 \times 10^{-3}}{3400} \\
& \approx 0.0177 \times 10^{-3} \mathrm{~m}^{2} / \mathrm{s}
\end{aligned}
$$

For aluminum

$$
\begin{aligned}
\alpha_{\text {aluminum }} & =\left(\frac{k}{\rho C}\right)_{\text {aluminum }} \\
& =\frac{237}{2430} \times 10^{-3} \\
& \approx 0.1119 \times 10^{-3} \mathrm{~m}^{2} / \mathrm{s}
\end{aligned}
$$

Hence, the correct option is (C).
Question Number: 32
Question Type: MCQ
In a wire-cut EDM process, the necessary conditions that have to be met for making a successful cut are that
(A) wire and sample are electrically non- conducting
(B) wire and sample are electrically conducting
(C) wire is electrically conducting and sample is electrically non-conducting
(D) sample is electrically conducting and wire is electrically non-conducting
Solution: The correct option is (B).
Question Number: 33 Question Type: MCQ
Internal gears are manufactured by
(A) hobbing
(B) shaping with pinion cutter
(C) shaping with rack cutter
(D) milling

Solution: Internal gears are manufactured by shaping with pinion cutter.
Hence, the correct option is (B).
Question Number: 34
Question Type: MCQ
Match the following part programming codes with their respective functions.

| Part programming <br> codes | Functions |
| :--- | :--- |
| P. G01 | I. Spindle stop |
| Q. G03 | II. Spindle rotation, clockwise |
| R. M03 | III. Circular interpolation, <br> anticlockwise |
| S. M05 | IV. Linear interpolation |

(A) P - II, Q - I, R - IV, S - III
(B) P - IV, Q - II, R - III, S - I
(C) P - IV, Q - III, R - II, S - I
(D) P - III, Q - IV, R - II, S - I

Solution: G01 - linear interpolation, G03 - circular interpolation, ACW.
M03 - spindle rotation, CW; M05 - spindle stop.
Hence, the correct option is (C).
Question Number: 35
Question Type: MCQ
In PERT chart, the activity time distribution is
(A) Normal
(B) Binormal
(C) Poisson
(D) Beta

Solution: In Pert activity, time distribution is Beta distribution.

Hence, the correct option is (D).
Question Number: 36
Question Type: NAT
The number of linearly independent eigenvectors of matrix $A=\left[\begin{array}{lll}2 & 1 & 0 \\ 0 & 2 & 0 \\ 0 & 0 & 3\end{array}\right]$ is $\longrightarrow$.

Solution:
Given matrix is $A=\left[\begin{array}{lll}2 & 1 & 0 \\ 0 & 2 & 0 \\ 0 & 0 & 3\end{array}\right]$
The eigen values of $A$ are 2,2 , and 3 .

Let $X_{1}=\left[\begin{array}{l}x_{1} \\ x_{2} \\ x_{3}\end{array}\right]$ be an eigen vector of $A$ corresponding to the eigen value $\lambda=2$

$$
\begin{gathered}
\therefore(A-\lambda I) X_{1}=0 \\
\Rightarrow(A-2 I) X_{1}=0 \\
\Rightarrow\left[\begin{array}{lll}
0 & 1 & 0 \\
0 & 0 & 0 \\
0 & 0 & 1
\end{array}\right]\left[\begin{array}{l}
x_{1} \\
x_{2} \\
x_{3}
\end{array}\right]=\left[\begin{array}{l}
0 \\
0 \\
0
\end{array}\right] \\
x_{2}=0 \\
x_{3}=0
\end{gathered}
$$

and $x_{1}$ can be arbitrary.
Let $x_{1}=k$, where $k$ is arbitrary.
The eigen vector of $A$ corresponding to the eigen value $\lambda=2$ is $X_{1}=\left[\begin{array}{l}k \\ 0 \\ 0\end{array}\right]=k\left[\begin{array}{l}1 \\ 0 \\ 0\end{array}\right]$
i.e., only one arbitrary value occurred in the eigen vector of $A$ corresponding to the eigen value that is repeated. From this, one can conclude that the total number of linearly independent eigen vectors of $A$ is 2 .
Hence, the correct answer is 02 .
Question Number: 37
Question Type: NAT
The value of the line integral $\oint_{C} \bar{F} \cdot \bar{r}$ ' $d s$, where $C$ is a circle of radius $\frac{4}{\sqrt{\pi}}$ units is $\qquad$ -.

Here, $\bar{F}(x, y)=y \hat{i}+2 x \hat{j}$ and $\bar{r}$, is the UNIT tangent vector on the curve $C$ at an arc length $s$ from a reference point on the curve. $\hat{i}$ and $\hat{j}$ are the basis vectors in the $x-y$ Cartesian reference. In evaluating the line integral, the curve has to be traversed in the counter-clockwise direction.

Solution: We have to evaluate $\oint_{C} \bar{F} \cdot \bar{r} d s$ where $C$ is a circle with radius $\frac{4}{\sqrt{\pi}}$
and

$$
\begin{aligned}
\bar{F} & =y \bar{i}+2 x \bar{j} \\
\oint_{C} \bar{F} \cdot \bar{r} d s & =\oint_{C}[y \bar{i}+2 x \bar{j}] \cdot[d x \bar{i}+d y j]
\end{aligned}
$$

$$
\begin{aligned}
& =\oint_{C}[y d x+2 x d y] \\
& =\iint_{R}\left[\frac{\partial(2 x)}{\partial x}-\frac{\partial(y)}{\partial y}\right] d x d y
\end{aligned}
$$

(By Green's theorem)
where $R$ is the region of the circle $C$.

$$
\begin{aligned}
& =\iint_{R}[2-1] d x d y \\
& =\iint_{R}[2-1] d x d y \\
& =\text { Area of the circle } C\left(=\pi r^{2}\right) \\
& =\pi\left(\frac{4}{\sqrt{\pi}}\right)^{2}=16
\end{aligned}
$$

Hence, the correct answer is 16 .
Question Number: 38
Question Type: MCQ
$\lim _{x \rightarrow \infty} \sqrt{x^{2}+x-1}-x$ is
(A) 0
(B) $\infty$
(C) $1 / 2$
(D) $-\infty$

Solution: We have $\lim _{x \rightarrow \infty} \sqrt{x^{2}+x-1}-x$

$$
\begin{aligned}
& =\lim _{x \rightarrow \infty}\left[\sqrt{x^{2}+x-1}-x\right] \times \frac{\left.\sqrt{x^{2}+x-1}+x\right]}{\left[\sqrt{x^{2}+x-1}+x\right]} \\
& =\lim _{x \rightarrow \infty} \frac{\left(x^{2}+x-1\right)-x^{2}}{\left[\sqrt{x^{2}+x-1}+x\right]} \\
& =\lim _{x \rightarrow \infty} \frac{x-1}{\left[\sqrt{x^{2}\left(1+\frac{1}{x}-\frac{1}{x^{2}}\right)}+x\right]} \\
& =\lim _{x \rightarrow \infty} \frac{x\left(1-\frac{1}{x}\right)}{\left[x\left[\sqrt{\left(1+\frac{1}{x}-\frac{1}{x^{2}}\right)}\right]+1\right]} \\
& =\lim _{x \rightarrow \infty} \frac{1-\frac{1}{x}}{\left[\sqrt{1+\frac{1}{x}-\frac{1}{x^{2}}}+1\right]}=\frac{1}{2}
\end{aligned}
$$

Hence, the correct option is (C).

## Question Number: 39 <br> Question Type: MCQ

Three cards were drawn from a pack of 52 cards. The probability that they are a king, a queen, and a jack is
(A) $\frac{16}{5525}$
(B) $\frac{64}{2197}$
(C) $\frac{3}{13}$
(D) $\frac{8}{16575}$

Solution: When three cards were drawn from a pack of 52 cards, the probability that they are a king, a queen, and a jack

$$
=\frac{{ }^{4} C_{1} \times{ }^{4} C_{1} \times{ }^{4} C_{1}}{{ }^{52} C_{3}}=\frac{16}{5525}
$$

Hence, the correct option is (A).
Question Number: 40
Question Type: NAT
An inextensible mass less string goes over a frictionless pulley. Two weights of 100 N and 200 N are attached to the two ends of the string. The weights are released from rest, and start moving due to the gravity. The tension in the string (in N ) is $\qquad$ -.


## Solution:



If $a$ is the acceleration of system, $g$ is the acceleration due to gravity, and $T$ be the tension, then we have

$$
\begin{align*}
& T-200=\frac{200}{g} a  \tag{1}\\
& T-100=-\frac{100}{g} a \tag{2}
\end{align*}
$$

Solving Eqs. (1) and (2), we have

$$
T=133.33 \mathrm{~N}
$$

Hence, the correct answer is $133-134$.

## Question Number: 41

Question Type: NAT
A circular disc of radius 100 mm and mass 1 kg , initially at rest at position $A$, rolls without slipping down a curved path as shown in the figure. The speed $v$ of the disc when it reaches position $B$ is $\qquad$ $\mathrm{m} / \mathrm{s}$.
Acceleration due to gravity $g=10 \mathrm{~m} / \mathrm{s}^{2}$.


Solution: Radius of circular disc $r=100 \mathrm{~m}$
Mass $m=1 \mathrm{~kg}$
Speed of the disc $v$ ?
Applying conservation of energy,


Now we get

$$
\begin{aligned}
m g h & =\frac{1}{2} m v^{2}+\frac{1}{2}\left(I_{\mathrm{o}}\right) \omega^{2} \\
& =\frac{1}{2} m v^{2}+\frac{1}{2} \times \frac{1}{2} m r^{2} \omega^{2} \\
& =\left(\frac{1}{2}+\frac{1}{4}\right) m v^{2}=\frac{3}{4} m v^{2} \\
v & =\sqrt{\frac{4 g h}{3}} \\
v & =20 \mathrm{~m} / \mathrm{s}
\end{aligned}
$$

Hence, the correct answer is 20 .
Question Number: 42
Question Type: NAT A rigid $\operatorname{rod}(A B)$ of length $L=\sqrt{2} \mathrm{~m}$ is undergoing translational as well as rotational motion in the $x-y$ plane (see the figure). The point $A$ has the velocity $V_{1}=\hat{i}+2 \hat{j} \mathrm{~m} / \mathrm{s}$. The end $B$ is constrained to move only along the $x$ direction.
The magnitude of the velocity $V_{2}$ (in $\mathrm{m} / \mathrm{s}$ ) at the end $B$ is
$\qquad$
-.


## Solution:



Velocity at point $A$ is $V_{1}=\hat{i}+2 \hat{j} \mathrm{~m} / \mathrm{s}$
Magnitude of velocity at point $A$
$V_{1}=\sqrt{1+4}=\sqrt{5} \mathrm{~m} / \mathrm{s}$
$V_{1}$ makes an angle $\tan ^{-1}(2)$ or $63.43^{\circ}$ with $x$-axis.

$$
\begin{aligned}
\therefore \quad \phi & =63.43^{\circ}-45^{\circ} \\
& =18.43^{\circ}
\end{aligned}
$$

Let the velocity at point $B$ be $V_{2}=$ ?
For $\operatorname{rod} A B$ to be rigid,

$$
\begin{aligned}
V_{1} \cos \phi & =V_{2} \cos 45^{\circ} \\
V_{2} & =3 \mathrm{~m} / \mathrm{s}
\end{aligned}
$$

Hence, the correct answer is 3 .

## Question Number: 43

Question Type: NAT
A square plate of dimension $L \times L$ is subjected to a uniform pressure load $p=250 \mathrm{MPa}$ on its edges as shown in the figure. Assume plane stress conditions. The Young's modulus $E=200 \mathrm{GPa}$.


The deformed shape is a square of dimension $L-2 \delta$. If $L=2 \mathrm{~m}$ and $\delta=0.001 \mathrm{~m}$, the Poisson's ratio of the plate material is $\qquad$ .

Solution:
Uniform pressure $P=250 \mathrm{MPa}$
$\delta=0.001 \mathrm{~m}$
Young's modulus $E=200 \mathrm{GPa}$
Length $L=2 \mathrm{~m}$
$\varepsilon_{\mathrm{x}}=28$


$$
\begin{aligned}
\varepsilon_{\mathrm{x}} & =\frac{\sigma_{x}}{E}-\frac{\mu \sigma_{y}}{\epsilon} \Rightarrow \frac{1}{E}\left[\sigma_{\mathrm{x}}-\mu \sigma_{\mathrm{y}}\right] \\
\frac{2 \times 0.001}{2} & =\frac{1}{200 \times 10^{9}} \times 250 \times 10^{6}[1-\mu] \\
0.8 & =1-\mu \Rightarrow \mu=0.2
\end{aligned}
$$

Hence, the correct answer is 0.2 .
Question Number: 44
Question Type: MCQ
Two circular shafts made of the same material, one solid $(S)$ and one hollow $(H)$, have the same length and polar moment of inertia. Both are subjected to same torque. Here, $\theta_{\mathrm{s}}$ is the twist and $\tau_{\mathrm{s}}$ is the maximum shear stress in the solid shaft, whereas $\theta_{\mathrm{H}}$ is the twist and $\tau_{\mathrm{H}}$ is the maximum shear stress in the hollow shaft. Which one of the following is TRUE?
(A) $\theta_{\mathrm{S}}=\theta_{\mathrm{H}}$ and $\tau_{\mathrm{S}}=\tau_{\mathrm{H}}$
(B) $\theta_{\mathrm{S}}>\theta_{\mathrm{H}}$ and $\tau_{\mathrm{S}}>\tau_{\mathrm{H}}$
(C) $\theta_{\mathrm{S}}<\theta_{\mathrm{H}}$ and $\tau_{\mathrm{S}}<\tau_{\mathrm{H}}$
(D) $\theta_{\mathrm{S}}=\theta_{\mathrm{H}}$ and $\tau_{\mathrm{S}}<\tau_{\mathrm{H}}$

Solution:
Material ${ }_{\text {hollow }}=$ Material $_{\text {solid }}$
$\frac{T}{J}=\frac{\tau}{r}=\frac{G \theta}{L}$
$\Rightarrow \quad \tau \propto r$
$\because$
$r_{\mathrm{H}}>r_{\mathrm{S}}$
$\Rightarrow \quad \tau_{\mathrm{H}}>\tau_{\mathrm{S}}$
$\theta_{\text {solid }}=\theta_{\text {hollow }}$
Hence, the correct option is (D).

## Question Number: 45 <br> Question Type: MCQ

A beam of length $L$ is carrying a uniformly distributed load $w$ per unit length. The flexural rigidity of the beam is $E I$. The reaction at the simple support at the right end is

(A) $\frac{w L}{2}$
(B) $\frac{3 w L}{8}$
(C) $\frac{w L}{4}$
(D) $\frac{w L}{8}$

## Solution:



Let $R_{\mathrm{B}}$ be reaction at $B$.
Deflection due to U.D.L $=\frac{\omega L^{4}}{8 E I}$
Upward deflection due to simple support at

$$
\begin{equation*}
B=\frac{R_{\mathrm{B}} L^{3}}{3 E I} \tag{2}
\end{equation*}
$$

From Eqs. (1) and (2), we get
We have $R_{\mathrm{B}}=\frac{3}{8} w L$.
Hence, the correct option is (B).
Question Number:46
Question Type: NAT
Two masses $m$ are attached to opposite sides of a rigid rotating shaft in the vertical plane. Another pair of equal masses $m_{1}$ is attached to the opposite sides of the shaft in the vertical plane as shown in the figure. Consider $m=1 \mathrm{~kg}, e=50 \mathrm{~mm}, e_{1}=20$ $\mathrm{mm}, b=0.3 \mathrm{~m}, a=2 \mathrm{~m}$, and $a_{1}=2.5 \mathrm{~m}$. For the system to be dynamically balanced, $m_{1}$ should be $\qquad$ kg.


Solution:


Mass $m=1 \mathrm{~kg}$,
Distance $e=50 \mathrm{~mm}$,
Distance $e_{1}=20 \mathrm{~mm}$,
Length $b=0.3 \mathrm{~m}$,
Length $a=2 \mathrm{~m}$,
Length $a_{1}=2.5 \mathrm{~m}$,
Mass $m_{1}=$ ?
We know that couple on any plane should be zero for dynamic balance

$$
\begin{aligned}
M_{\mathrm{A}} & =0 \\
\Rightarrow \quad m e b+m_{1} e_{1} a_{1} & =m(a+b) e \\
m_{1} e_{1} a_{1} & =m e a \\
m_{1} & =\frac{1 \times 50 \times 2}{20 \times 2.5}=2 \\
m_{1} & =2 \mathrm{~kg}
\end{aligned}
$$

Hence, the correct answer is 2 .
Question Number: 47
Question Type: NAT
A single degree of freedom spring-mass is subjected to a harmonic force of constant amplitude. For an excitation frequency of $\sqrt{\frac{3 k}{m}}$, the ratio of the amplitude of steadystate response to the static deflection of the spring is
$\qquad$


Solution: Excitation frequency $\omega=\sqrt{\frac{3 k}{m}}$ Also, we have $\omega_{\mathrm{n}}=\sqrt{\frac{k}{m}}$

Magnification factor can be calculated as

$$
\begin{aligned}
& =\frac{1}{\sqrt{\left(1-\left(\frac{\omega}{\omega_{n}}\right)^{2}\right)^{2}+\left(2 \xi \frac{\omega}{\omega_{n}}\right)^{2}}} \\
& = \pm \frac{1}{1-\left(\frac{\omega}{\omega_{n}}\right)^{2}}= \pm \frac{1}{1-3}=\frac{1}{2}
\end{aligned}
$$

(Neglecting -ve sign)
Hence, the correct answer is 0.5 .

## Question Number: 48

## Question Type: NAT

A bolted joint has four bolts arranged as shown in the figure. The cross-sectional area of each bolt is $25 \mathrm{~mm}^{2}$. A torque $T=200 \mathrm{~N}-\mathrm{m}$ is acting on the joint. Neglecting friction due to clamping force, the maximum shear stress in a bolt is $\qquad$ MPa.


Solution: Torque $T=200 \mathrm{~N}-\mathrm{m}$
Cross-sectional area of each bolt $A=25 \mathrm{~mm}^{2}$
Number of bolts $n=4$
Now using the relation

$$
\begin{aligned}
T & =n \times A \times \tau_{\max } \times \frac{P C D}{2} \\
200 \times 10^{3} & =4 \times 25 \times \frac{100}{2} \times \tau_{\max }
\end{aligned}
$$

Maximum torque will be

$$
\tau_{\max }=\frac{200 \times 10^{3} \times 4 \times 2}{25 \times 100}=40 \mathrm{~N} / \mathrm{mm}^{2}
$$

Hence, the correct answer is 40 .
Question Number: 49
Question Type: NAT
Consider a fully developed steady laminar flow of an incompressible fluid with viscosity $\mu$ through a circular pipe of radius $R$. Given that the velocity at a radial location of $R / 2$ from the centerline of the pipe is $U_{1}$, the shear stress at the wall is $K \mu \mathrm{U}_{1} / R$, where $K$ is $\qquad$ .

## Solution:

As per problem, $U_{R / 2}=U_{1}$

$$
\begin{aligned}
& \text { Also } \quad \begin{aligned}
\tau_{\text {wall }} & =k \mu u_{1} / R \\
u & =u_{\max }\left(1-\frac{r^{2}}{R^{2}}\right) \\
U_{1} & =\frac{3}{4} u_{\max } \\
\tau_{\text {wall }} & =\mu \frac{d u}{d r}=\mu u_{\max } \frac{2 r}{R^{2}} \\
& =2 \mu \frac{u_{\max }}{R}=2 \frac{\mu}{R} \times \frac{4}{3} u_{1} \\
& =\frac{8}{3} \mu \frac{u_{1}}{R} \\
\therefore \quad k & =\frac{8}{3}=2.667
\end{aligned}
\end{aligned}
$$

Hence, the correct answer is $2.6-2.7$.
Question Number: 50
Question Type: NAT
The water jet exiting from a stationary tank through a circular opening of diameter 300 mm impinges on a rigid wall as shown in the figure. Neglect all minor losses and assume the water level in the rank to remain constant. The net horizontal force experienced by the wall is $\qquad$ kN .
Density of water is $1000 \mathrm{~kg} / \mathrm{m}^{3}$.

## Acceleration due to gravity $\boldsymbol{g}=\mathbf{1 0} \mathbf{~ m} / \mathrm{s} 2$.



## Solution:



Force exerted by a set of water striking fixed wall is

$$
=\rho a v^{2}=10^{3} \times \frac{\pi}{4} \times 0.3^{2} \times V^{2}
$$

Height $h=6.2 \mathrm{~m}$
Velocity of jet can be calculated as

$$
\begin{gathered}
V=\sqrt{2 g h} \\
=10^{3} \times \frac{\pi}{4} \times 0.3^{2} \times 2 \times 10 \times 6.2=8.76 \mathrm{kN}
\end{gathered}
$$

Hence, the correct answer is $8.7-8.8$.

## Question Number: 51

Question Type: MCQ
For a two-dimensional flow, the velocity field is $\vec{u}=\frac{x}{x^{2}+y^{2}} \hat{i}+\frac{y}{x^{2}+y^{2}} \hat{j}$, where $\hat{i}$ and $\hat{j}$ are the basis vectors in the $x-y$ Cartesian coordinate system. Identify the CORRECT statements from the following.
(1) The flow is incompressible.
(2) The flow is unsteady.
(3) $y$-component of acceleration,

$$
a_{y}=\frac{-y}{\left(x^{2}+y^{2}\right)^{2}}
$$

(4) $x$-component of acceleration,

$$
a_{x}=\frac{-(x+y)}{\left(x^{2}+y^{2}\right)^{2}}
$$

(A) (2) and (3)
(B) (1) and (3)
(C) (1) and (2)
(D) (3) and (4)

## Solution:

$$
\begin{gathered}
u=\frac{x}{x^{2}+y^{2}} \quad V=\frac{y}{x^{2}+y^{2}} \\
a_{x}=\frac{u \delta u}{\delta x}+V \frac{\delta u}{\delta y} \\
\frac{\delta u}{\delta x}=\frac{y^{2}-x^{2}}{\left(x^{2}+y^{2}\right)^{2}}, \frac{\delta v}{\delta y}=\frac{x^{2}-y^{2}}{\left(x^{2}+y^{2}\right)^{2}} \\
\frac{\delta u}{\delta y}=\frac{-2 x y}{\left(x^{2}+y^{2}\right)^{2}} \\
a_{x}=\frac{-x}{\left(x^{2}+y^{2}\right)^{2}} \\
\text { also, } \quad \frac{\delta u}{\delta x}+\frac{\delta v}{\delta y}=0
\end{gathered}
$$

Flow is steady and incompressible.
Hence, the correct option is (B).
Question Number: 52
Question Type: NAT
Two large parallel plates having a gap of 10 mm in between them are maintained at temperatures $T_{1}=1000 \mathrm{~K}$ and $T_{2}$ $=400 \mathrm{~K}$. Given emissivity values, $\varepsilon_{1}=0.5, \varepsilon_{2}=0.25$, and Stefan-Boltzmann constant $\sigma=5.67 \times 10^{-8} \mathrm{~W} / \mathrm{m}^{2}-\mathrm{K}^{4}$, the heat transfer between the plates (in $\mathrm{kW} / \mathrm{m}^{2}$ ) is $\qquad$ .

## Solution:



Temperature $T_{1}=1000 \mathrm{~K}$
Temperature $T_{2}=400 \mathrm{~K}$
Emissivity value $\varepsilon_{1}=0.5$
Emissivity value $\varepsilon_{2}=0.25$
Heat transfer between the plates (in $\mathrm{kW} / \mathrm{m}^{2}$ ) can be calculated using

$$
\begin{aligned}
Q & =\frac{\sigma\left(T_{1}^{4}-T_{2}^{4}\right)}{1 / \varepsilon_{1}+1 / \varepsilon_{2}-1} \\
& =\frac{5.67 \times 10^{-8}\left(1000^{4}-400^{4}\right)}{1 / 0.5^{+1} / 0.25^{-1}}=11049.69 \\
& \approx 11.049 \mathrm{~kW} / \mathrm{m}^{2}
\end{aligned}
$$

Hence, the correct answer is $10.9-11.2$.
Question Number: 53
Question Type: NAT
A cylindrical steel rod, 0.01 m in diameter and 0.2 m in length, is first heated to $750^{\circ} \mathrm{C}$ and then immersed in a water bath at $100^{\circ} \mathrm{C}$. The heat transfer coefficient is $250 \mathrm{~W} /$ $\mathrm{m}^{2}-\mathrm{K}$. The density, specific heat, and thermal conductivity of steel are $\rho=7801 \mathrm{~kg} / \mathrm{m}^{3}, c=473 \mathrm{~J} / \mathrm{kg}-\mathrm{K}$, and $k=43$ $\mathrm{W} / \mathrm{m}-\mathrm{K}$, respectively. The time required for the rod to reach $300^{\circ} \mathrm{C}$ is $\qquad$ seconds.
Solution: Diameter of steel rod $d=0.01$
Length of steel $\operatorname{rod} L=0.2$
Temperature to which steel rod is heated $T_{0}=750^{\circ} \mathrm{C}$
Temperature of water in bath $T_{\infty}=100^{\circ} \mathrm{C}$
Heat transfer coefficient $h=250 \mathrm{~W} / \mathrm{m}^{2}-\mathrm{K}$

Density of steel $\rho=7801 \mathrm{~kg} / \mathrm{m}^{3}$
Specific heat $c=473 \mathrm{~J} / \mathrm{K}$
Thermal conductivity $k=43 \mathrm{~W} / \mathrm{mK}$
Temperature $T_{1}=300^{\circ} \mathrm{C}$
Time required $t=$ ?
Now using the relation

$$
\frac{T_{o}-T_{\infty}}{T_{1}-T_{\infty}}=e^{(h A / \rho V C) t}
$$

where area $A=\pi d L$, volume $v=\frac{\pi}{4} d^{2} L$ )

$$
\begin{gathered}
\frac{750-100}{300-100}=e^{\frac{250 \pi d L \times 4}{7801 \times \pi d^{2} L \times 473} t} \\
t=43.49 \text { seconds }
\end{gathered}
$$

Hence, the correct answer is $42-45$.
Question Number: 54
Question Type: NAT
Steam at an initial enthalpy of $100 \mathrm{~kJ} / \mathrm{kg}$ and inlet velocity of $100 \mathrm{~m} / \mathrm{s}$, enters an insulated horizontal nozzle. It leaves the nozzle at $200 \mathrm{~m} / \mathrm{s}$. The exit enthalpy (in $\mathrm{kJ} / \mathrm{kg}$ ) is $\qquad$ .

Solution: Initial enthalpy $h_{1}=100 \mathrm{~kJ} / \mathrm{kg}$
Inlet velocity $V_{1}=100 \mathrm{~m} / \mathrm{s}$
Velocity when it leaves the nozzle $V_{2}=200 \mathrm{~m} / \mathrm{s}$


Now using the relation

$$
\begin{gathered}
h_{1}+\frac{V_{1}^{2}}{L}=h_{2}+\frac{V_{2}^{2}}{L} \\
\Rightarrow 100+\frac{100^{2}-200^{2}}{2000}=h_{2} \\
h_{2}=85 \mathrm{~kJ} / \mathrm{kg}
\end{gathered}
$$

Hence, the correct answer is 85 .
Question Number: 55
Question Type: NAT
In a mixture of dry air and water vapor at a total pressure of 750 mm of Hg , the partial pressure of water vapor is 20 mm of Hg . The humidity ratio of the air in grams of water vapor per kg of dry air $\left(\mathrm{g}_{\mathrm{w}} / \mathrm{kg}_{\text {da }}\right)$ is $\qquad$ -.

Solution: Total pressure of mixture of dry air and water vapor $P_{\text {Total }}=750 \mathrm{~mm}$ of Hg
Pressure of water vapor $P_{\mathrm{v}}=20 \mathrm{~mm}$ of Hg
Humidity ratio of the air $\omega=$ ?
Now using the relation

$$
\begin{aligned}
\omega & =0.622 \frac{P_{\mathrm{v}}}{P_{\text {Total }}-P_{\mathrm{v}}}=0.622 \times \frac{20}{750-20} \\
& =0.622 \times \frac{20}{730} \\
& \approx 0.01704 \mathrm{~kg} / \mathrm{kg} \text { dry air } \\
& =17.04 \mathrm{~g} / \mathrm{kg} \text { dry air }
\end{aligned}
$$

Hence, the correct answer is $16.9-17.1$.

## Question Number: 56

Question Type: NAT
In a three-stage air compressor, the inlet pressure is $p_{1}$, discharge pressure is $p_{4}$, and the intermediate pressures are $p_{2}$ and $p_{3}\left(p_{2}<p_{3}\right)$. The total pressure ratio of the compressor is 10 and the pressure ratios of the stages are equal. If $p_{1}=100 \mathrm{kPa}$, the value of the pressure $p_{3}($ in kPa$)$ is $\qquad$ -.

## Solution:

Inlet pressure $P_{1}=100 \mathrm{kPa}$
Ratio of discharge pressure and inlet pressure is

$$
\frac{P_{4}}{P_{1}}=10
$$

For three-stage compression

$$
\begin{aligned}
\frac{P_{2}}{P_{1}}=\frac{P_{3}}{P_{2}} & =\frac{P_{4}}{P_{3}}=\text { Constant } \\
\frac{P_{2}}{P_{1}} & =10^{1 / 3} \\
\therefore \quad \frac{P_{3}}{P_{1}} & =10^{2 / 3} \\
P_{3} & =P_{1} \times 10^{2 / 3} \\
& =100 \times 10^{2 / 3}=464 \mathrm{kPa}
\end{aligned}
$$

Hence, the correct answer is $460-470$.
Question Number: 57
Question Type: NAT
In the vapor compression cycle shown in the figure, the evaporating and condensing temperatures are 260 K and 310 K , respectively. The compressor takes in liquid--vapor mixture (state 1) and isentropically compresses it to a dry saturated vapor condition (state 2). The specific heat of the liquid refrigerant is $4.8 \mathrm{~kJ} / \mathrm{kg}-\mathrm{K}$ and may be treated as constant. The enthalpy of evaporation for the refrigerant at 310 K is $1054 \mathrm{~kJ} / \mathrm{kg}$.

The difference between the enthalpies at state points 1 and $0($ in $\mathrm{kJ} / \mathrm{kg})$ is $\qquad$ -.


Solution: We now that $d Q=C_{\mathrm{p}} d t$

$$
\begin{aligned}
& s_{2}-s_{0}=\int_{0}^{3} \frac{C_{\mathrm{p}} d T}{T}+\frac{h_{\mathrm{fg}}}{310} \\
& s_{1}-s_{0}=4.8 \ln \left(\frac{310}{260}\right)+\frac{1054}{310} \\
& \frac{h_{1}-h_{0}}{260}=4.8 \ln \left(\frac{310}{260}\right)+\frac{1054}{310} \\
& \Rightarrow \quad h_{1}-h_{0}=1103.5 \mathrm{~kJ} / \mathrm{kg}
\end{aligned}
$$

Hence, the correct answer is $1103.5 \mathrm{~kJ} / \mathrm{kg}$.

## Question Number: 58 <br> Question Type: NAT

Spot welding of two steel sheets each 2 mm thick is carried out successfully by passing 4 kA of current for 0.2 seconds through the electrodes. The resulting weld nugget formed between the sheets is 5 mm in diameter. Assuming cylindrical shape for the nugget, the thickness of the nugget is
$\qquad$ mm .

| Latent heat of fusion for steel | $1400 \mathrm{~kJ} / \mathrm{kg}$ |
| :--- | :--- |
| Effective resistance of the weld joint | $200 \mu \Omega$ |
| Density of steel | $8000 \mathrm{~kg} / \mathrm{m}^{3}$ |

## Solution:



Current $I=4 \mathrm{kA}=4000 \mathrm{~A}$
Resistance of weld joint $R=200 \times 10^{-6} \Omega$
Time $t=0.2 \mathrm{~s}$
Density of steel $\rho=8000 \mathrm{~kg} / \mathrm{m}^{3}$
Diameter of weld nugget $d_{\mathrm{n}}=5 \mathrm{~mm}$
Latent heat of fusion for steel $=1400 \mathrm{~kJ} / \mathrm{kg}$
$t_{\text {nugget }}=$ ?

Volume of nugget $=\frac{\pi}{4} d_{\mathrm{n}}^{2} \times t_{\text {nugget }}$

$$
\begin{aligned}
\text { Heat supplied } & =r^{2} R t=4000^{2} \times 0.2 \times 200 \times 10^{-6} \\
& =640 \mathrm{~J}
\end{aligned}
$$

Heat required to melt $=\mathrm{L} . \mathrm{H} \times \rho \times$ volume

$$
\begin{aligned}
& =1400 \times 10^{3} \times 8000 \times \frac{\pi}{4} \times 25 \times 10^{-6} \times t_{\text {nugget }} \times 10^{-3} \\
& =219.8 t_{\text {nugget }}
\end{aligned}
$$

So, $t_{\text {nugget }}=\frac{640}{219.8}=2.911 \mathrm{~mm}$
Hence, the correct answer is 2.85 to 2.95 .
Question Number: 59
Question Type: MCQ
For an orthogonal cutting operation, tool material is HSS, rake angle is $22^{\circ}$, chip thickness is 0.8 mm , speed is 48 $\mathrm{m} / \mathrm{min}$, and feed is $0.4 \mathrm{~mm} / \mathrm{rev}$. The shear plane angle (in degrees) is
(A) 19.24
(B) 29.70
(C) 56.00
(D) 68.75

Solution: Rake angle $\alpha_{0}=22^{\circ}$
Chip thickness $t_{2}=0.8 \mathrm{~mm}$
Feed $t_{1}=0.4 \mathrm{~mm} / \mathrm{rev}$
Chip reduction coefficient

$$
k=\frac{t_{2}}{t_{1}}=2
$$

Let the shear plane angle be $\theta=$ ?
Now using the relation

$$
\begin{aligned}
\tan \theta & =\frac{\cos \alpha_{0}}{k-\sin \alpha_{0}}=\frac{\cos 22}{2-\sin 22}=0.57 \\
\theta & =29.70
\end{aligned}
$$

Hence, the correct option is (B).
Question Number: 60
Question Type: NAT
In a sheet metal of 2 mm thickness, a hole of 10 mm diameter needs to be punched. The yield strength in tension of the sheet material is 100 MPa and its ultimate shear strength is 80 MPa . The force required to punch the hole (in kN ) is
$\qquad$ —.

Solution: Thickness of metal sheet $t=2 \mathrm{~mm}$
Diameter of hole $d=10 \mathrm{~mm}$
Ultimate shear strength $\tau_{\mathrm{s}}=80 \mathrm{MPa}$
Yield strength $S_{\text {yT }}=100 \mathrm{MPa}$
Force required to punch the hole can be calculated using the following relation:

$$
\begin{aligned}
F & =\pi d t \times \tau_{\mathrm{s}} \\
& =\pi \times 2 \times 10 \times 80=5.026 \mathrm{kN}
\end{aligned}
$$

Hence, the correct answer is $4.9-5.1$.
Question Number: 61
Question Type: NAT
In a single turning operation with cemented carbide tool and steel work piece, it is found that the Taylor's exponent is 0.25 . If the cutting speed is reduced by $50 \%$, then the tool life changes by $\qquad$ times.

Solution: If the cutting speed is reduced by $50 \%$, then $\mathrm{V}_{2}$ $=V_{1} / 2$
Taylor's exponent $n=0.25$
Now using the relation

$$
\begin{gathered}
V_{1} T_{1}^{n}=V_{2} T_{2}^{n} \\
V_{1} T_{1}^{0.25}=\frac{V_{1}}{2} T_{2}^{0.25} \\
\Rightarrow \frac{T_{2}}{T_{1}}=16
\end{gathered}
$$

Hence, the correct answer is 16 .

## Question Number: 62

Question Type: NAT
Two optically flat plates of glass are kept at a small angle $\theta$ as shown in the figure. Monochromatic light is incident vertically.


If the wavelength of light used to get a fringe spacing of 1 mm is 450 nm , the wavelength of light (in mm) to get a fringe spacing of 1.5 mm is $\qquad$ —.

Solution: Wavelength of light $\lambda_{1}=450 \mathrm{~mm}$
Fringe spacing $(\Delta h)_{1}=1 \mathrm{~mm}$
Fringe spacing $(\Delta h)_{2}=1.5 \mathrm{~mm}$
Let the wavelength of light (in mm ) to get a fringe spacing of 1.5 mm be $\lambda_{2}=$ ?
Now using the relation

$$
\Delta h=\frac{n \lambda}{2}
$$

$$
\begin{gathered}
\frac{(\Delta h)_{1}}{(\Delta h)_{2}}=\frac{\lambda_{1}}{\lambda_{2}} \\
\Rightarrow \frac{1}{1.5}=\frac{450}{\lambda_{2}} \Rightarrow \lambda_{2}=675 \mathrm{~mm}
\end{gathered}
$$

Hence, the correct answer is 675.

## Question Number: 63

## Question Type: MCQ

A point $P(1,3,-5)$ is translated by $2 \hat{i}+3 \hat{j}-4 \hat{k}$ and then rotated counter clockwise by $90^{\circ}$ about the $z$-axis. The new position of the point is
(A) $(-6,3,-9)$
(B) $(-6,-3,-9)$
(C) $(6,3,-9)$
(D) $(6,3,9)$

Solution: After translation and then angular rotation final point is $(-6,3,-9)$.
Hence, the correct option is (A).
Question Number: 64
Question Type: MCQ
The demand for a two-wheeler was 900 units and 1030 units in April 2015 and May 2015, respectively. The forecast for the month of April 2015 was 850 units. Considering a smoothing constant of 0.6 , the forecast for the month of June 2015 is
(A) 850 units
(B) 927 units
(C) 965 units
(D) 970 units

Solution: Consider the table given below:

|  | Demand | Forecast |
| :---: | :---: | :---: |
| April | 900 | 850 |
| May | 1030 |  |

Now the forecast for the month of May will be

$$
\begin{aligned}
F_{\text {May }} & =F_{\text {April }}+\alpha\left[D_{\text {April }}-F_{\text {Aprill }}\right] \\
& =850+0.6[50] \\
& =880
\end{aligned}
$$

And the forecast for the month of June will be

$$
\begin{aligned}
F_{\mathrm{June}} & =F_{\mathrm{May}}+\alpha\left[D_{\mathrm{May}}-F_{\mathrm{May}}\right] \\
& =880+0.6[1030-880] \\
& =970
\end{aligned}
$$

Hence, the correct option is (D).
Question Number: 65
Question Type: NAT
A firm uses a turning center, a milling center, and a grinding machine to produce two parts. The table below provides the machining time required for each part and the maximum machining time available on each machine. The profit per unit on parts I and II are Rs. 40 and Rs. 100, respectively. The maximum profit per week of the firm is Rs. $\qquad$ -.
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|  | Machining time required <br> for the machine part <br> (minutes) | Maximum <br> machining <br> time of <br> machine | II |
| :--- | :---: | :---: | :---: |
| available <br> per week <br> (minutes) |  |  |  |
| Turning <br> center | 12 | 6 | 6000 |
| Milling <br> center <br> Grinding <br> machine | 4 | 10 | 4000 |

Solution: Let parts be $X$ and $Y$

$$
Z=40 X+100 Y
$$

Constraints

$$
\begin{aligned}
& 12 X+6 Y \leq 6000 \\
& 4 X+10 Y \leq 4000 \\
& 2 X+3 Y \leq 1800
\end{aligned}
$$



Objective function maximizes at $A$ (375,
250)

Now using

$$
Z=40 X+100 Y
$$

Substituting $X=375$ and $Y=250$ in the above relation, we get
$\mathrm{Z}_{\text {max }}=40 \times 375+250 \times 100=15,000+25,000=40,000$.
Hence, the correct answer is 40,000 .

